

EPTC Professional Development Course

Reliability Mechanics and Modeling for IC Packaging

– *Theory, Implementation and Practices*

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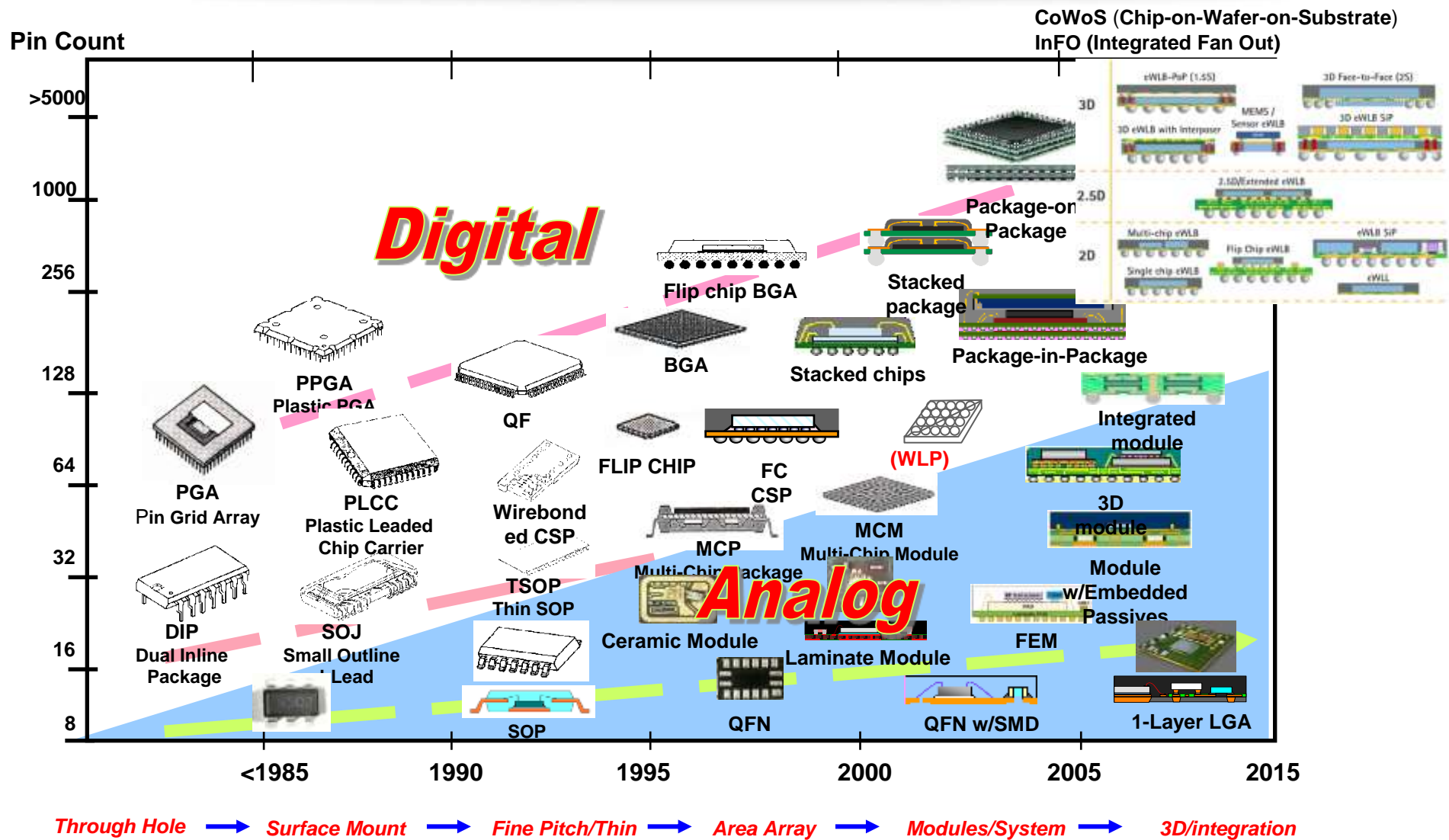


Outline

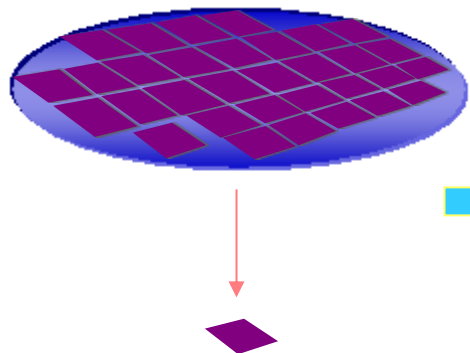
- Introduction
- Temperature Loading
- Mechanical Loading
- Moisture Loading
- Electrical Current Loading - Multi-Physics Modeling
- Summary

- **Introduction**
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Electronic Packaging Evolution



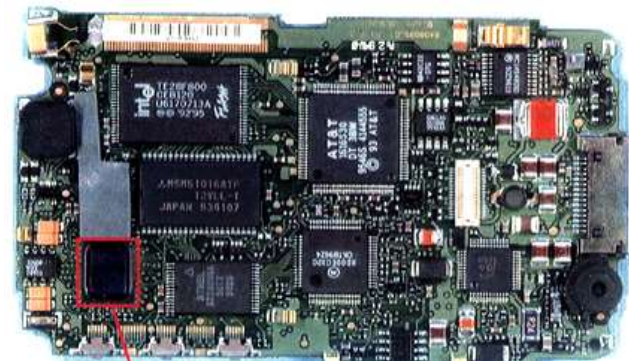
Wafer, Package, and Board Levels



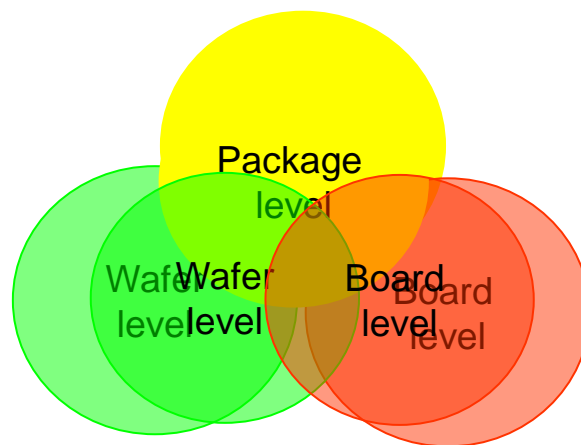
Wafer Fabrication & Backend Process



Electronic Packaging

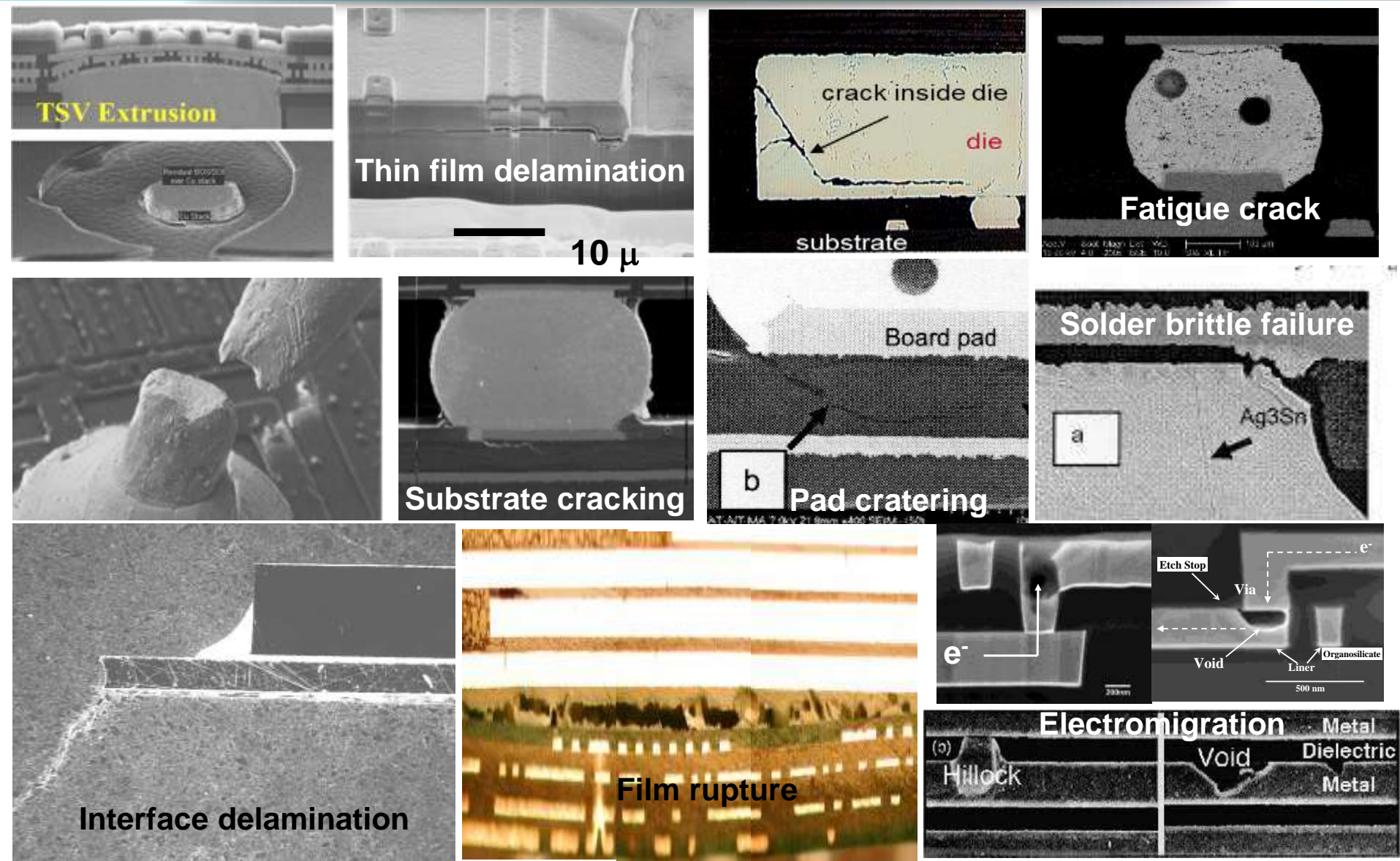


Surface Mounting



- Design of a package must consider the interactions among wafer, package, and board (e.g. CPI – chip-package-interaction).

Examples of Failures



Thermal-mechanical Stress

Stress-free



Cooling-down



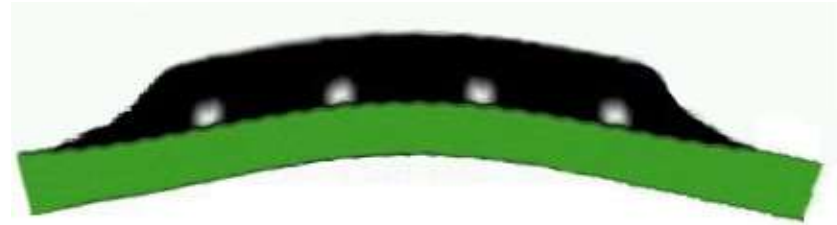
Heating-up



Thermal stress in solder



Cooling-down



Package warpage

Mechanical Load

Handphone



PDA



MPEG-4



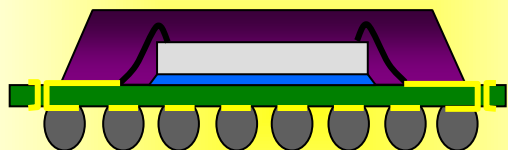
MP3 Player



BAMI

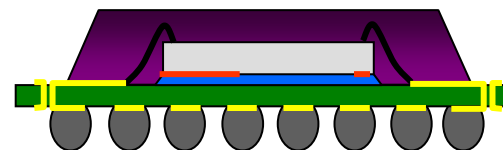
- Handheld electronic products are susceptible to drop impact failure.

Moisture-induced Failure



Stage 1: Moisture absorption

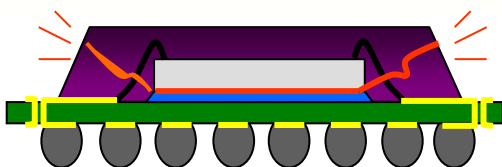
Device exposed to reflow temperature, typically 260°C



Stage 2: Initiation of delamination

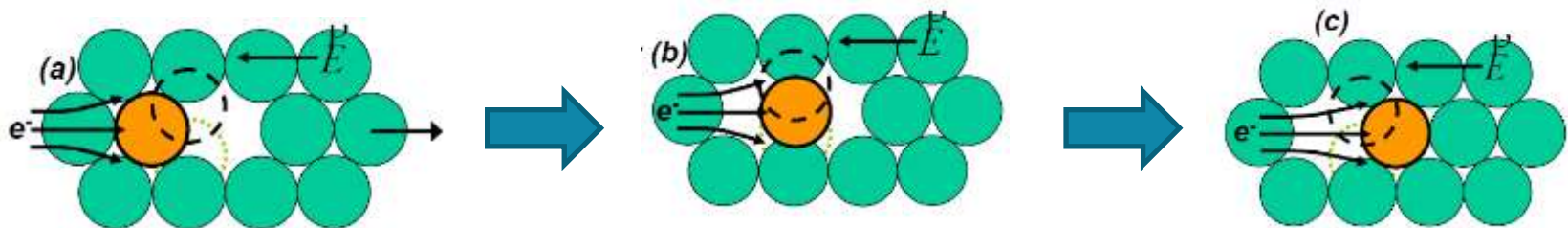
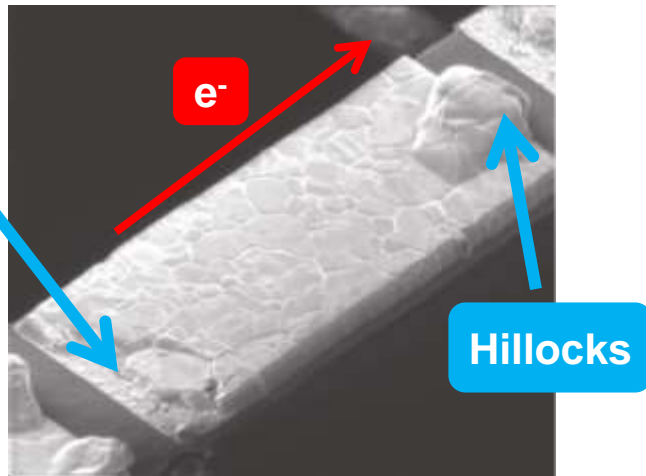
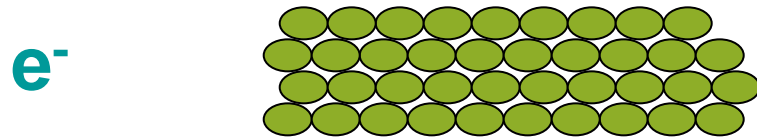


Stage 3: Delamination propagation



Stage 4: Package cracking and vapor release

Electrical Current Stressing



Electron wind

Organization of Course

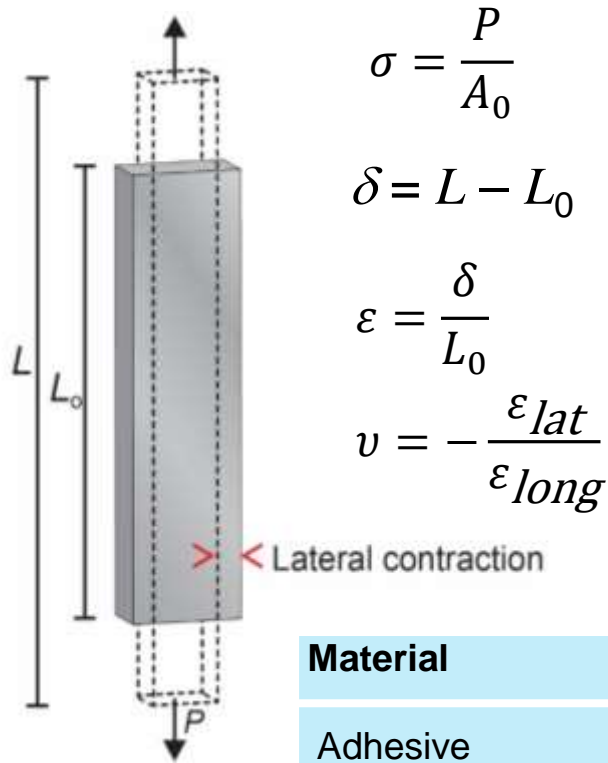
- **Four modules**
 - Temperature loading
 - Thermal mismatch, temperature gradient, etc.
 - Mechanical loading
 - Drop and impact, bend, etc.
 - Moisture loading
 - Moisture diffusion, vapor pressure, swelling.
 - Electrical current loading - combined loading
 - Electrical-thermal, electrical-thermal-mechanical, and electromigration (electrical-thermal-mechanical-mass transport)
- **Theory, implementation, and best practices.**

- Introduction
- **Temperature Loading**
- Mechanical Loading
- Moisture Loading
- Electrical Current Loading - Multi-Physics Modeling
- Summary

Outline

- **Basic concepts and analytical solutions**
- **Applications**
 - Die-level thermal stress – thermal stress in TSV
 - Package-level thermal stress problem – warpage
 - Chip-package interaction (CPI) – sub-modeling technique
 - Board level thermal stress problems
 - Solder ball thermal cycling
 - Creep equations
- **Best practices**
 - Initial stress free condition
 - Full model vs. global/local model
 - Volume averaging
 - Stress singularity

Stress, Strain, and Basic Elastic Material Properties



$$\sigma = \frac{P}{A_0}$$

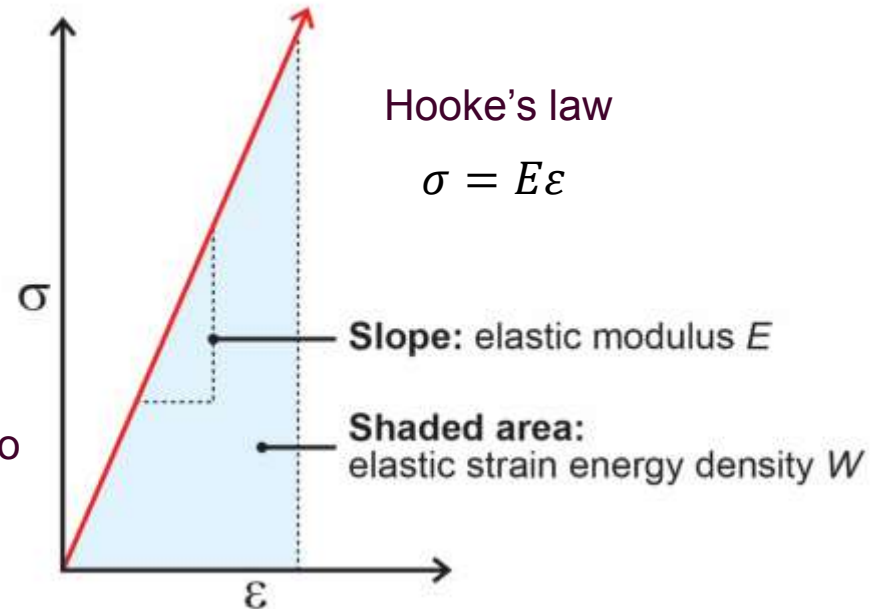
$$\delta = L - L_0$$

$$\varepsilon = \frac{\delta}{L_0}$$

$$\nu = -\frac{\varepsilon_{lat}}{\varepsilon_{long}}$$

Poisson ratio

Lateral contraction



Hooke's law

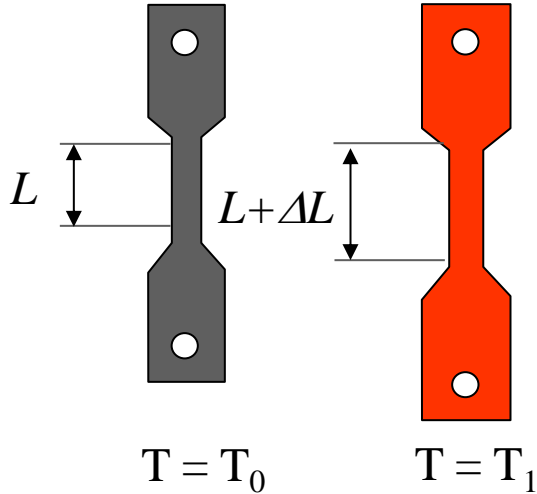
$$\sigma = E\varepsilon$$

Slope: elastic modulus E

Shaded area: elastic strain energy density W

Material	Elastic Modulus (GPa)	Poisson Ratio
Adhesive	0.01 – 10	0.4 – 0.49
Epoxy	10 – 30	0.35 – 0.48
Copper	120	0.35
Silicon	131	0.28
Solder	25 – 51	0.35
Substrate/PCB	11 – 26	various

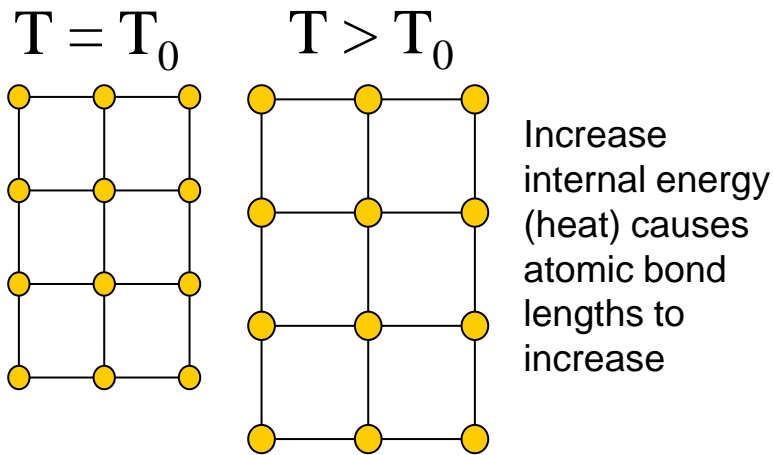
Coefficient of Thermal Expansion (CTE)



$$\varepsilon^{th} = \frac{\Delta L}{L}$$

$$\alpha = \frac{\varepsilon^{th}}{\Delta T}$$

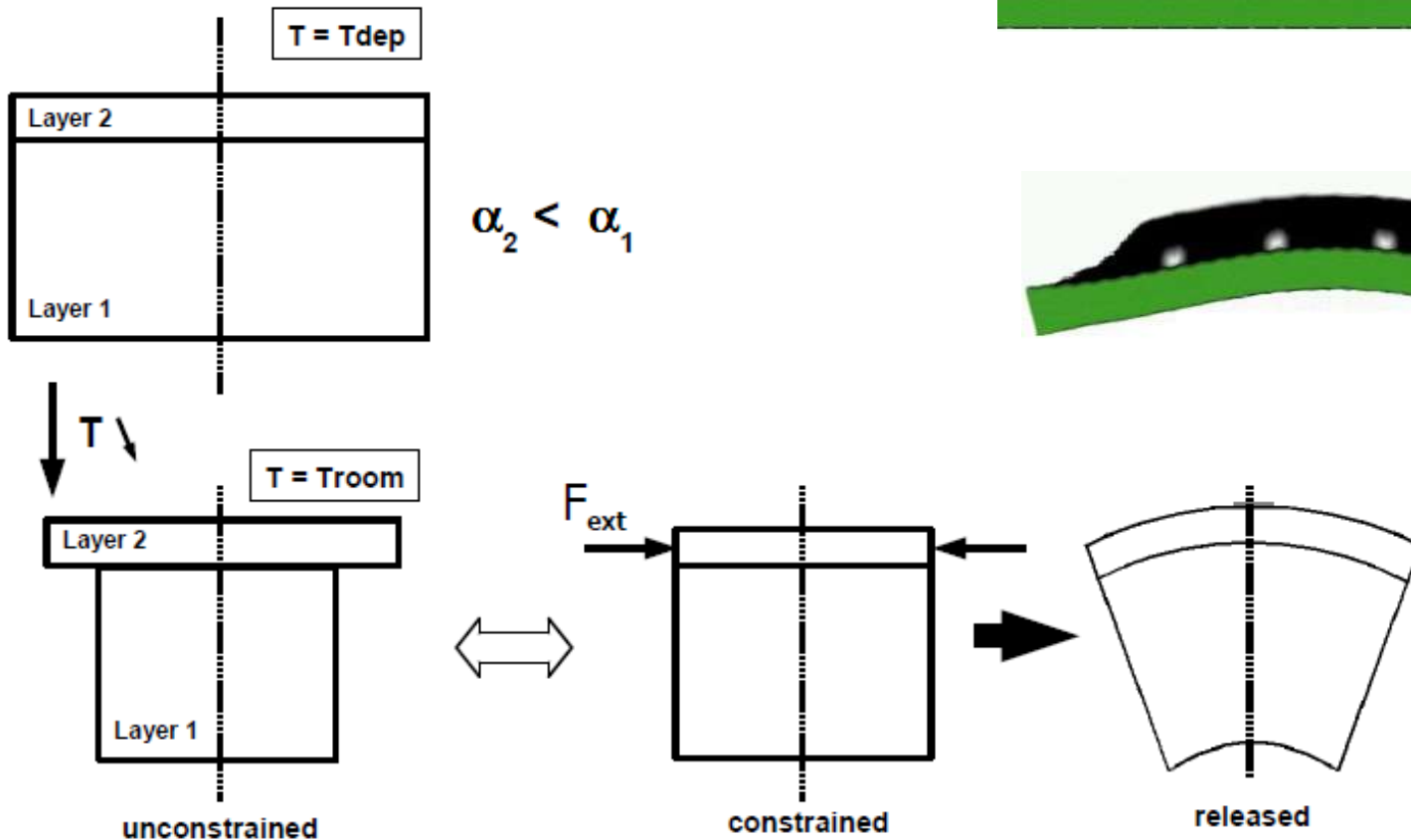
- Material dimensions change with temperature.
- The rate of dimensional change with temperature is called CTE.
- Primary driver of thermo-mechanical stress in electronic packages.



Material	α (ppm/°C)
Aluminum	23
Stainless Steel	12
Silicon	2.6
Copper	17
Underfill Epoxy	20, 70
FR4 (in plane)	15, 10
FR4 (z-axis)	65, 180

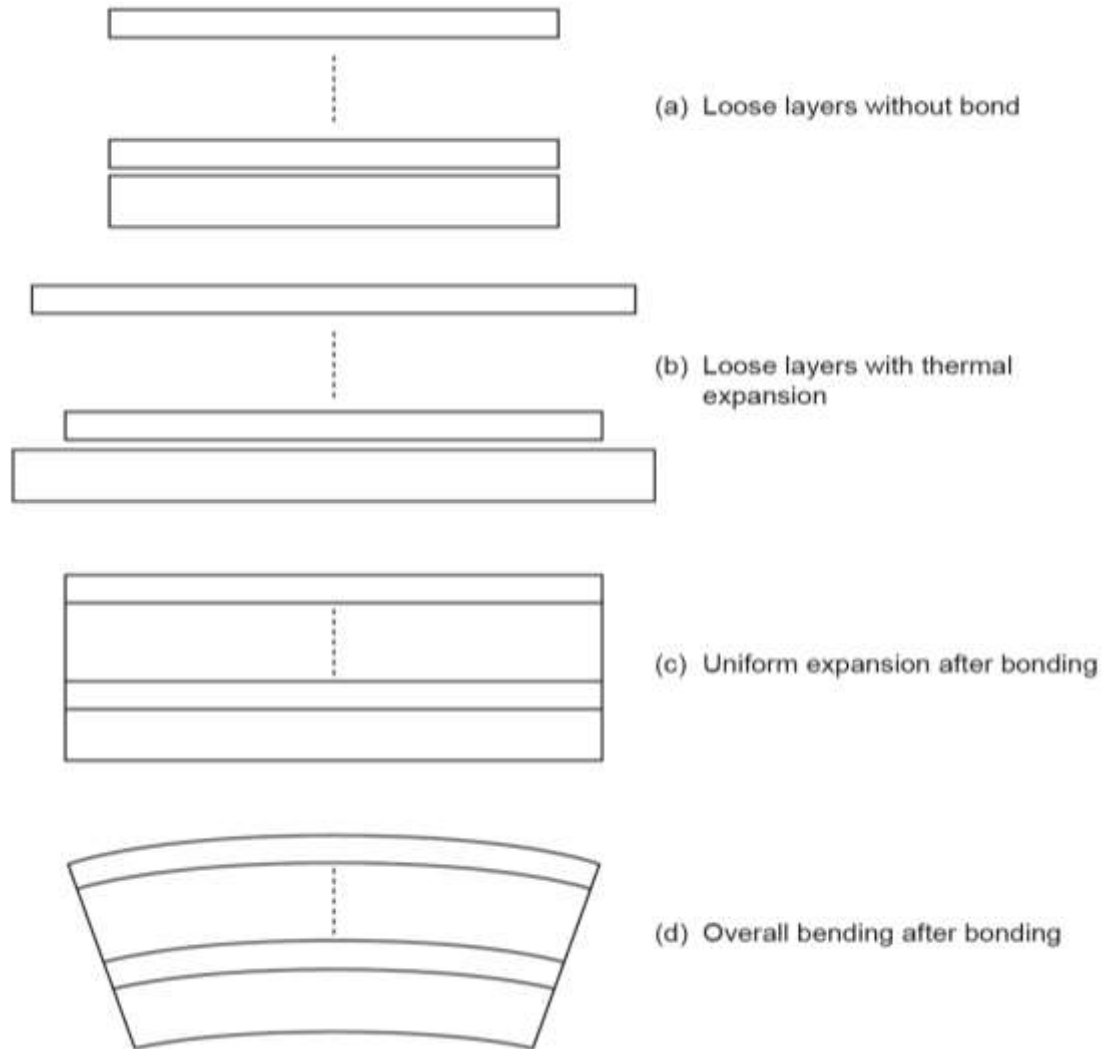
Thermal Mismatch – Two-Layer Structure

Superposition : axial loading + bending



Thermal-Mismatch – Multi-Layer Structure

Superposition : axial loading + bending



Thermal-Mismatch : Analytical Solution

Multi-layer structure

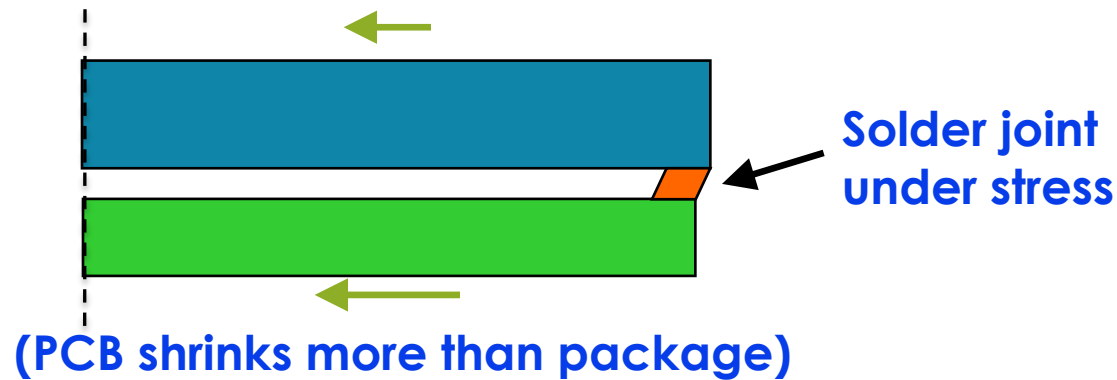
$$\left\{ \begin{array}{l} \varepsilon_{xi}(y) = (y - h_b)K + C, \text{ with } h_{i-1} \leq y \leq h_i \\ \sigma_{xi}(y) = \bar{Y}_i [(y - h_b)K + C + \nu_i A - \eta_i \varepsilon_i^0], \text{ with } h_{i-1} \leq y \leq h_i. \quad (\text{stress in each layer}) \\ h_b = \frac{\sum_{i=1}^n \bar{Y}_i D_i h_{mi}}{\sum_{i=1}^n \bar{Y}_i D_i} \\ C = \frac{\sum_{i=1}^n \bar{Y}_i D_i [\eta_i \varepsilon_i^0 - \nu_i A]}{\sum_{i=1}^n \bar{Y}_i D_i} \\ K = K_{nat} + K_{app} = \frac{\sum_{i=1}^n \bar{Y}_i D_i [h_{mi} - h_b] (\eta_i \varepsilon_i^0 - C - \nu_i A)}{\sum_{i=1}^n \bar{Y}_i D_i \left[h_i^2 + \frac{D_i^2}{3} - h_i D_i + h_b (h_b - 2h_{mi}) \right]} + K_{app} \\ \varepsilon_i^0 = \alpha_i (T_{room} - T_{depi}) + \varepsilon_i^{Btin} \end{array} \right.$$

- h_b is the Bending Axis of the composite device;
- C is the Uniform Strain of the composite device; (effective CTE)
- A is the Axial constant Strain of the composite device;
- K is the Curvature of the composite device; (warpage)
- K_{nat} is the component due to Natural bending of the curvature of the composite device;
- K_{app} is the Applied curvature of the composite device;
- ε_i^0 is the Stress-free Strain of layer i ;

Thermal Mismatch - Shear Stress and Strain on Solder Joint

A simplified and approximate solution

Cooling to a lower temperature



$$\gamma = \frac{L(\alpha_{\text{PCB}} - \alpha_{\text{chip}})\Delta T}{h}$$

$$\tau = G \cdot \gamma$$

τ : shear stress in solder

γ : solder ball shear strain

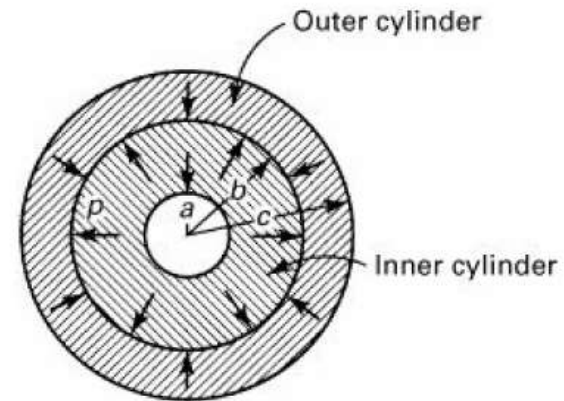
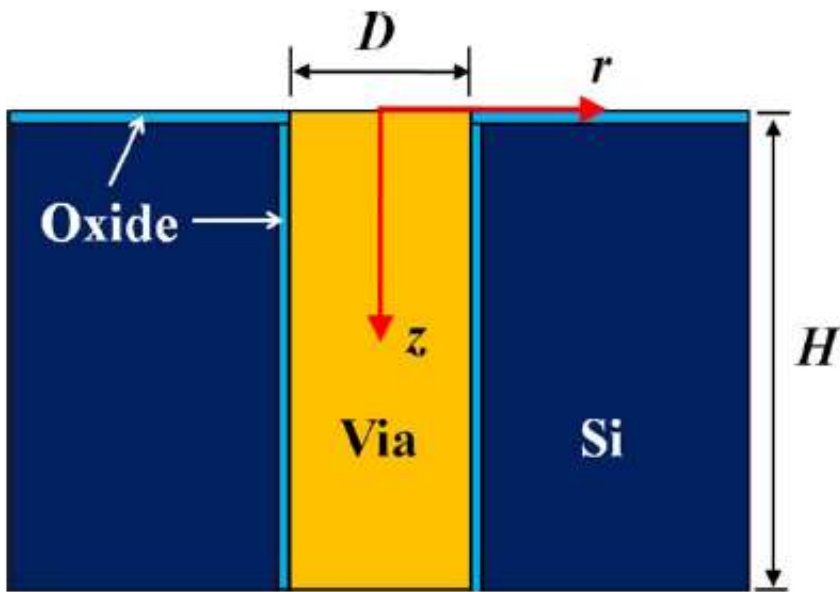
G: elastic shear modulus

L: distance of natural point (half-die size, diagonal)

h: solder ball stand-off

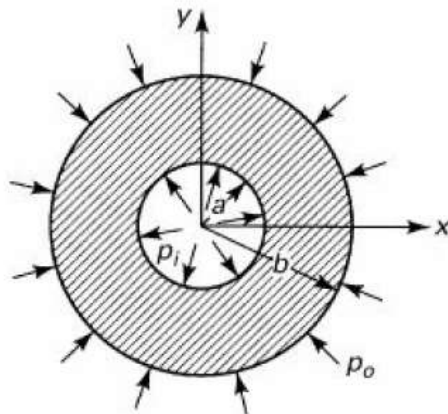
α : CTE

Thermal Mismatch – TSV Structure



Shrink fit of compound cylinder

$$\delta = \frac{bp}{E_o} \left(\frac{b^2 + c^2}{c^2 - b^2} + \nu_o \right) + \frac{bp}{E_i} \left(\frac{a^2 + b^2}{b^2 - a^2} - \nu_i \right)$$



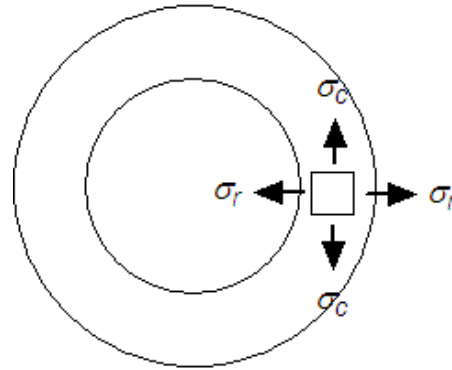
$$\sigma_r = \frac{a^2 p_i - b^2 p_o}{b^2 - a^2} - \frac{(p_i - p_o) a^2 b^2}{(b^2 - a^2) r^2}$$

$$\sigma_\theta = \frac{a^2 p_i - b^2 p_o}{b^2 - a^2} + \frac{(p_i - p_o) a^2 b^2}{(b^2 - a^2) r^2}$$

$$u = \frac{1 - \nu}{E} \frac{(a^2 p_i - b^2 p_o) r}{b^2 - a^2} + \frac{1 + \nu}{E} \frac{(p_i - p_o) a^2 b^2}{(b^2 - a^2) r}$$

Single cylinder under external and internal pressure

Thermal Stress due to Temperature Gradient



$$\sigma_r = \alpha E \left[-\frac{1}{r^2} \int_a^r T r \, dr + \frac{r^2 - a^2}{r^2(b^2 - a^2)} \int_a^b T r \, dr \right]$$

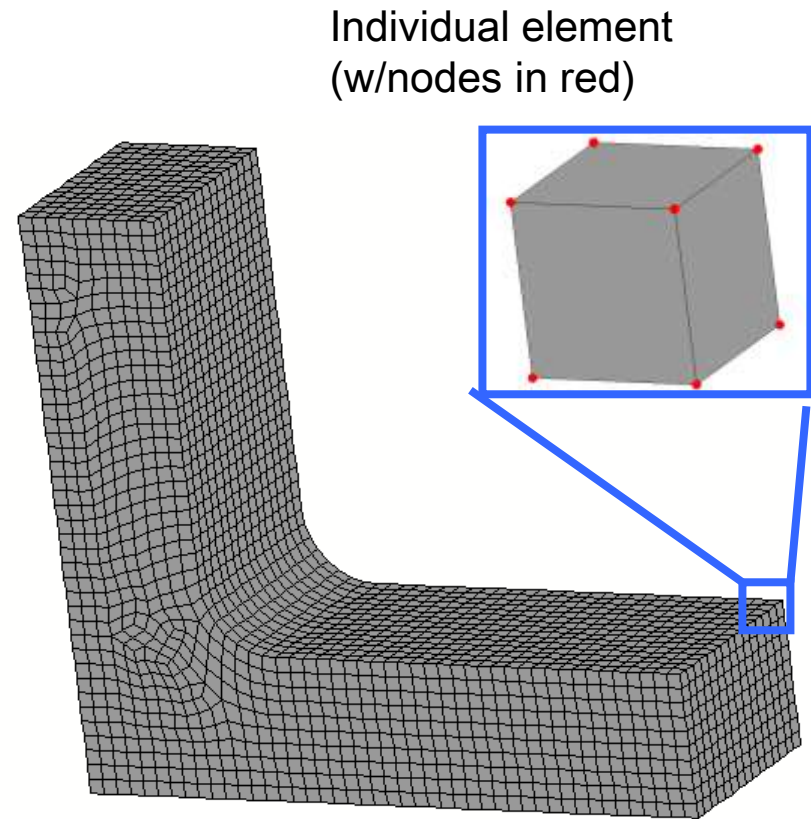
$$\sigma_\theta = \alpha E \left[-T + \frac{1}{r^2} \int_a^r T r \, dr + \frac{r^2 + a^2}{r^2(b^2 - a^2)} \int_a^b T r \, dr \right]$$

$$\sigma_r = \alpha E \left[\frac{1}{b^2} \int_0^b T r \, dr - \frac{1}{r^2} \int_0^r T r \, dr \right]$$

$$\sigma_\theta = \alpha E \left[-T + \frac{1}{b^2} \int_0^b T r \, dr + \frac{1}{r^2} \int_0^r T r \, dr \right]$$

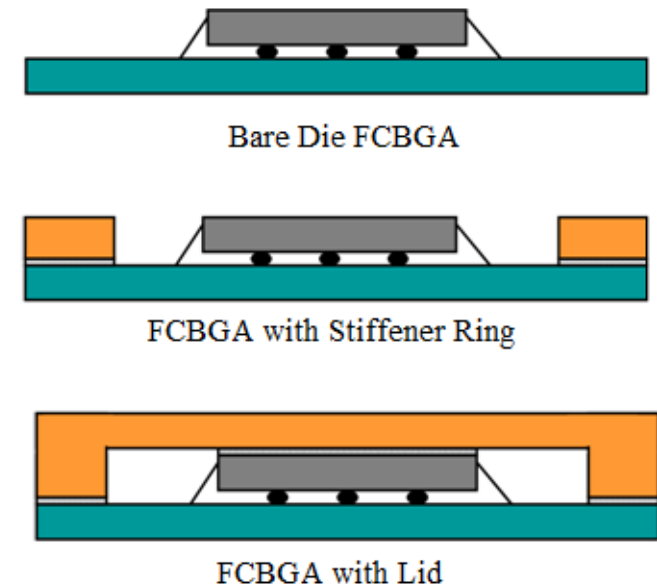
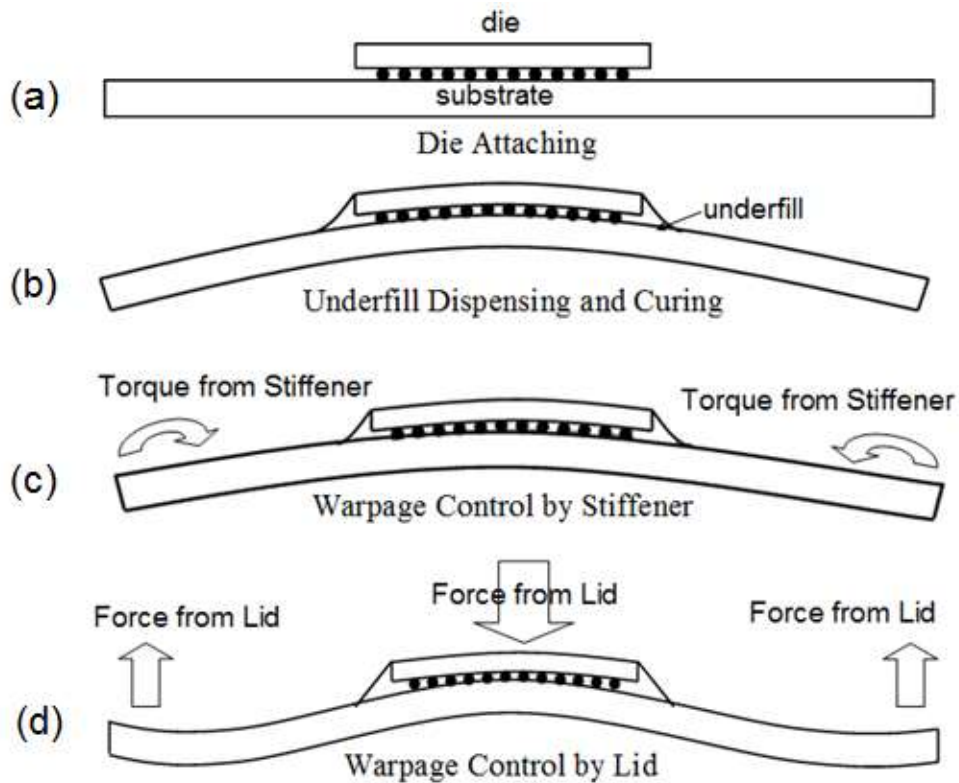
Finite Element Analysis (FEA)

- Analytical solutions are usually not applicable to complex problems.
- Based on dividing the problem into discrete elements for which analytical solutions exist.
- Analytical equations are then formed into a matrix and solved simultaneously.
- Method may require multiple iterations to converge and may have to divide “time” steps to reach a solution.
- Used for structural, thermal, EMI, fluid dynamics, diffusion, stock analysis, etc.

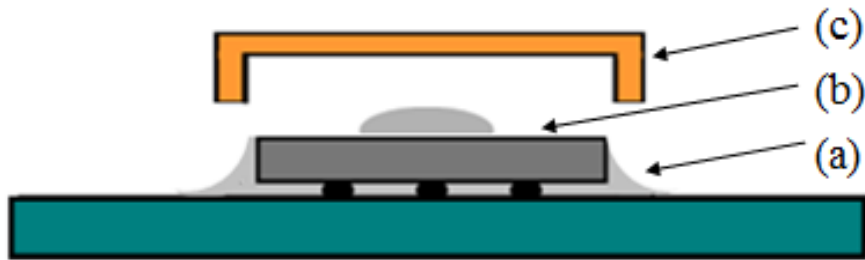


Bracket meshed into 11,295 elements.

Application – Warpage Analysis



Application – Warpage Control



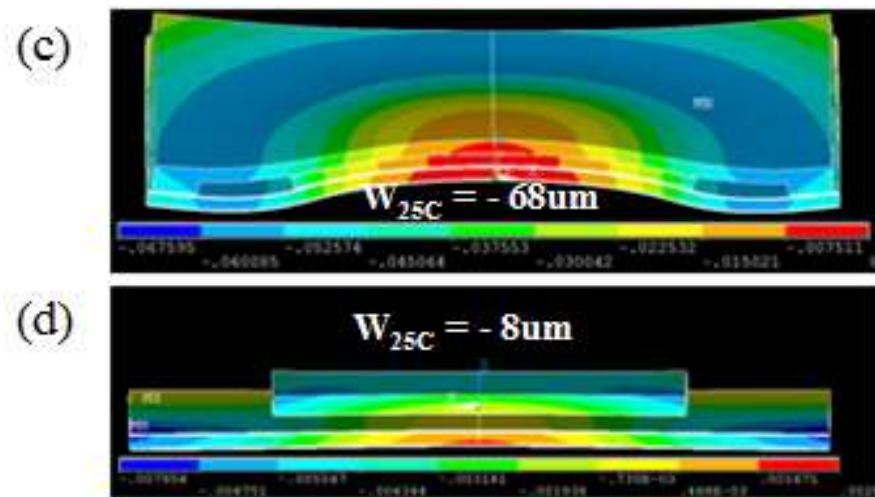
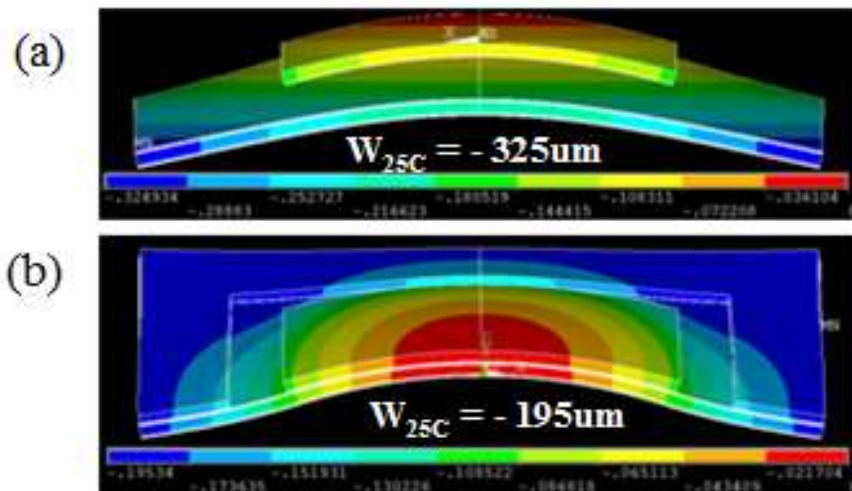
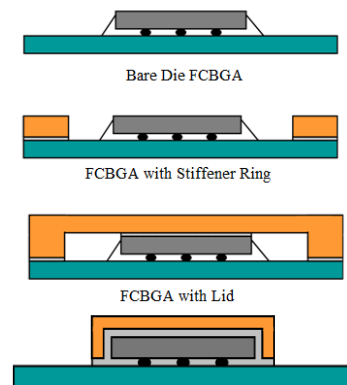
Capped-Die Flip Chip Package Design - assembly process

Using an underfill-like encapsulant, a metallic cap is attached to the die.

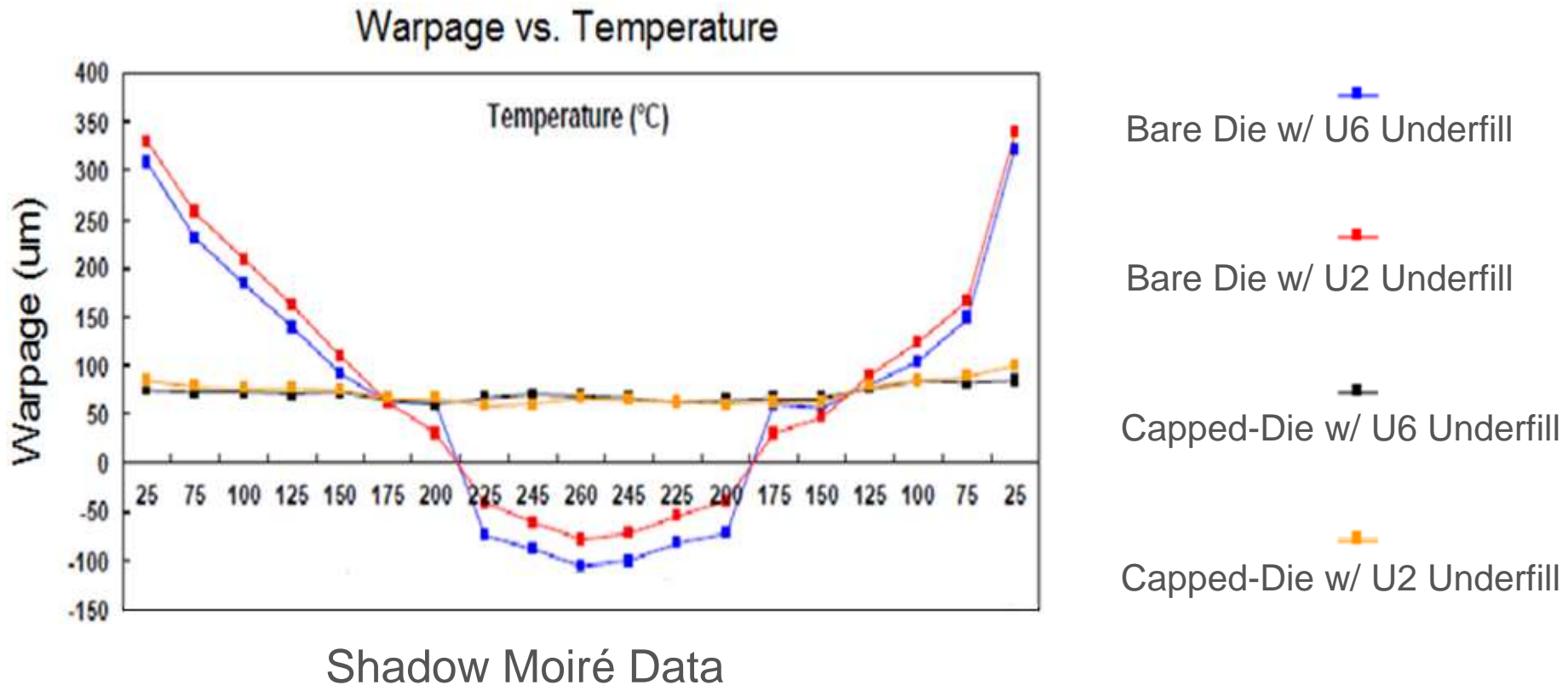


Comparison for Warpage Control Methods by FEA

- a) Bare die on substrate
- b) Package with stiffener
- c) Package with lid
- d) Capped-die package

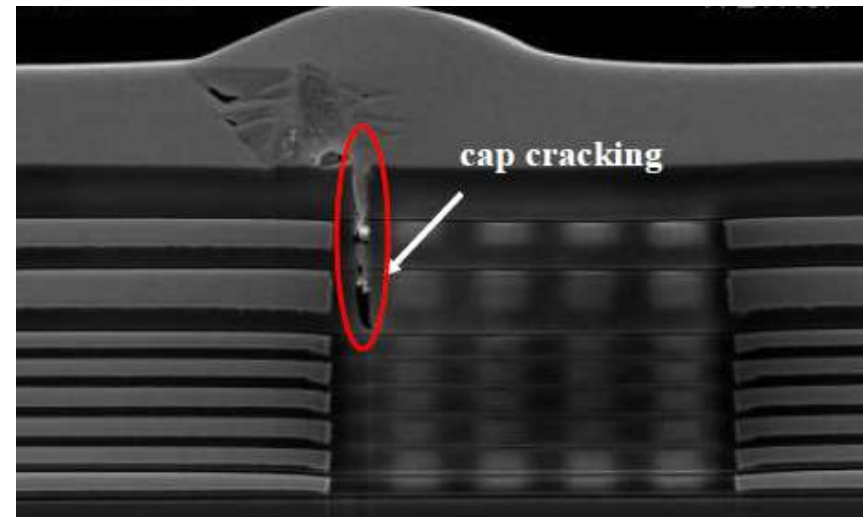
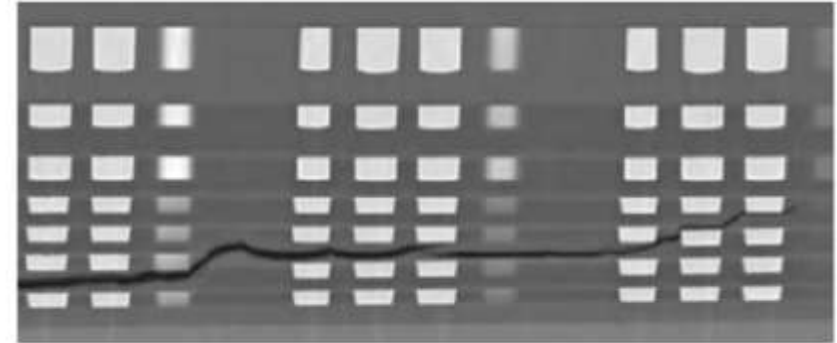
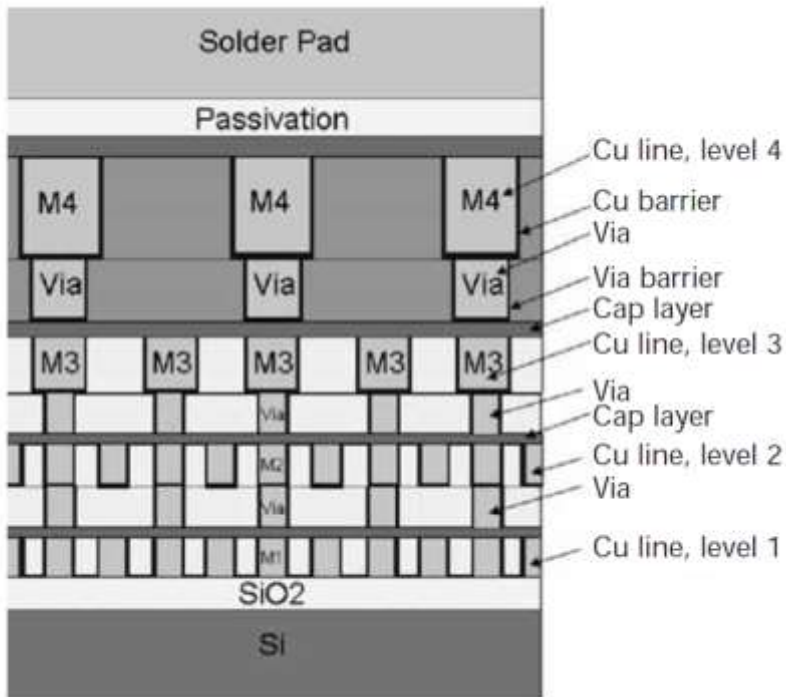


Experimental Verification - Warpage

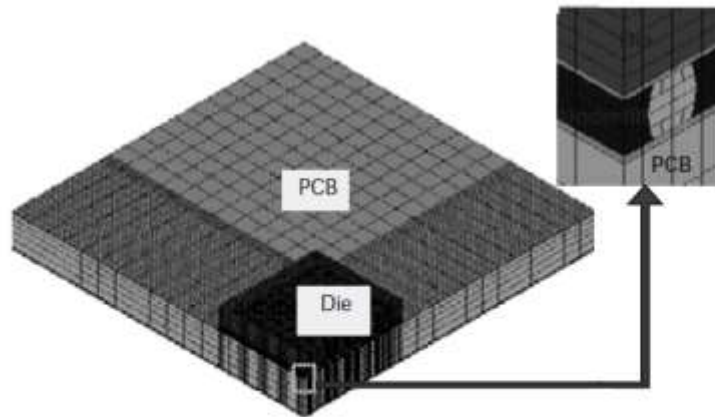


Shen Y, Zhang L, Zhu WH, Zhou J, Fan XJ. Finite-element analysis and experimental test for a capped-die flip-chip package design, *IEEE Transactions on Components, Packaging and Manufacturing Technology*. 6(9), 1308 – 1316. 2016.

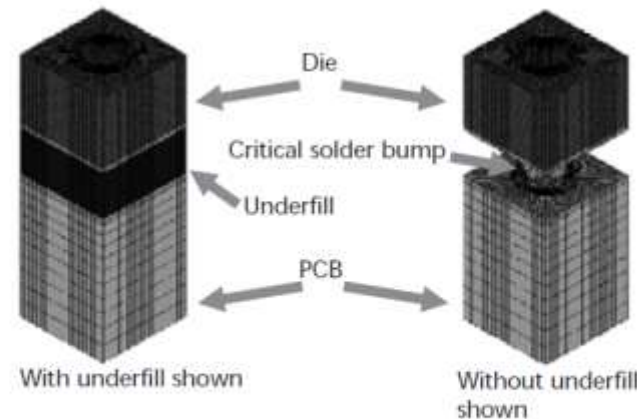
Chip-Package Interaction (CPI)



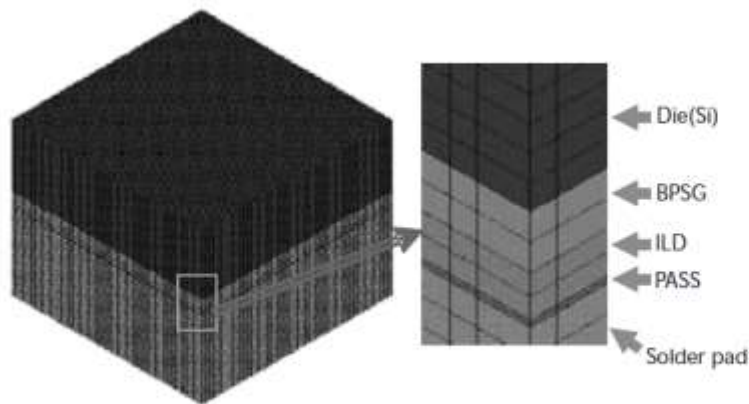
Multilevel Sub-modeling Technique



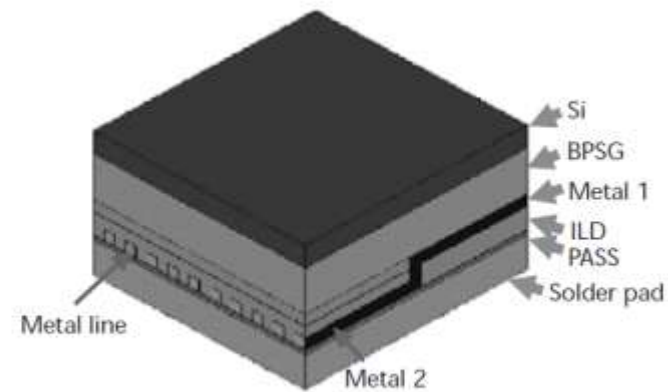
(a) package level



(b) critical solder level

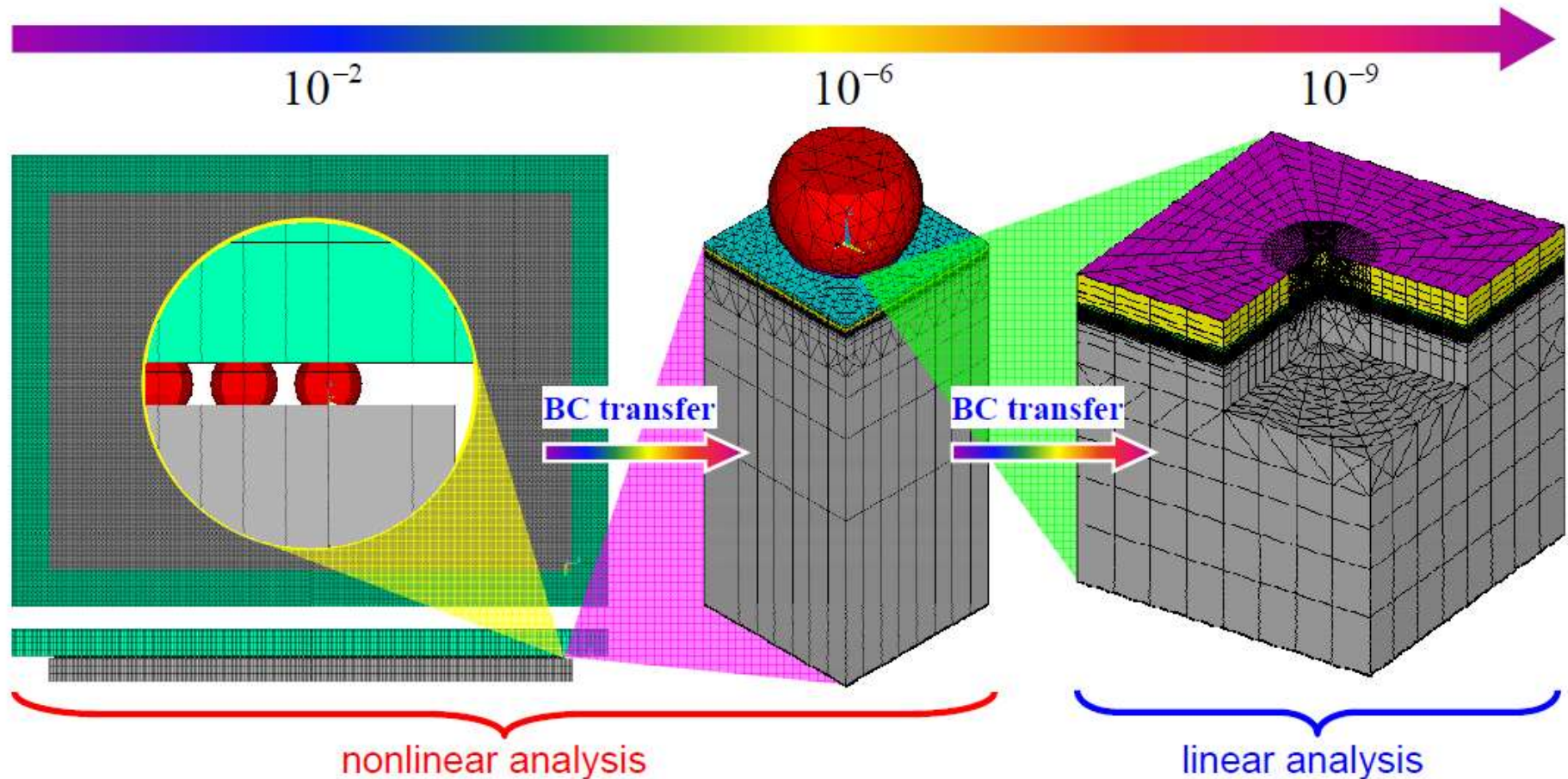


(c) die-solder interface level



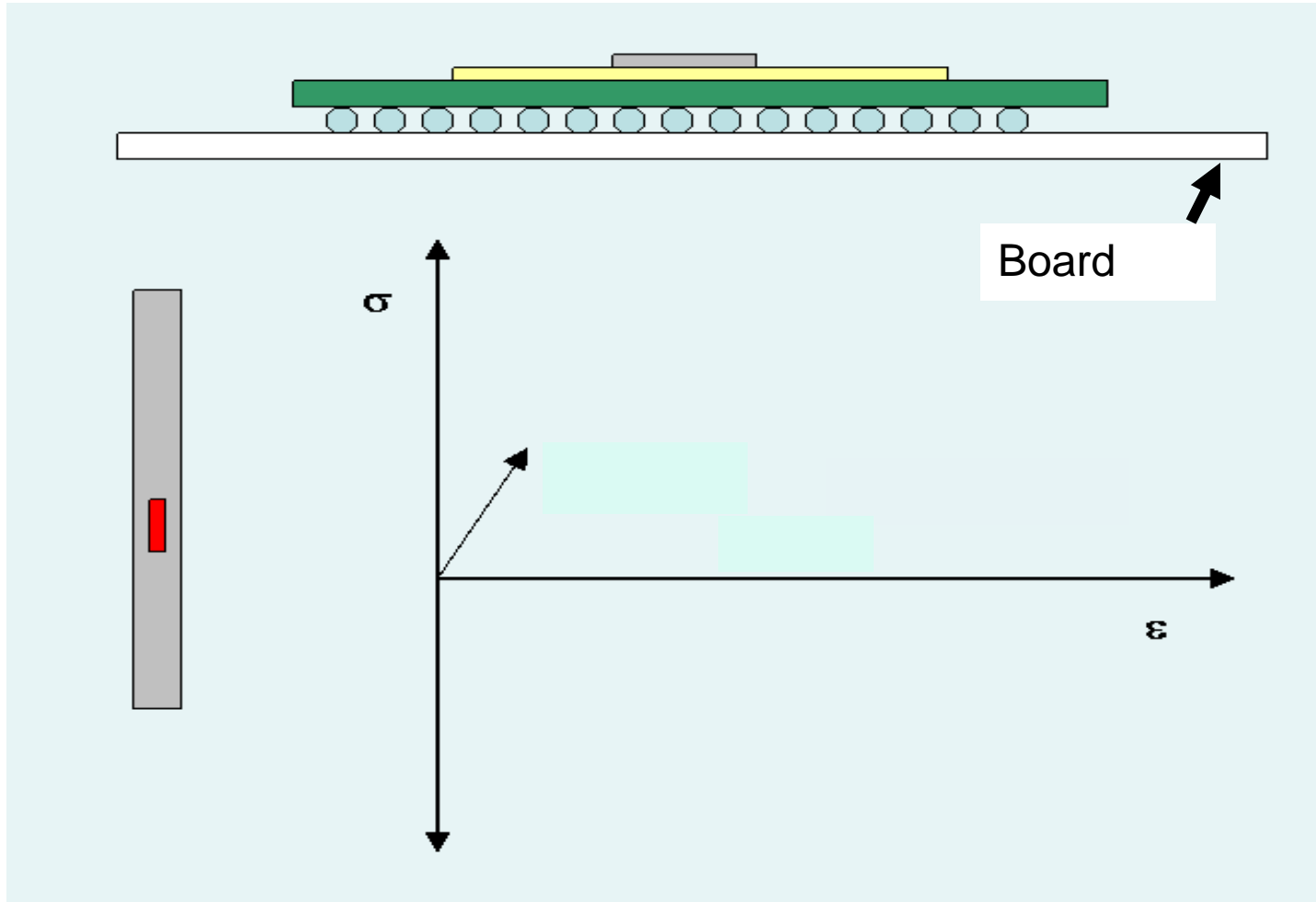
(d) interconnect level

Multilevel Sub-modeling Technique



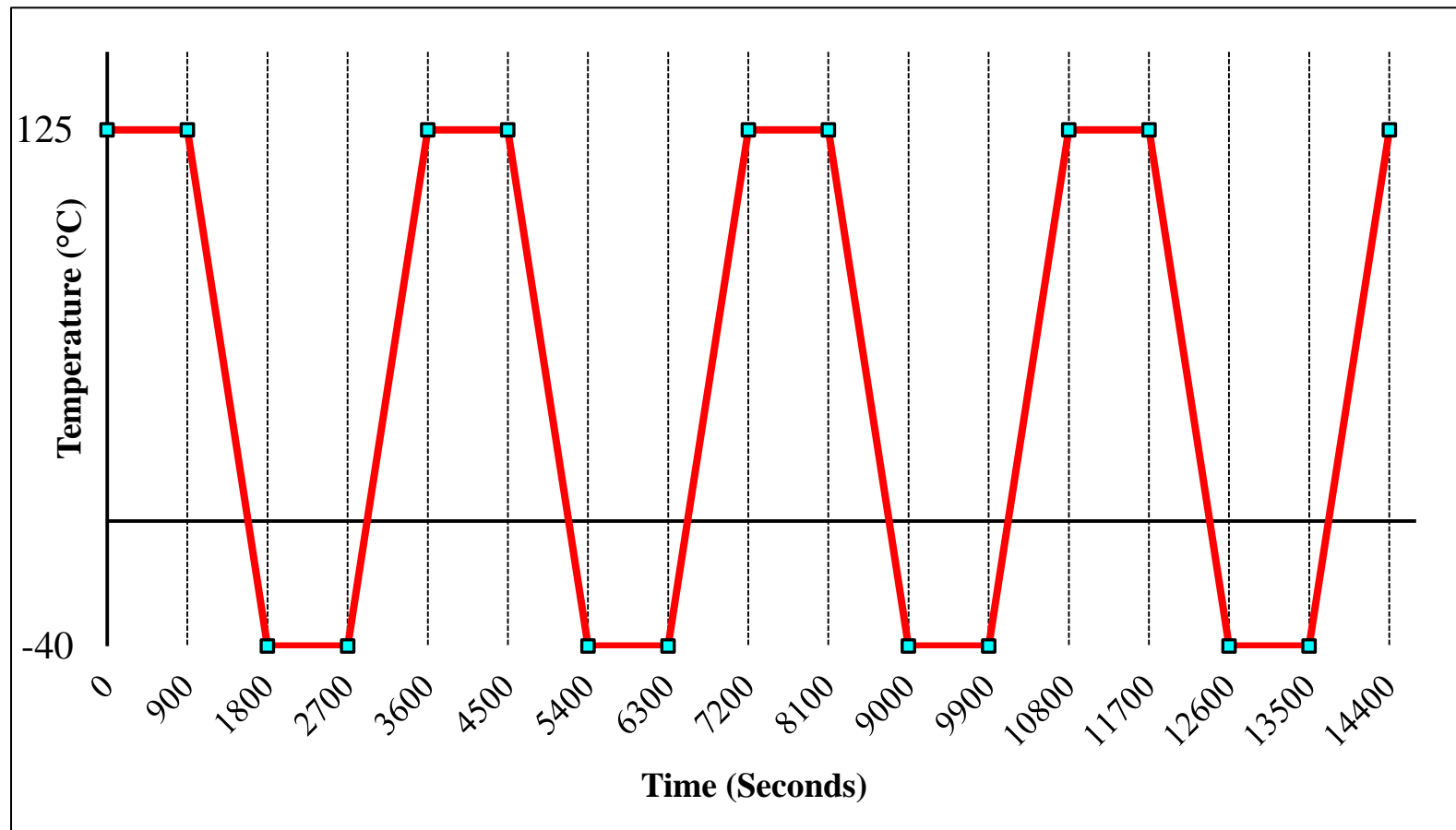
- ❑ Multilevel models are chained to obtain the driving force for delamination.
- ❑ Thousands of lines in ANSYS APDL codes have been written for the model.
- ❑ Typical model has one million DOF and takes a few hours to solve.

Temperature Cycling

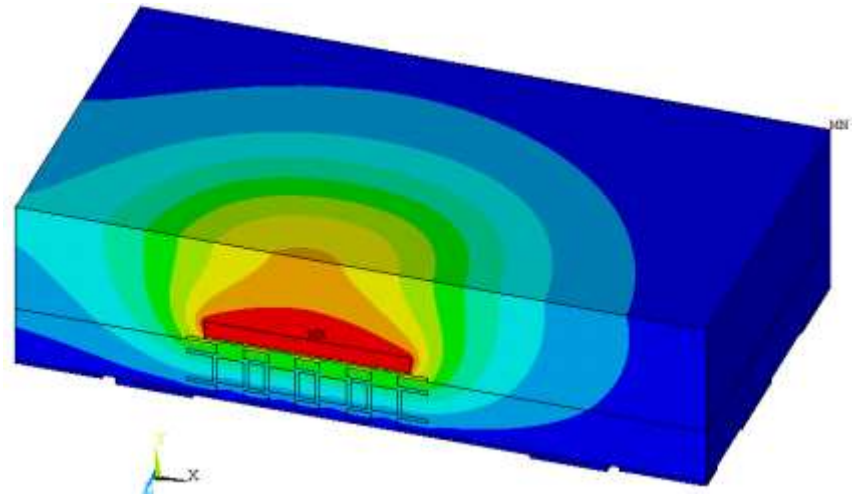


Temperature Cycling

- Temperature cycling – uniform temperature condition assumed

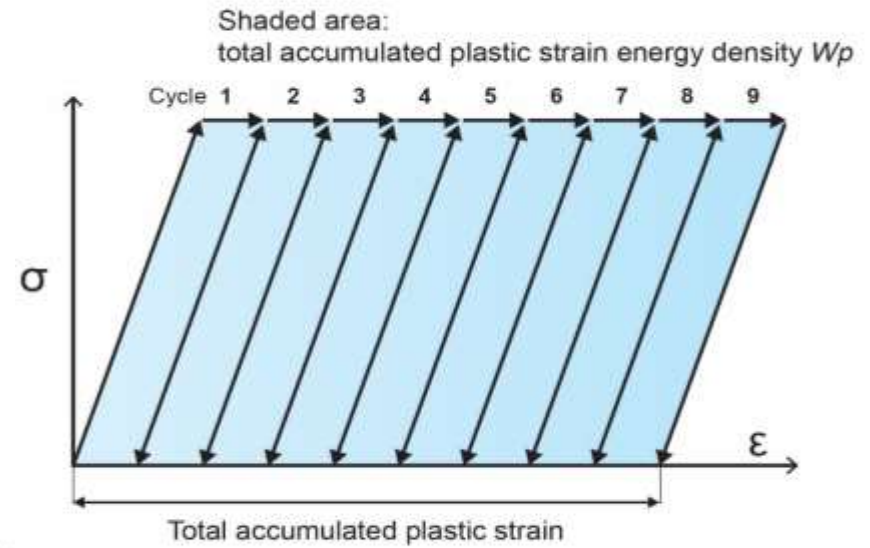
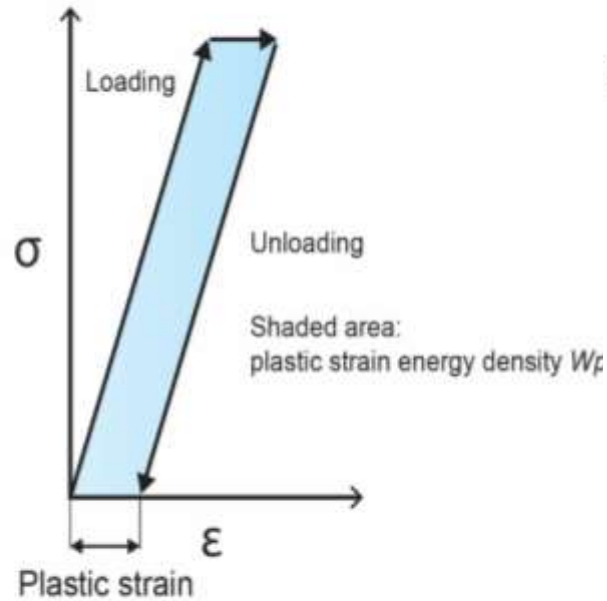


Power Cycling



- Temperature gradient exists.
- Much less severe than temperature cycling.

Fatigue Life Estimate



$$W_p = \int_0^{\varepsilon_p} \sigma d\varepsilon_p$$

$$N_f = A(\Delta W_p)^{-n}$$

Coffin-Manson Equation

Solder Creep Equations (ANAND Model)

- Anand's model (rate-dependent visco-plastic model)

$$\frac{ds}{dt} = \left\{ h_0 (|B|)^\alpha \frac{B}{|B|} \right\} \frac{d\varepsilon_p}{dt}$$

$$\frac{d\varepsilon_p}{dt} = A \left[\sinh \left(\frac{\xi \sigma}{s} \right) \right]^{-\frac{1}{m}} \exp \left(-\frac{Q}{RT} \right)$$

$$B = 1 - \frac{s}{s^*}$$

$$s^* = \hat{s} \left[\frac{d\varepsilon_p/dt}{A} \exp \left(-\frac{Q}{RT} \right) \right]^n$$

Variable/Parameter	Meaning	Units
s_0	initial value of deformation resistance	Stress
Q/R	Q = activation energy R = universal gas constant	Energy/volume Energy/(volume temp)
A	pre-exponential factor	1/time
ξ	multiplier of stress	Dimensionless
m	strain rate sensitivity of stress	Dimensionless
h_0	hardening / softening constant	Stress
\hat{s}	coefficient for deformation resistance saturation value	Stress
n	strain rate sensitivity of saturation (deformation resistance) value	Dimensionless
α	strain rate sensitivity of hardening or softening	Dimensionless

Anand Constants for Various Solder Alloys

Parameter	SAC105	SAC305	SAC387	SAC405	SnPb	SnPbAg	SnAg
s_0	2.3479	18.07	3.2992	20	12	12.41	39.09
Q/R	8076	9096	9883	10561	9200	9400	8900
A	3.773	3484	15.773	325	4.2E6	4E6	2.23E4
ξ	0.9951	4	1.0673	10	1.5	1.5	6
m	0.4454	0.2	0.3686	0.32	0.3	0.303	0.182
h_0	4507.5	144000	1076.9	8E5	1.4E3	1379	3321.15
\hat{s}	3.5833	26.4	3.1505	42.1	14	13.79	73.81
n	0.012	0.01	0.0352	0.02	0.071	0.07	0.018
α	2.1669	1.9	1.6832	2.57	1.31	1.3	1.82

SAC105 D. Bhate et al., *Constitutive Behavior of Sn3.8Ag0.7Cu and Sn1.0Ag0.5Cu Alloys at Creep and Low Strain Rate Regimes*, IEEE Transactions on Components and Packaging Technologies, Vol. 31, No. 3, September 2008, p. 622.

SAC305 Motalab et al., *Determination of Anand Constants for SAC Solders using Stress-Strain or Creep Data*, ITherm 2012, pp. 909-921.

SAC387 D. Bhate et al., *Constitutive Behavior of Sn3.8Ag0.7Cu and Sn1.0Ag0.5Cu Alloys at Creep and Low Strain Rate Regimes*, IEEE Transactions on Components and Packaging Technologies, Vol. 31, No. 3, September 2008, p. 622.

SAC405 W. Qiang et al., *Experimental determination and modification of Anand model constants for Pb-free material 95.5Sn4.0Ag0.5Cu*, in EUROSIME Conf. IEEE, 2007.

SnPb R. Darveaux, "Effect of simulation methodology on solder joint crack growth correlation," in Electronic Components & Technology Conf. IEEE, 2000.

SnPbAg R. Darveaux, "Effect of simulation methodology on solder joint crack growth correlation," in Electronic Components & Technology Conf. IEEE, 2000.

SnAg Wang, Z. Cheng, K. Becker, and J. Wilde, *Applying Anand model to represent the viscoplastic deformation behavior of solder alloys*, ASME Journal of Electronic Packaging, Vol. 123, 2001.

Solder Creep Equations

Solder alloys	C_1 (1/sec)	C_2 (1/Pa)	C_3	C_4 (°K)
95.5Sn-3.9Ag-0.6Cu	441000	5×10^{-9}	4.2	5412
63Sn-37Pb	$926(508 - T)/T$	$1/(37.78 \times 10^6 - 74414T)$	3.3	6360

$$\frac{\partial \varepsilon}{\partial t} = C_1 [\sinh(C_2 \sigma)]^{C_3} \exp\left(\frac{-C_4}{T}\right)$$

C_1 , C_2 , C_3 , C_4 are the input constants for ANSYS finite element program. The stress unit is Pa.

Lau, Dauksher, and Vianco, "Acceleration Models, Constitutive Equations, and Reliability of Lead-Free Solders And Joints", *IEEE Electronic Components and Technology Conference*, June 2003, pp. 229-234.

Solder Creep Equations (NEW)

Solder alloys	C_1 (1/sec)	C_2 (1/Pa)	C_3	C_4 (°K)	
95.5Sn-3.9Ag-0.6Cu	441000	5×10^{-9}	4.2	5412	Sandia
Sn(3.5-3.9)Ag(.5-.8)Cu	500000	10×10^{-9}	5.0	5807	Agilent
Fraunhofer's LF	277984	24.47×10^{-9}	6.41	6504	IZM

$$\frac{\partial \varepsilon}{\partial t} = C_1 [\sinh(C_2 \sigma)]^{C_3} \exp\left(\frac{-C_4}{T}\right)$$

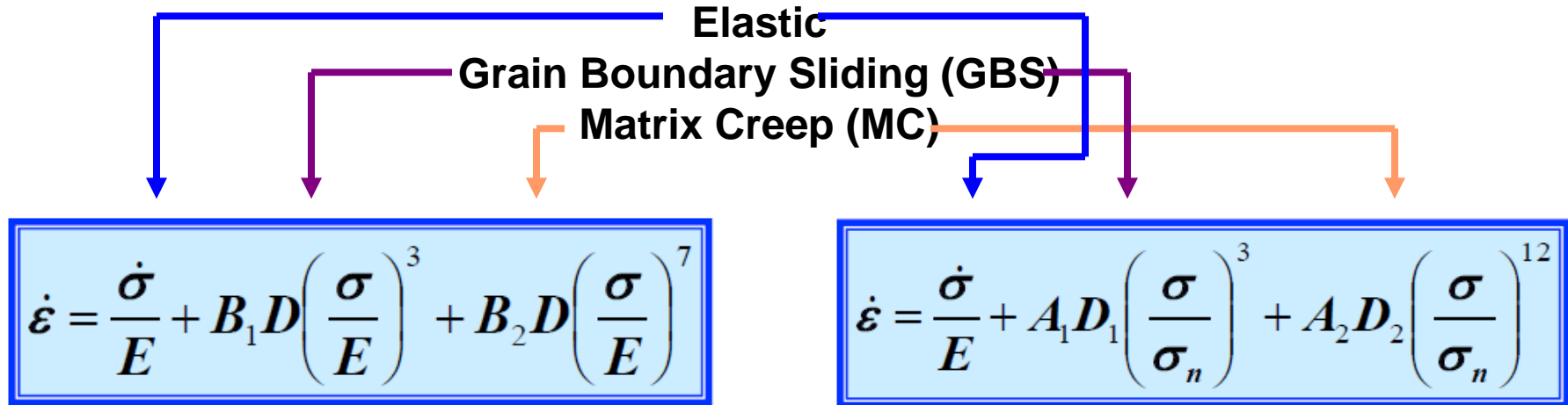
C_1 , C_2 , C_3 , C_4 are the input constants for ANSYS finite element program. The stress unit is Pa.

Lau, J., and W. Dauksher, "Creep Constitutive Equations of Sn(3.5-3.9)wt%Ag(0.5-0.8)wt%Cu Lead-Free Solder Alloys", in *Micromaterials and Nanomaterials*, edited by B. Michel, IZM, Berlin, Germany, 2004, pp. 54-62.

Solder Creep Equations

SnPb Creep Equation

SnAgCu Creep Equation



where

$\dot{\epsilon}$ = Total Strain Rate (1/sec)

σ = Stress (MPa)

E = Modulus of Elasticity (MPa) = $(56000 - 88T)$

T = Temperature (K)

$B_1 = 1.70 \times 10^{12}$ 1/sec

$B_2 = 8.90 \times 10^{24}$ 1/sec

$D = \exp\left(\frac{-5413}{T}\right)$

where

$\dot{\epsilon}$ = Total Strain Rate (1/sec)

σ = Stress (MPa)

E = Modulus of Elasticity (MPa) = $(59533 - 66.667T)$

T = Temperature (K)

$A_1 = 4.0 \times 10^{-7}$ 1/sec

$A_2 = 1.0 \times 10^{-12}$ 1/sec

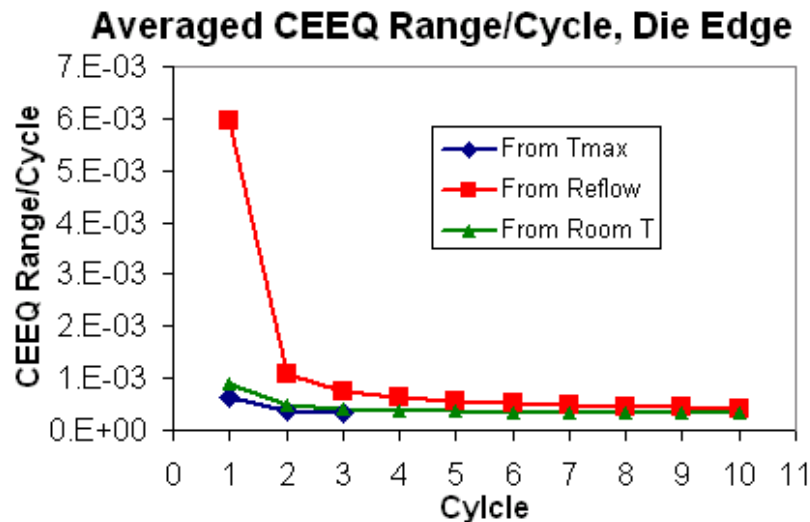
$D_1 = \exp(-3223/T)$

$D_2 = \exp(-7348/T)$

$\sigma_n = 1$ MPa

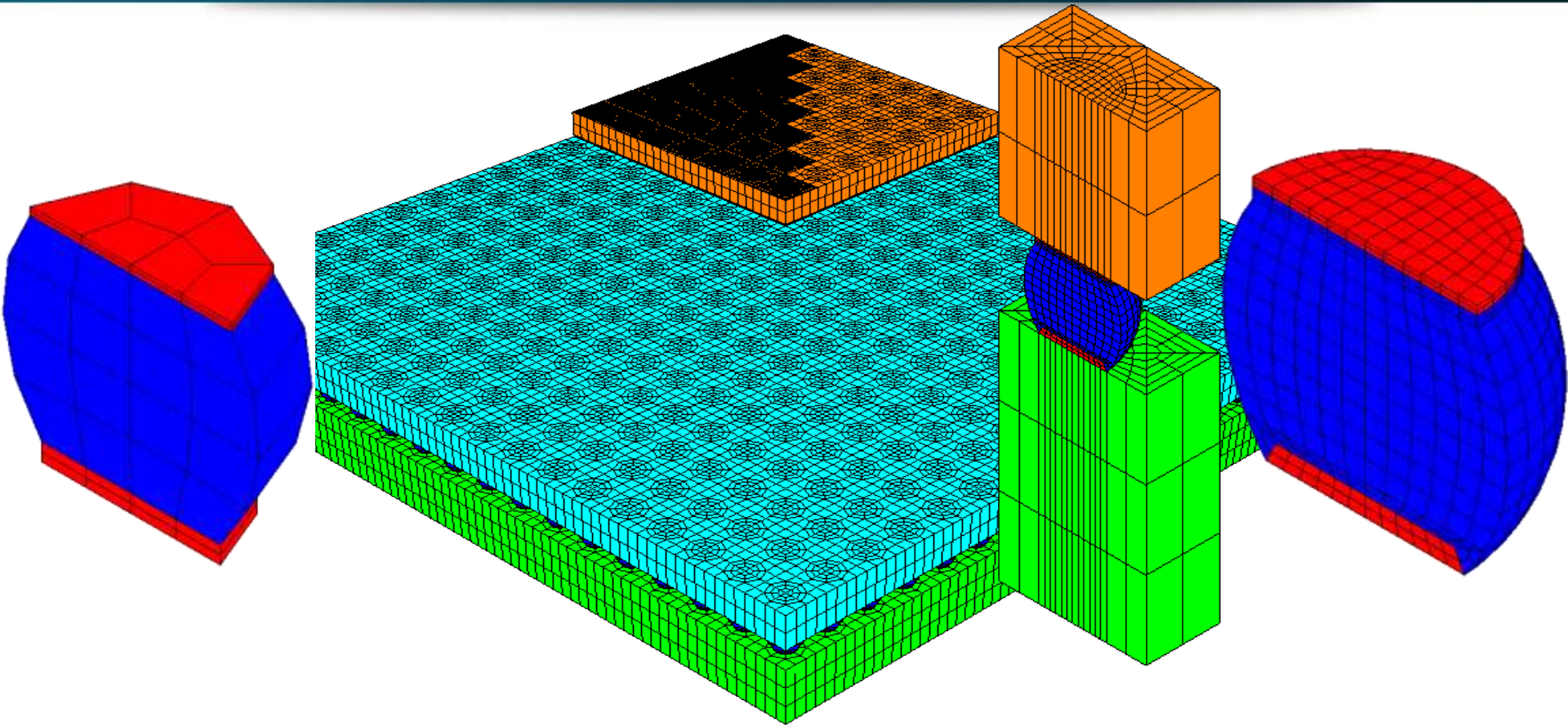
Best Practice (1) - Stress-Free Setting

- Three most commonly used initial stress-free temperatures
 - The solidus temperature of solder material (e.g., for SnAgCu, this temperature is 217°C)
 - The room temperature as initial stress-free (e.g. 25°C)
 - The high dwell temperature of thermal cycle or operating conditions (denoted as T_{max} , e.g. =125°C for thermal cycling from -25°C to 125°C)



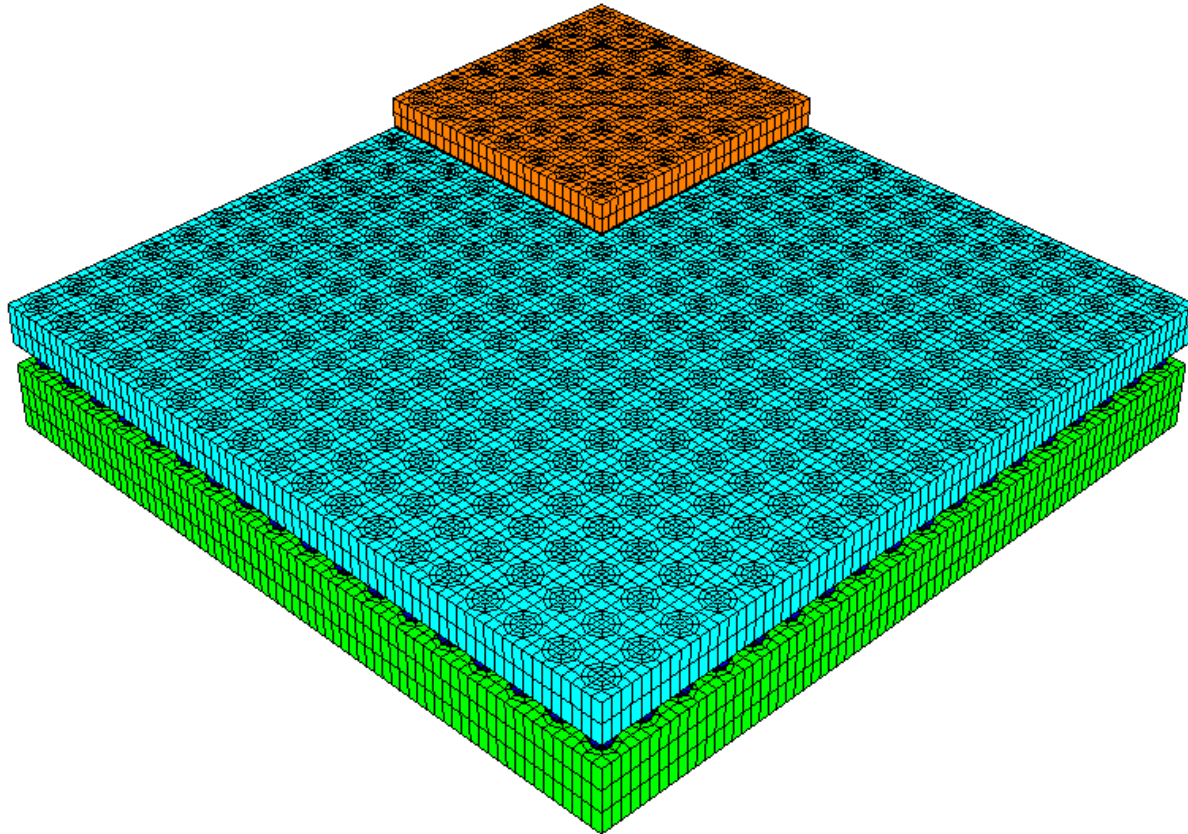
- **T_{max} as stress-free condition is recommended.**

Best Practice (2) – One Global Model (Full Model)

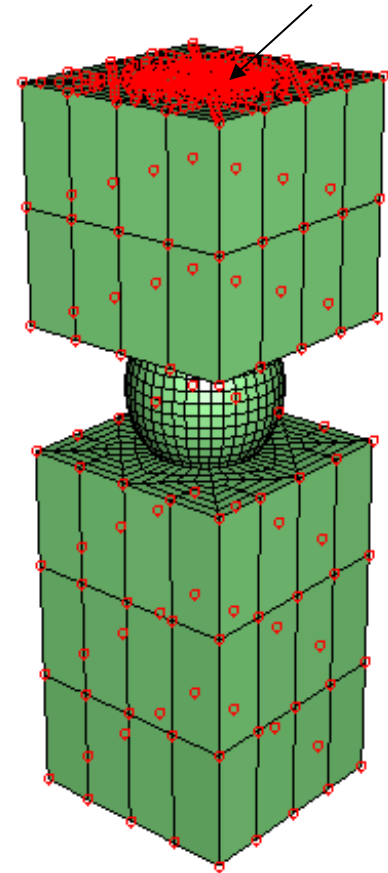


- Coarse element at most of the places
- Refined element at critical solder joints
- No tie element is used
- Refined solder balls consider both SMD and MD pads

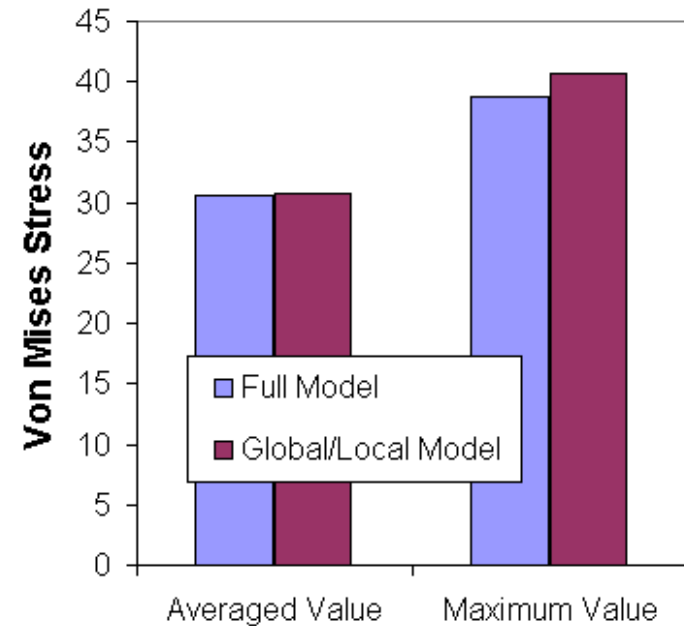
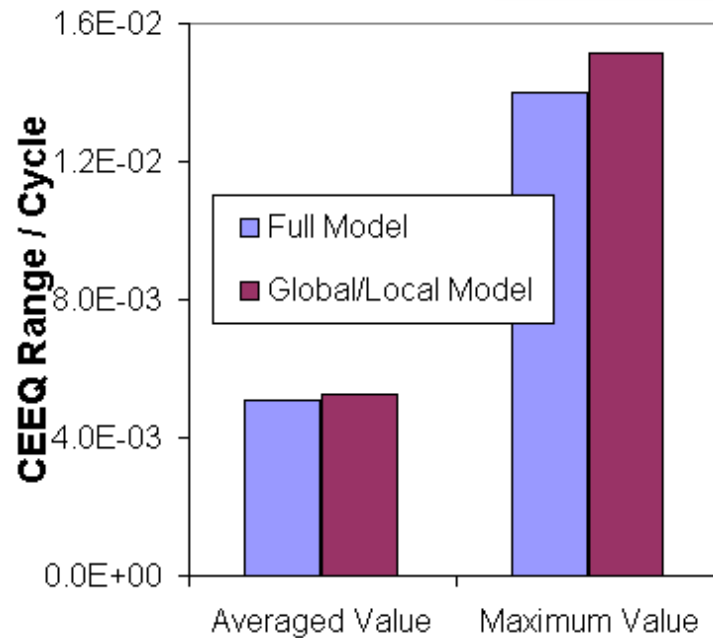
Best Practice (2) – Sub-modeling (Global/Local)



Substrate/underfill interface



Comparison - Global/Local Model and Full Model

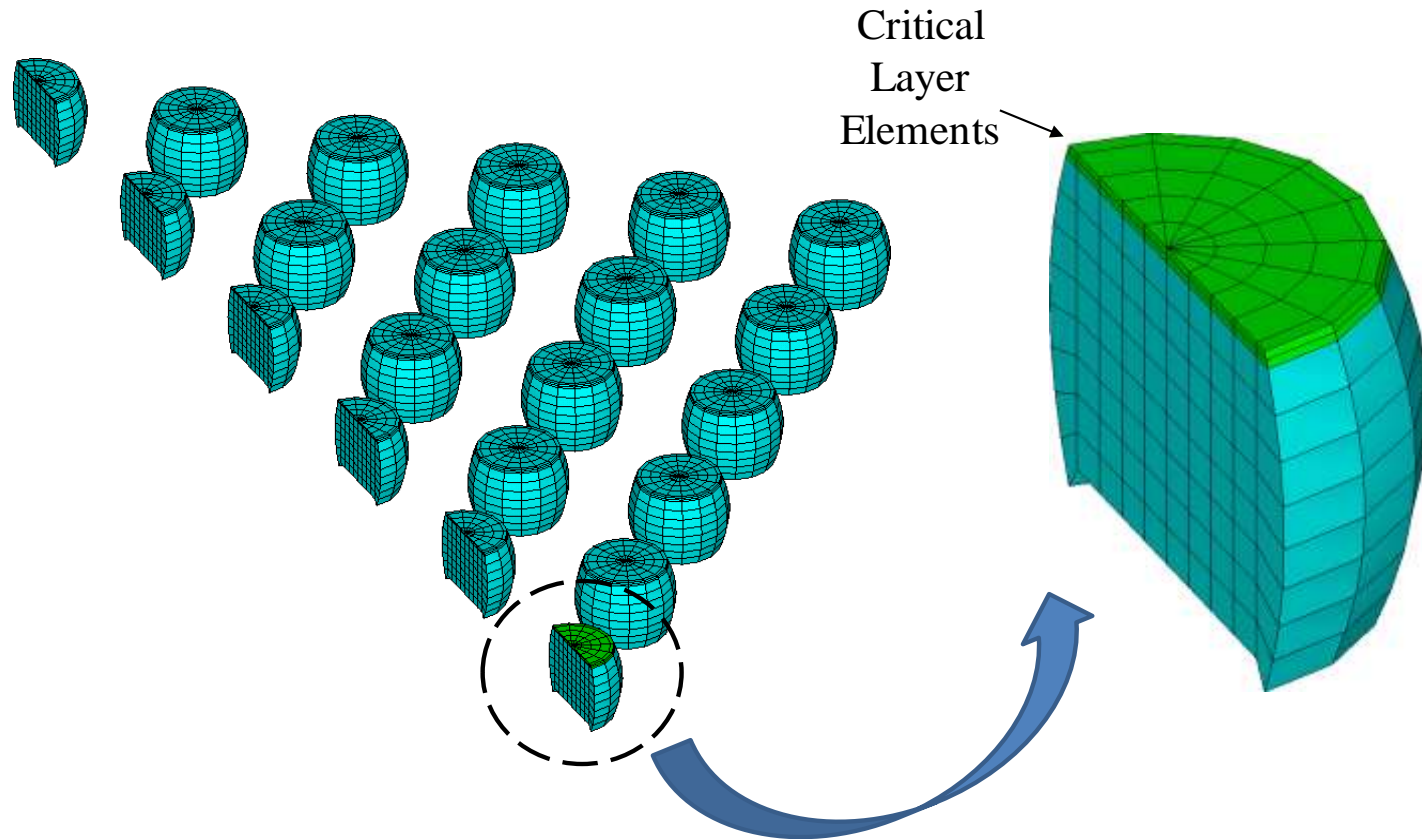


- The global/local modeling approach provides satisfactory results for the volumetrically averaged and maximum values
- For some location, the difference in maximum value can be as high as 20% (not shown here)

- **“Full Model” is recommended.**

Best Practice (3) - Volume Averaging

$$\Delta W_{ave} = \frac{\sum \Delta W_i V_i}{V_i}$$



ΔW_{ave} = average inelastic strain energy density accumulated per cycle for fixed thickness layer elements

ΔW_i = strain energy density accumulated per cycle for each element i

V_i = volume of each element i

- **Volume averaging over a fixed thickness of thin layer is recommended.**

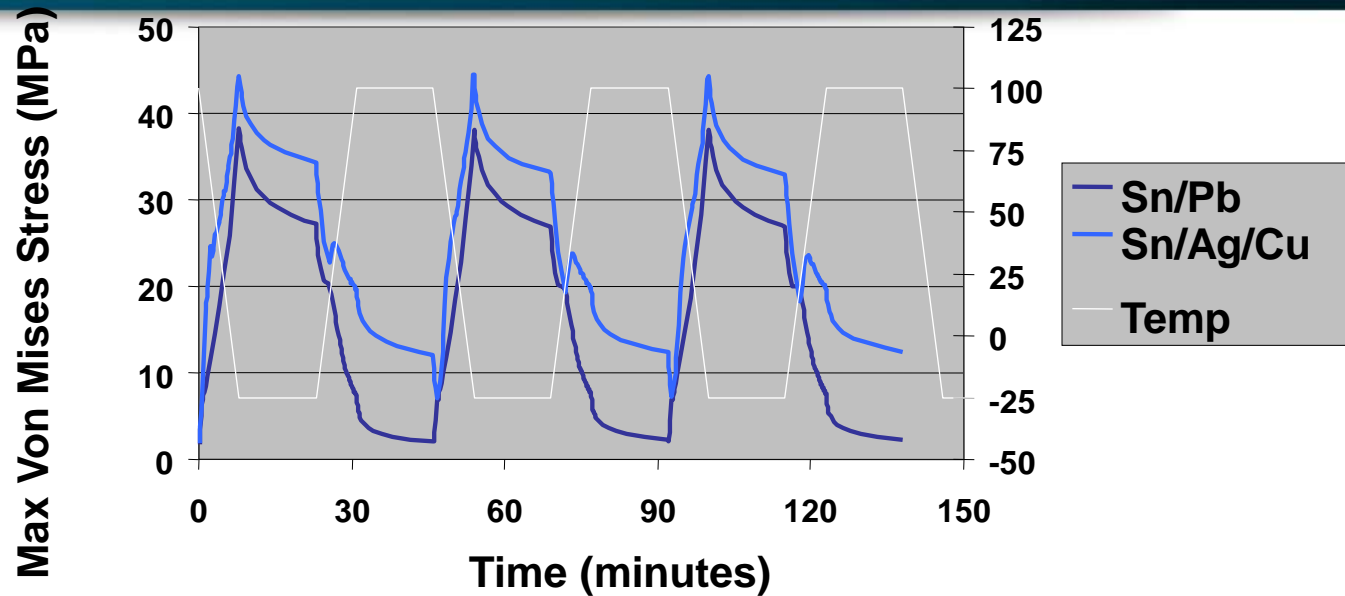
Best Practice (4) - Worst Case Solder Joint Location

- For FC-BGA package
 - It's commonly accepted that the worst case solder joint is located in the outermost along diagonal direction under the die shadow.
- Metrics
 - Von Mises stress, creep strain, creep strain energy density, peel stress
 - Maximum value, averaged value
- Results
 - Averaged per-cycle creep strain and stress shows the worst-case solder joint at the middle of die edge
 - The maximum value of these two parameters show the worst joint at the die corner
 - Peel stress finds the maximum tensile stress at the joint one row inside from the die shadow corner
- Experimental data
 - The solder joint one row inside from the corner of die shadow usually has the highest crack growth rate
 - All solder joints along the die edge have comparable crack growth rates
- **Correlation with experimental data is recommended.**

Bhatti PK, Pei M, Fan XJ. Reliability analysis of SnPb and SnAgCu solder joints in FC-BGA packages with thermal enabling preload. Proc. of Electronic Components and Technology Conference (ECTC), 601-606. 2006.

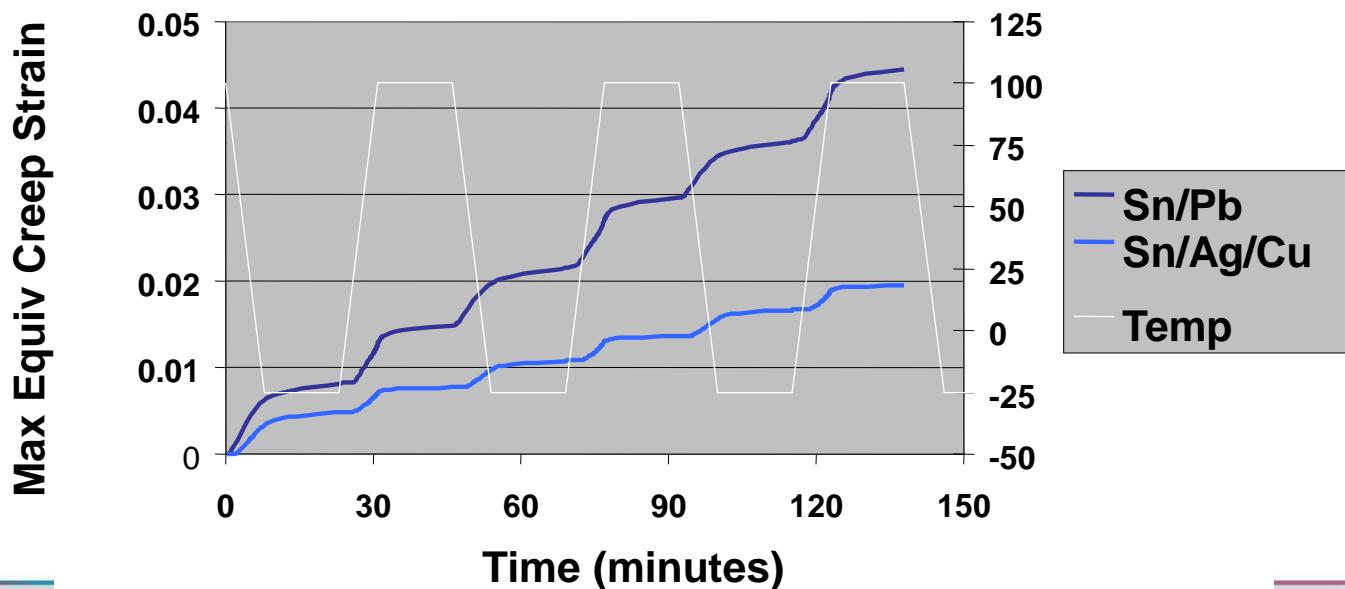
General Results: SnPb vs. SnAgCu

Maximum Von Mises Stress (MPa)

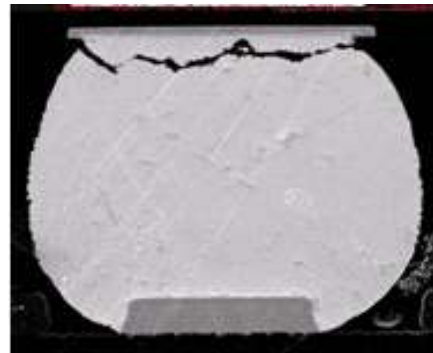
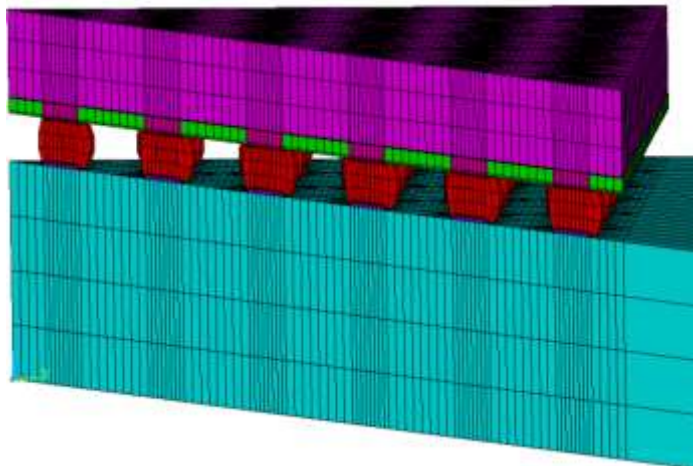
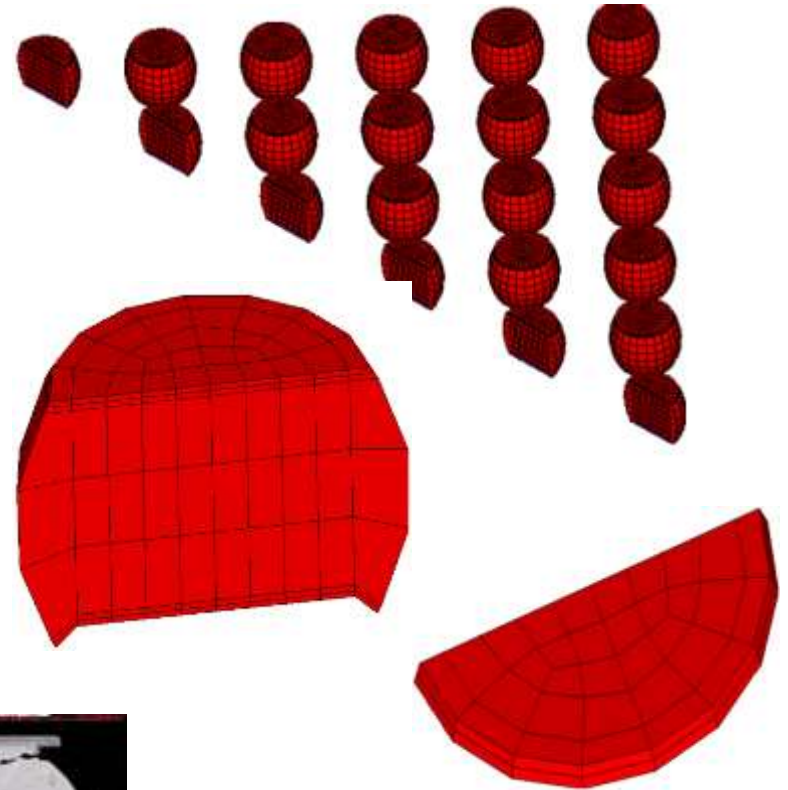
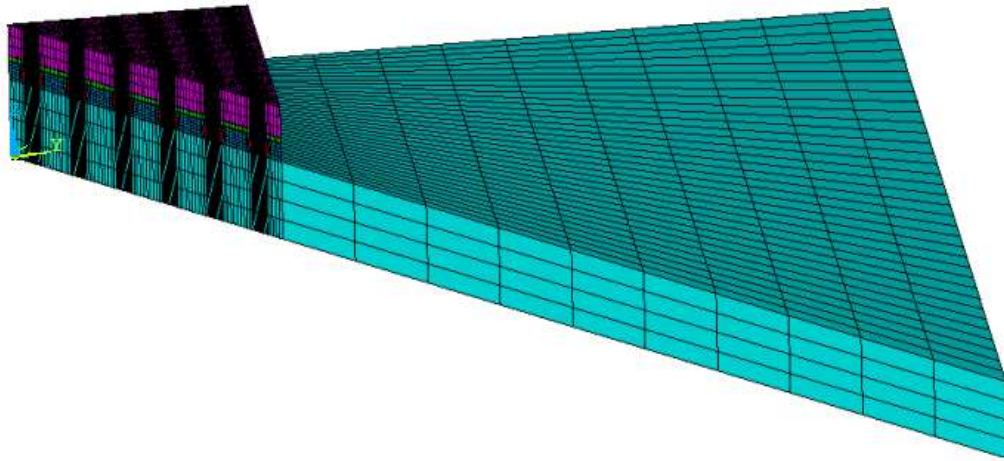


Temperature range
-25 to +100°C

Maximum Equiv Creep Strain



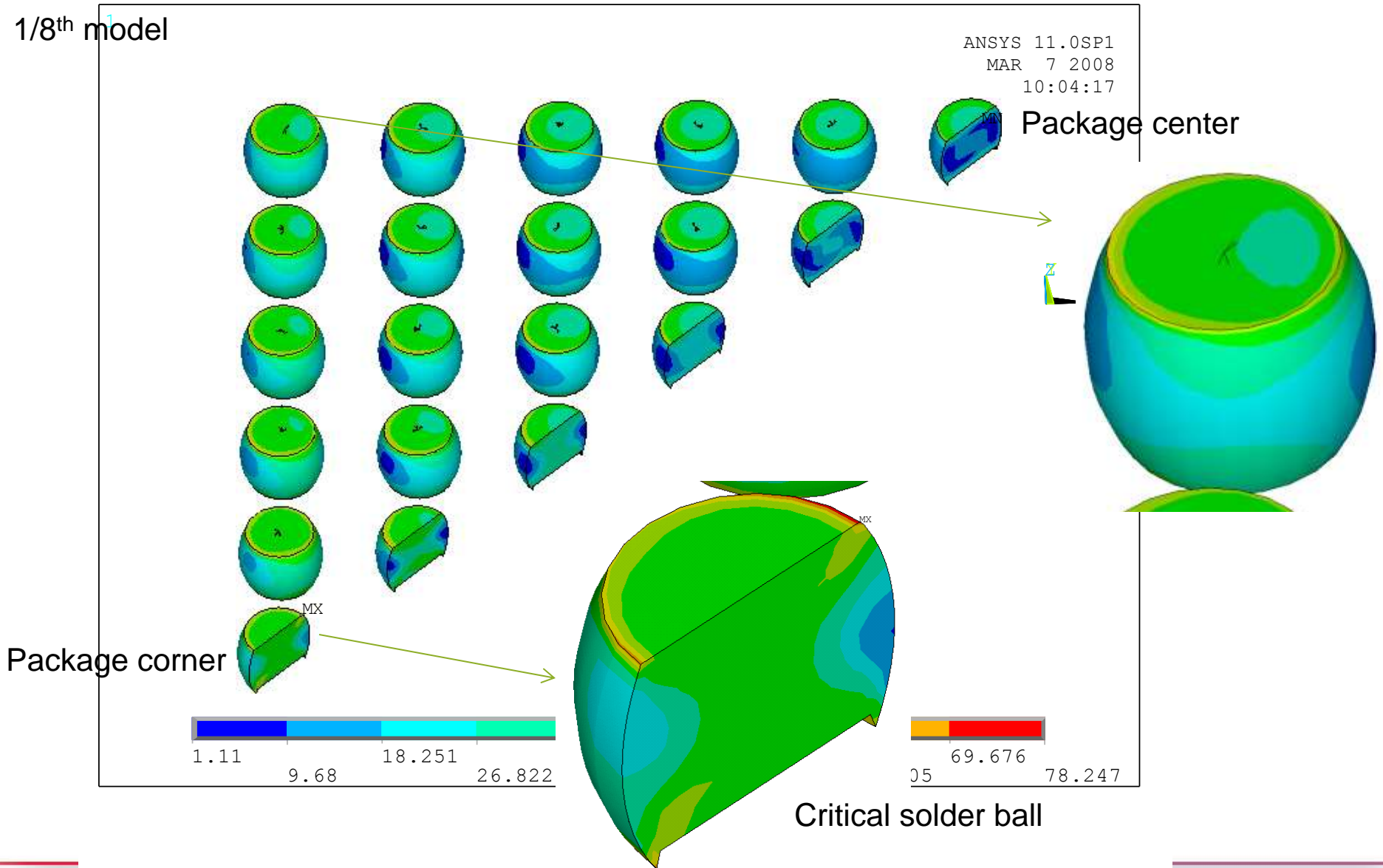
Wafer Level Package (Fan-in WLP) – Worst Location



$$\Delta W_{ave} = \frac{\sum \Delta w \cdot v}{\sum v}$$

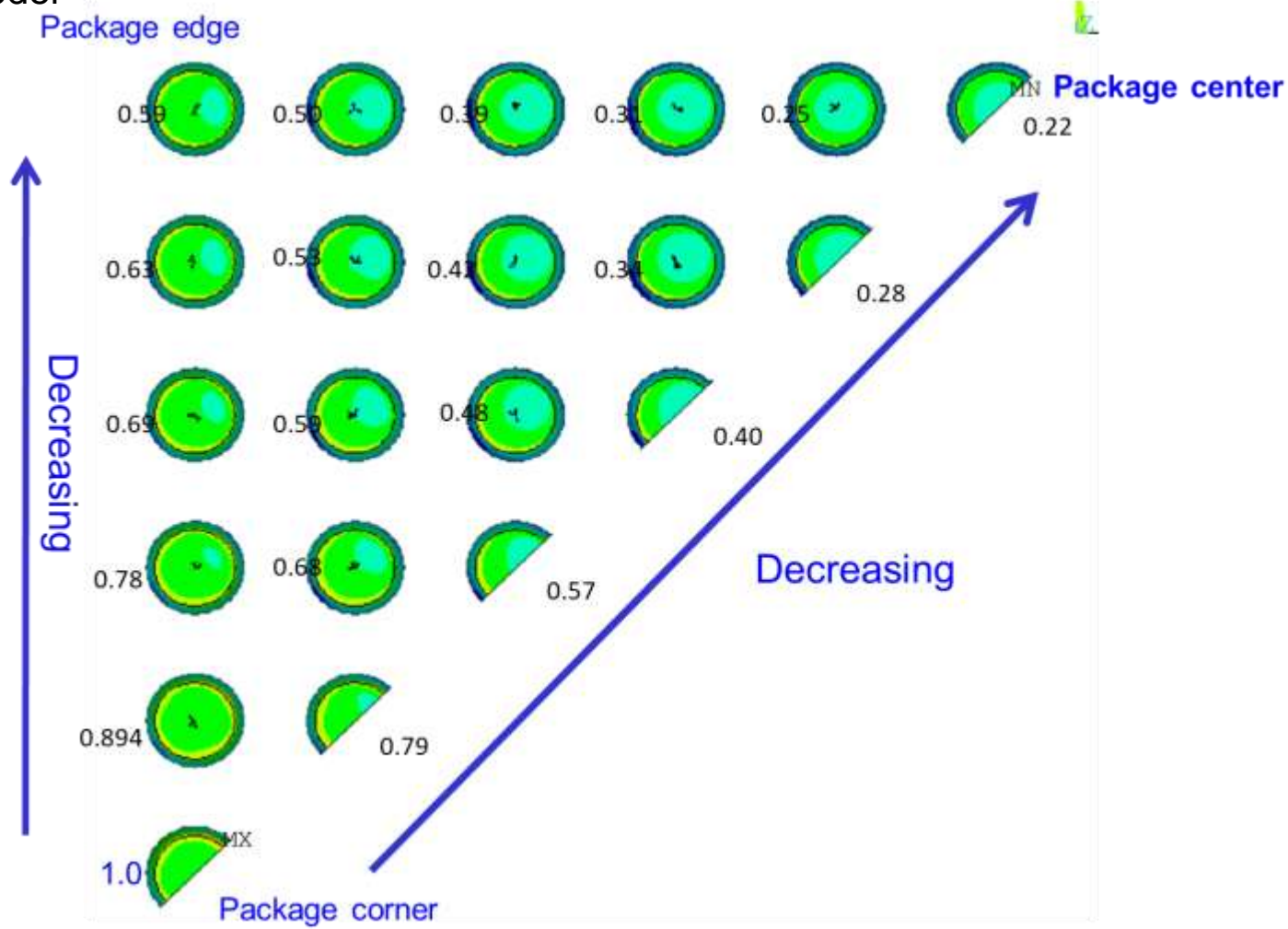
WLP - von Mises Stress Map in Solder Balls

1/8th model



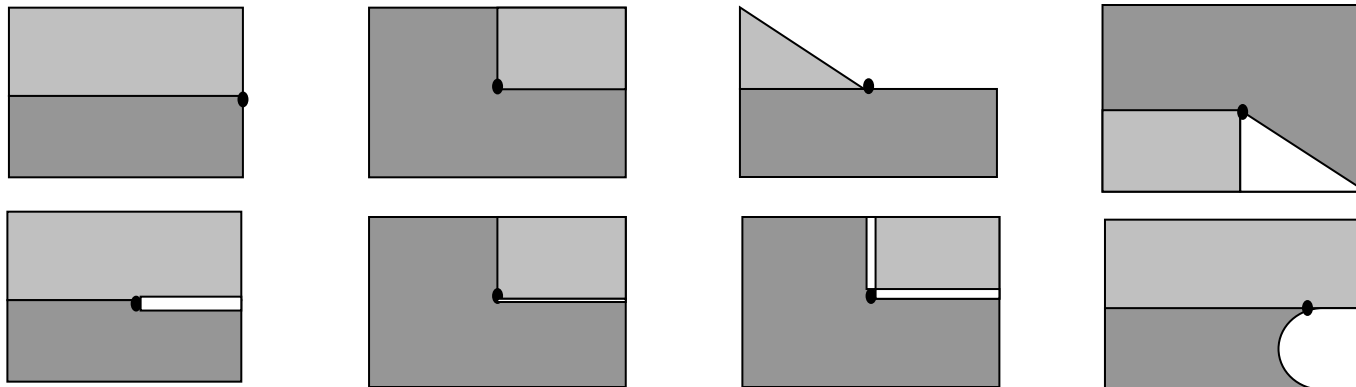
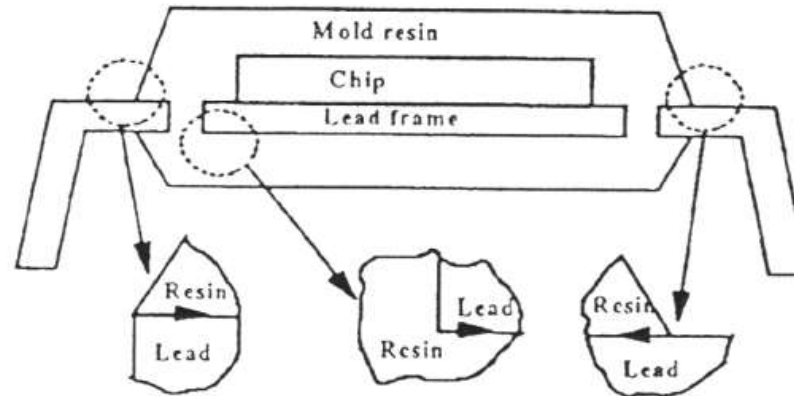
WLP - Plastic Work Density Map

1/8th model



Fan XJ, Varia B, Han Q. Design and optimization of thermo-mechanical reliability in wafer level packaging, *Microelectronics Reliability*, 50, 536–546, 2010.

Stress Singularities in Material Joints



Fan XJ, Wang HB, Lim TB. Investigation of the underfill delamination and cracking for flip chip modules under temperature cyclic loading. *IEEE Transactions on Components, Manufacturing and Packaging Technologies*, 24(1), 84-91. 2001.

Summary

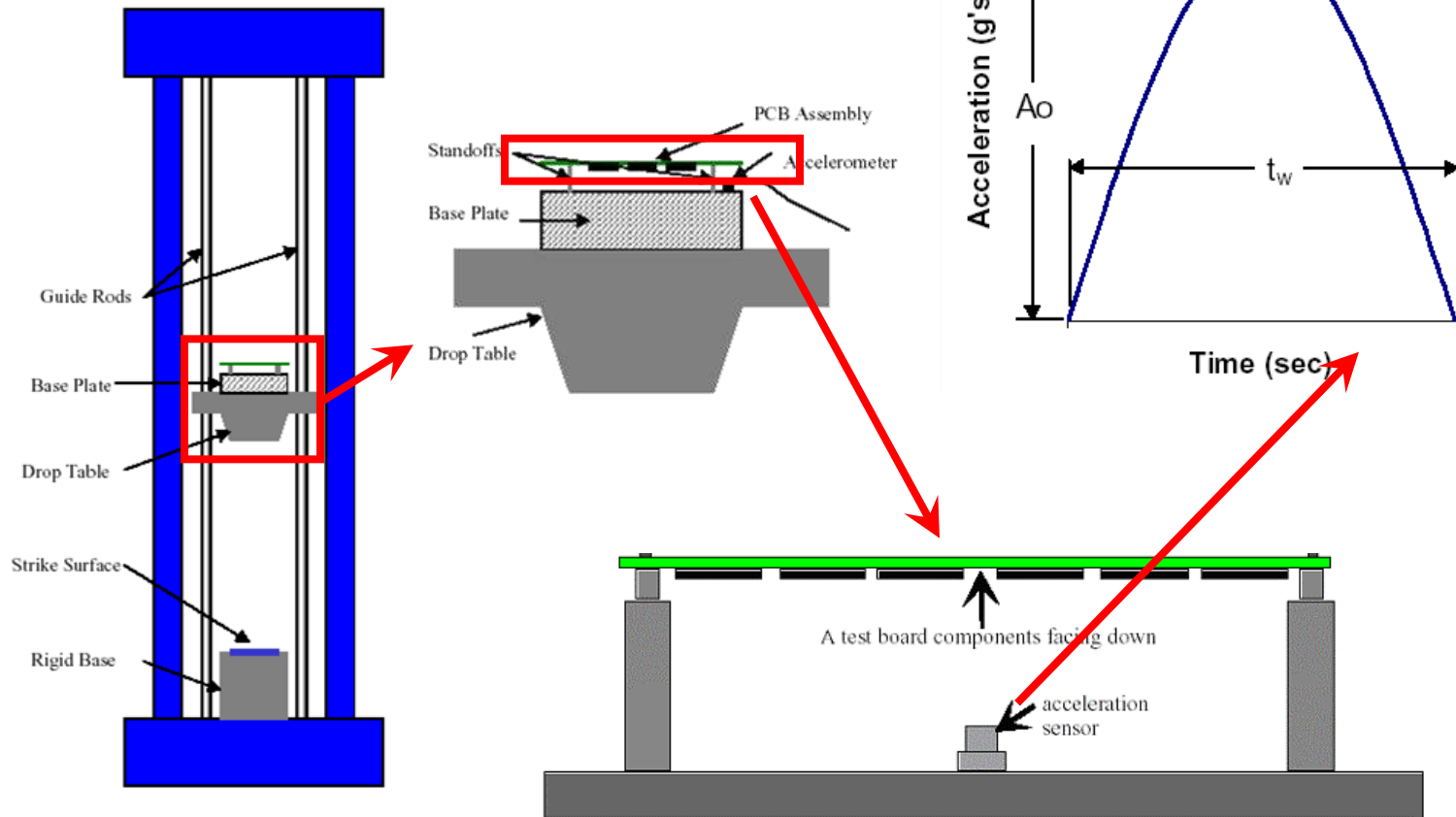
- Thermal mismatch vs. temperature gradient
- Analytical solution
 - Layered structure (stress, warpage, effective CTE)
 - Cylindrical structure (TSV)
- Die-level thermal stress – thermal stress in TSV
- Package-level thermal stress problem – warpage
- Chip-package interaction (CPI) – submodeling technique
- Board level thermal stress problem
 - Solder ball thermal cycling (Flip chip BGA, WLP)
 - Creep equations
 - Best method for practice
 - Initial stress free condition; full model vs. global/local model; worst solder ball location, volume averaging
- Stress singularity of joint materials

- Introduction
- Temperature Loading
- **Mechanical Loading**
- Moisture Loading
- Electrical Current Loading - Multi-Physics Modeling
- Summary

Outline

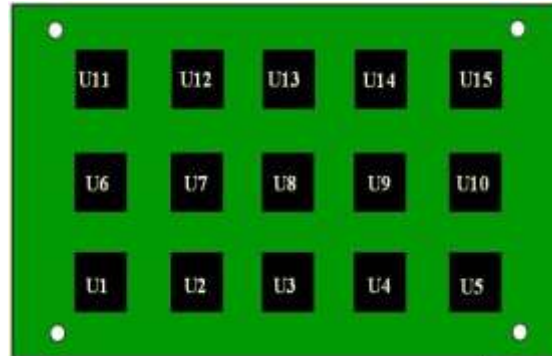
- **JEDEC drop test standard (old and new)**
- **Simulation method**
 - Input G
 - Large mass
 - Input D
 - Direct acceleration
 - Global/local method
- **4-point bending test**

JEDEC Drop Test Set-Up



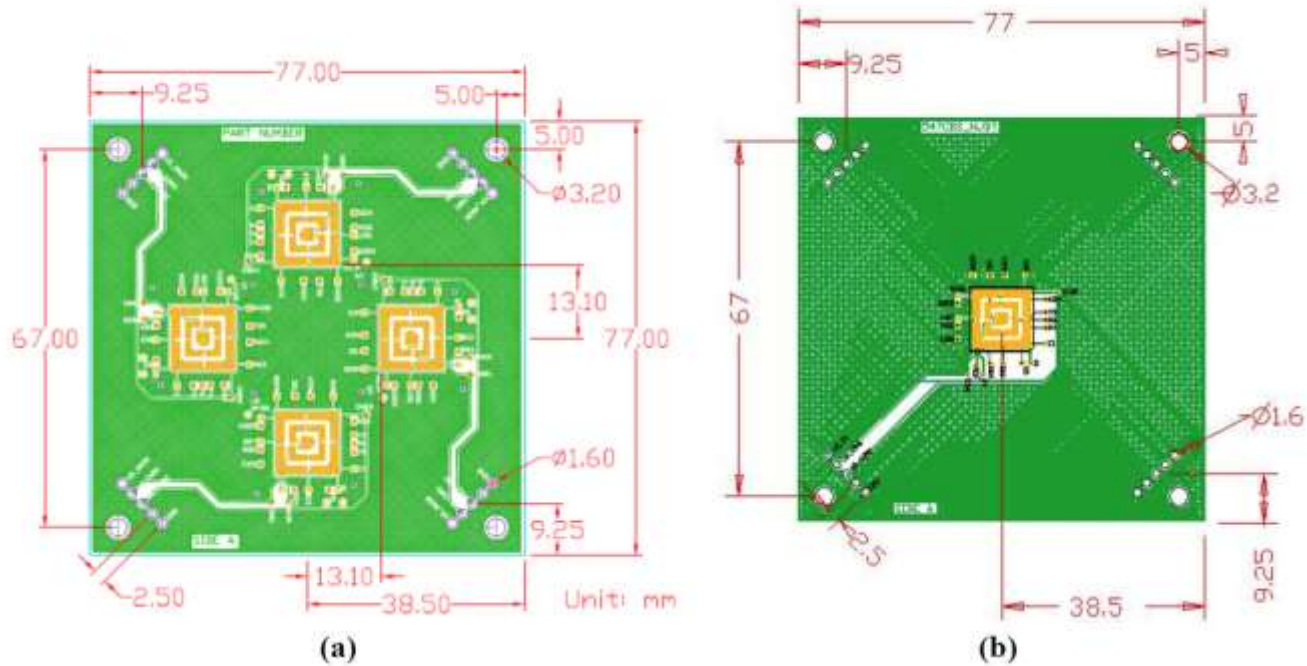
- Peak acceleration: 1500g
- Impulse time: 0.5ms with half-sine shape

JEDEC Test Standard JESD22-B111 (Old)

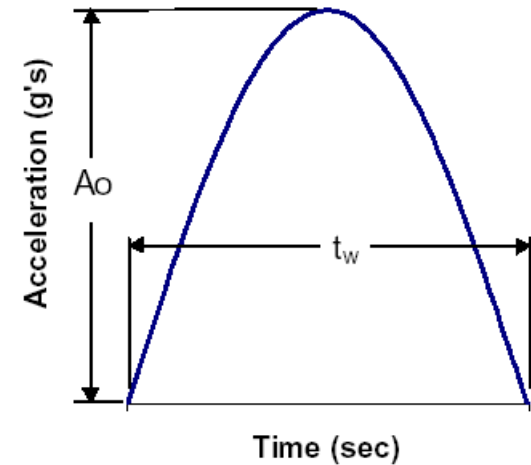
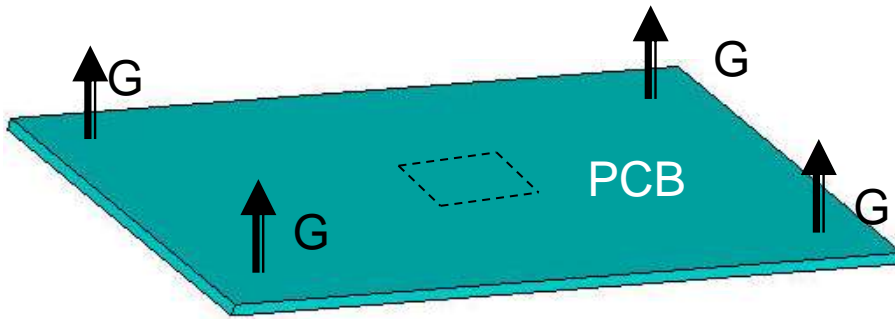


Group	Number Components	Component Location
A	4	U1, U5, U11, & U15
B	4	U2, U4, U12, & U14
C	2	U6 & U10
D	2	U7 & U9
E	2	U3 & U13
F	1	U8

JEDEC Test Standard JESD22-B111A (New, 11/2016)



FEA Modeling: Input-G Method



$$\{M\}[\ddot{u}_1] = \{C\}[\dot{u}_1] + \{K\}[u_1] = 0$$

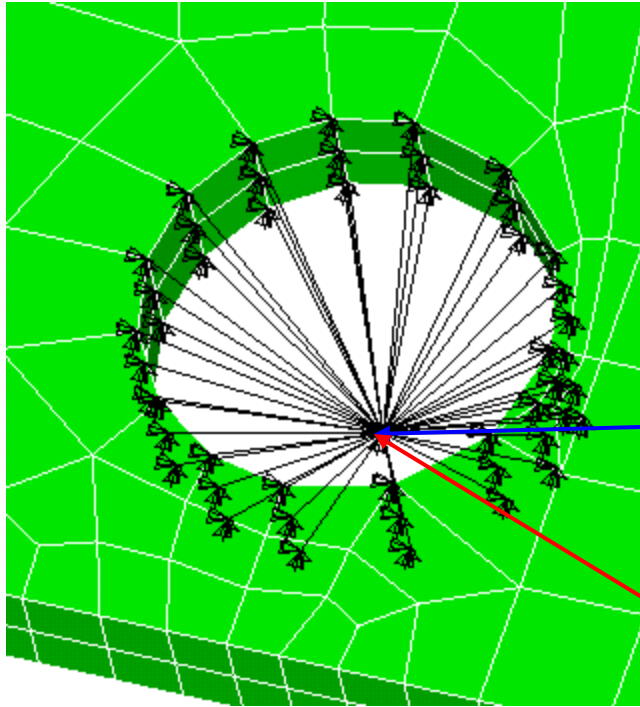
with initial conditions

$$[u_1]|_{t=0} = 0, \quad [\dot{u}_1]|_{t=0} = \sqrt{2gH}$$

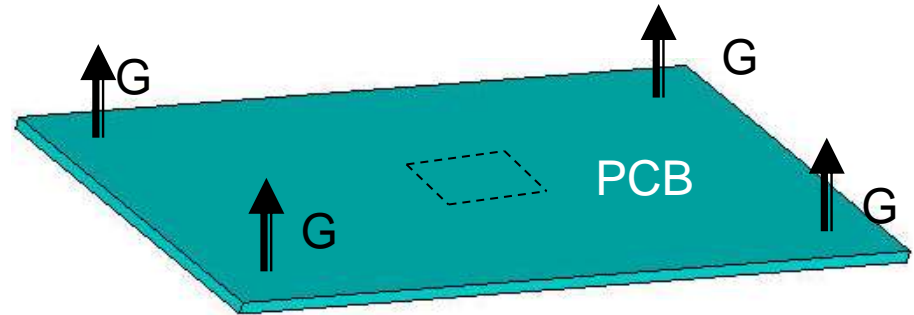
and boundary condition

$$a = \left\{ \begin{array}{ll} 1500g \sin \frac{\pi t}{t_w} & \text{for } t \leq t_w, \quad t_w = 0.5 \\ 0 & \text{for } t \geq t_w \end{array} \right\}$$

FEA Modeling: Input-G Method



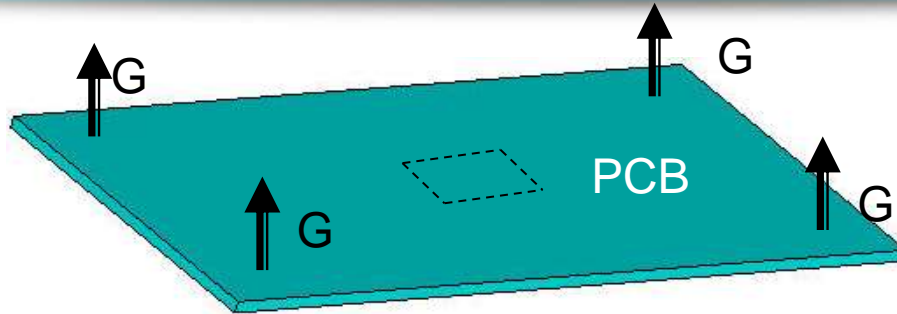
Large Mass Method



Large mass 10^8 times mass of structure coupled (1mm away from PCB bottom)

Acceleration = $1500 \times g \times \sin(\pi \times t/0.5)$
Force = $10^8 \times$ Acceleration

FEA Modeling: Input-D Method



$$\{M\}\{\ddot{u}_1\} = \{C\}\{\dot{u}_1\} + \{K\}[u_1] = 0$$

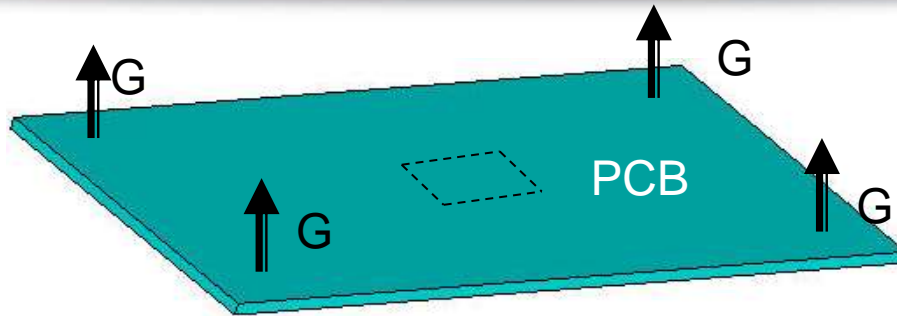
with initial conditions

$$[u_1]|_{t=0} = 0, \quad [\dot{u}_1]|_{t=0} = \sqrt{2gH}$$

and boundary condition

$$[u_1]|_{\text{at hole}} = \left\{ \begin{array}{l} -\left(\frac{t_w}{\pi}\right)^2 (1500g) \sin \frac{\pi t}{t_w} + \left(\frac{t_w}{\pi} (1500g) + \sqrt{2gH}\right) t, t \leq t_w \\ \left(2 \frac{t_w}{\pi} (1500g) + \sqrt{2gH}\right) t - \left(\frac{t_w}{\pi}\right)^2 (1500g) \quad , t \geq t_w \end{array} \right\}$$

FEA Modeling: Direct Acceleration Method



$$\{M\}[\ddot{u}_2] + \{C\}[\dot{u}_2] + \{K\}[u_2] = \left\{ \begin{array}{ll} -\{M\}1500g \sin \frac{\pi t}{t_w} & t \leq t_w, t_w = 0.5 \\ 0 & t \geq t_w \end{array} \right\}$$

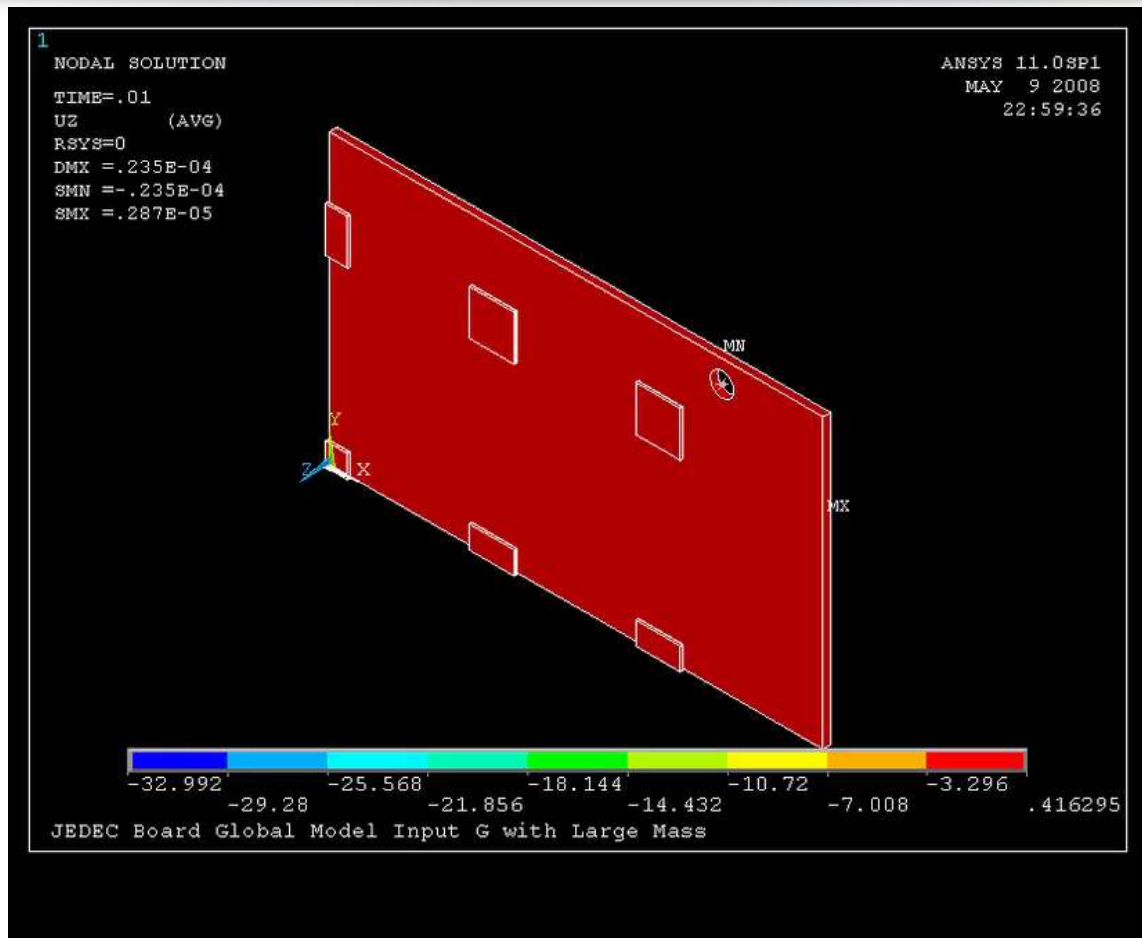
with initial conditions

$$[u_1]|_{t=0} = 0, \quad [\dot{u}_1]|_{t=0} = \sqrt{2gH}$$

and boundary conditions

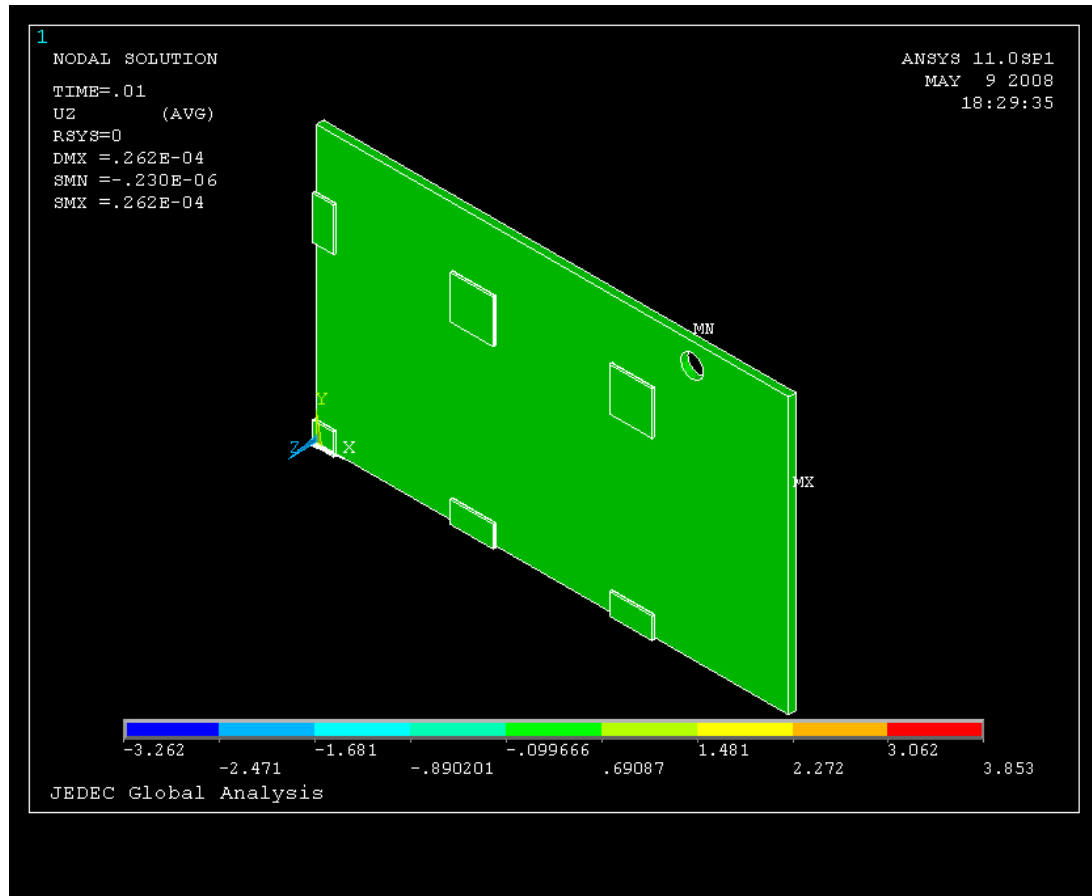
$$[u_2]|_{\text{at hole}} = 0$$

Input-G Method – Board Vibration



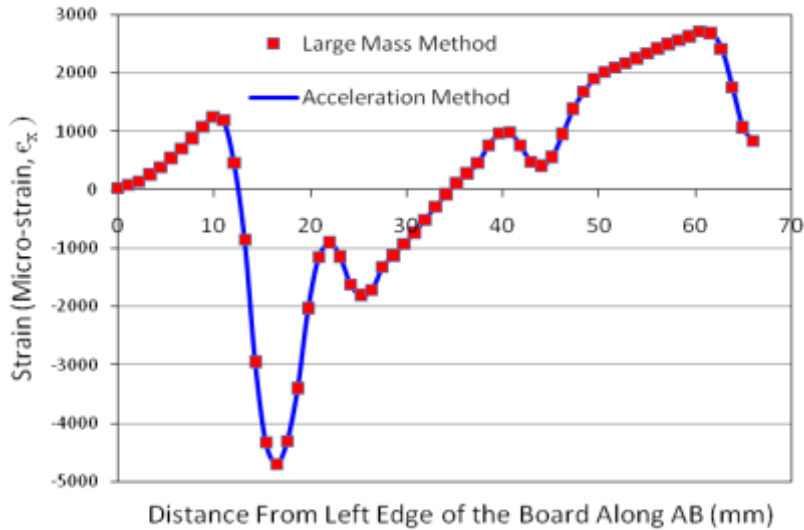
- Since acceleration impulse is given on the screw locations, the displacements are not fixed and the board is moving in one direction.

Direct Acceleration Method – Board Vibration

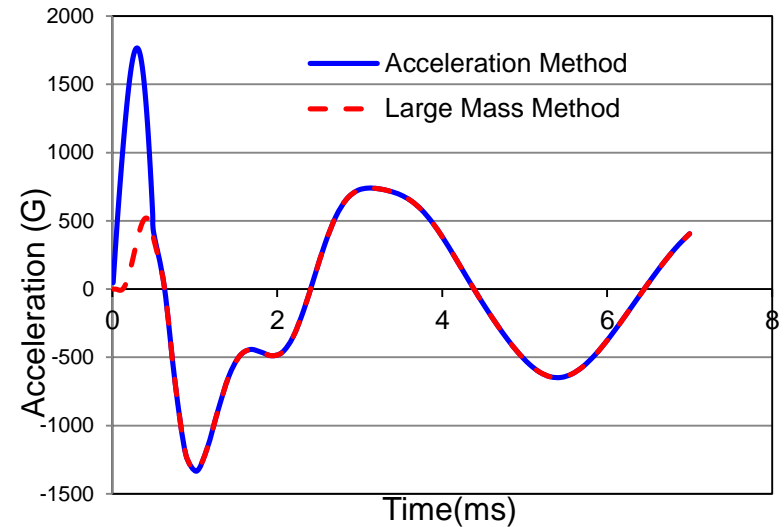


- Since the displacement is fixed at the screw locations, the board is vibrating as expected.

Input-G vs. Direct Acceleration Method



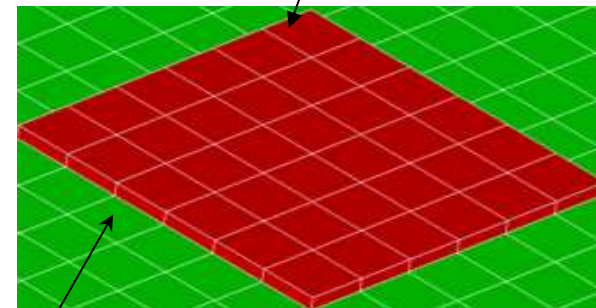
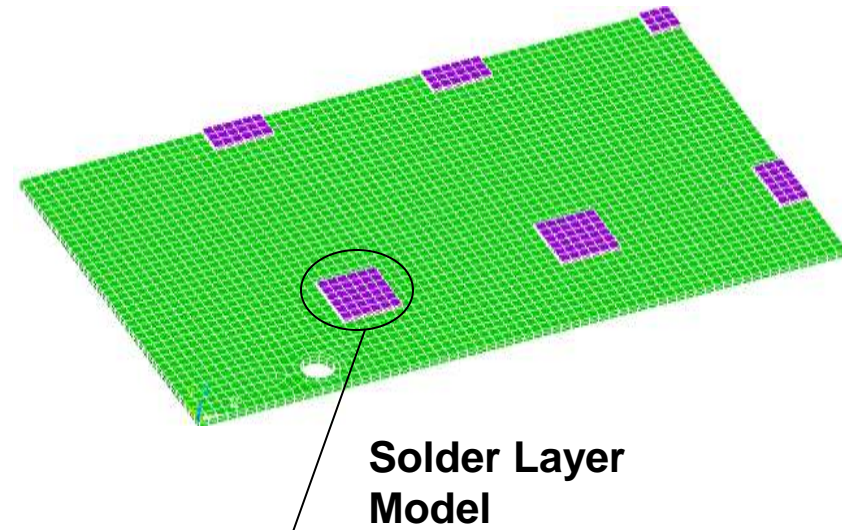
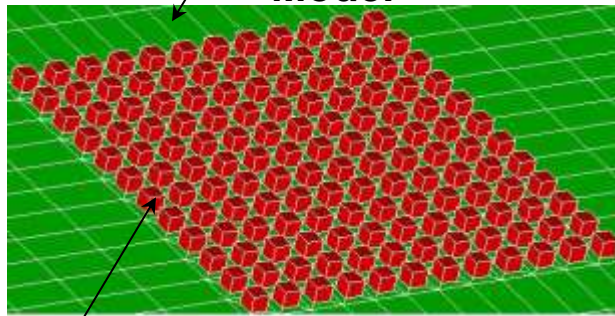
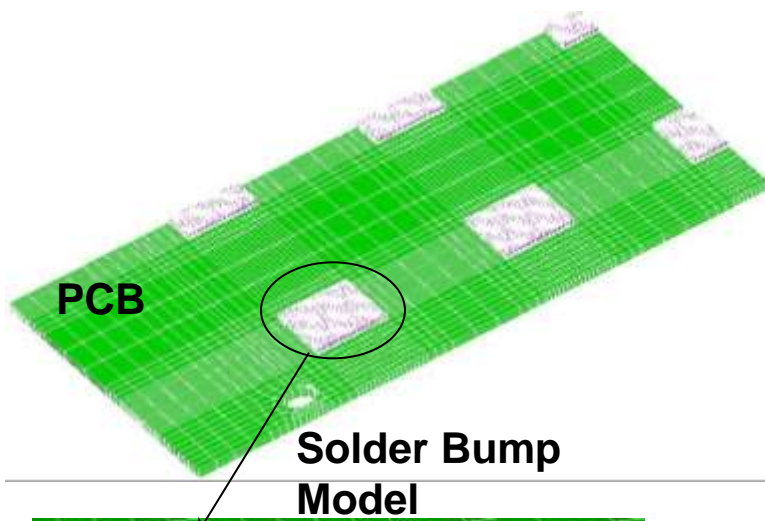
Distribution of Board Strain in x direction along line AB at $t = 1.5$ ms



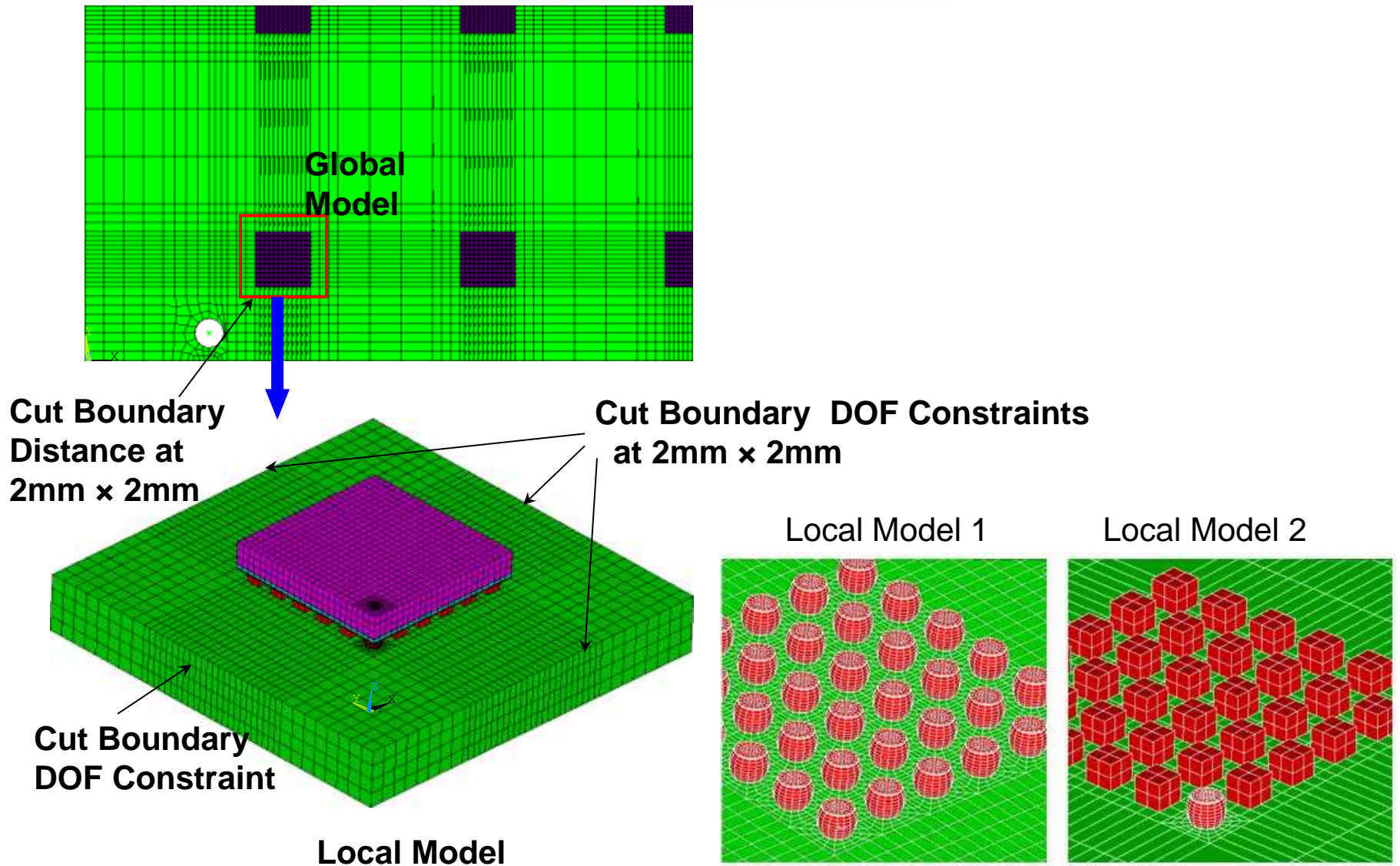
Comparison of acceleration time history

- **Direct Acceleration method is recommended, but all methods provide identical results.**

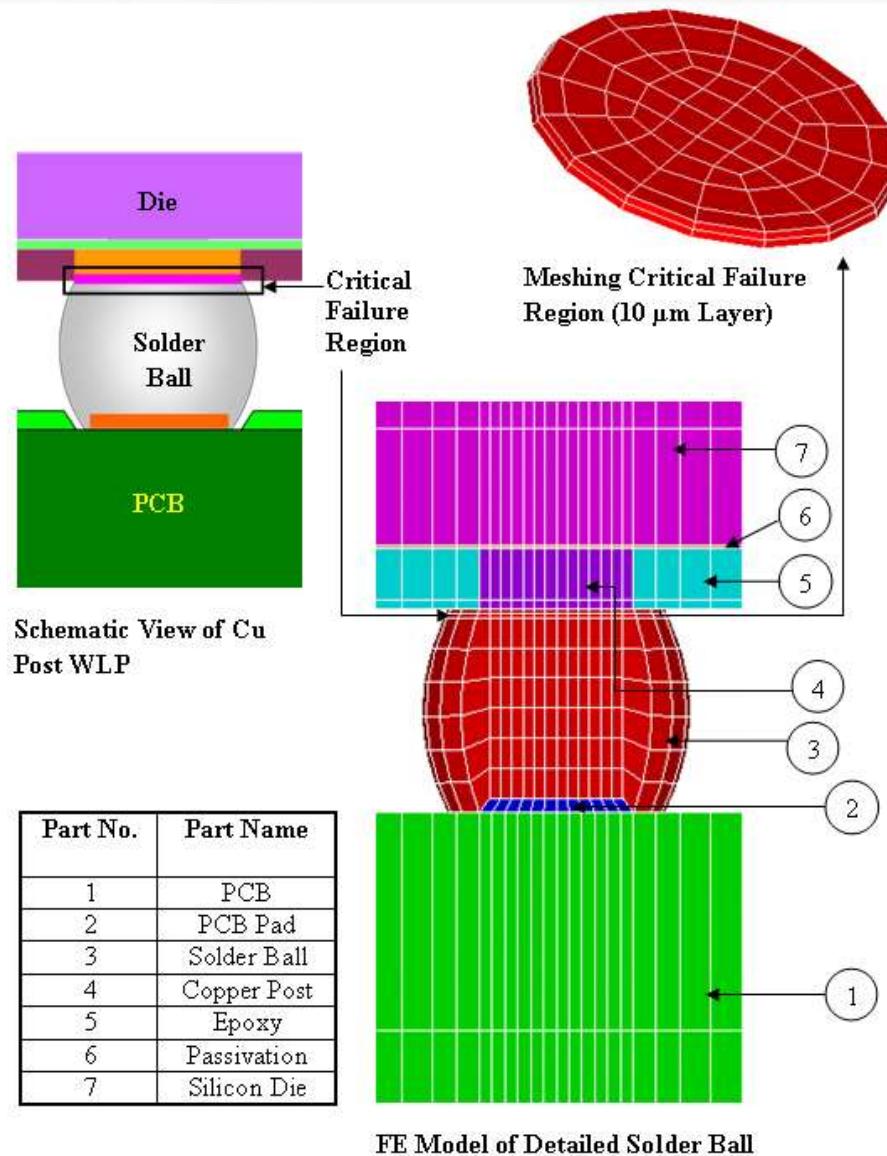
FEA Global Model



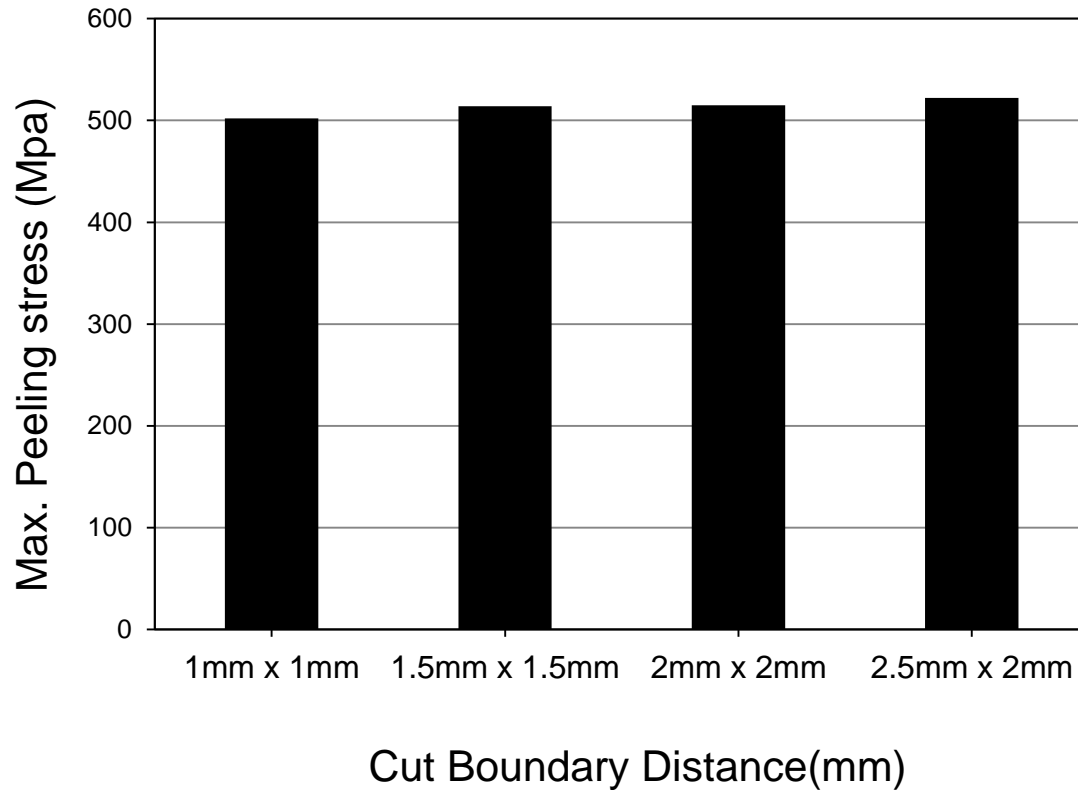
FEA Local Model



FEA Local Model – Solder Joint

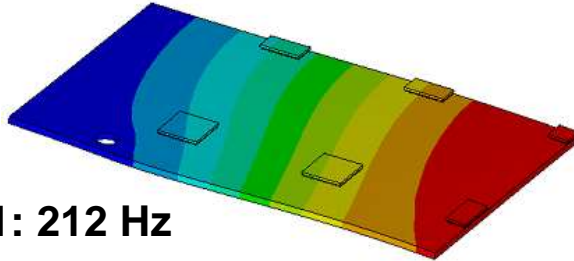


Effect of Cut Boundaries in Sub-modeling

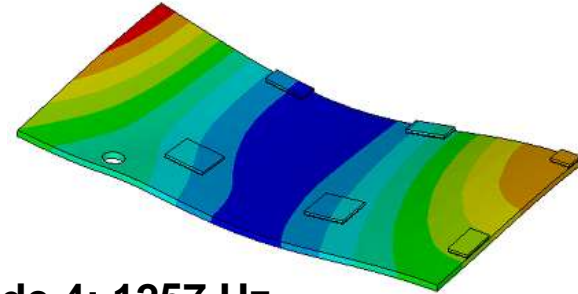


Global Model Results – Frequency and Mode Shape

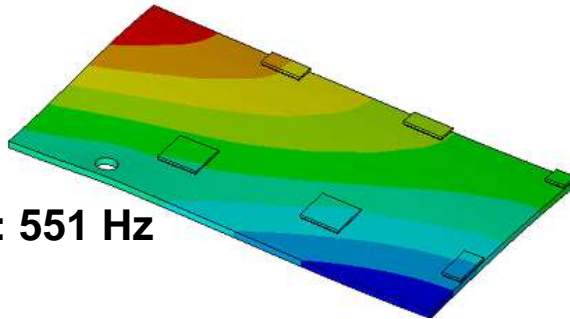
Mode 1: 212 Hz



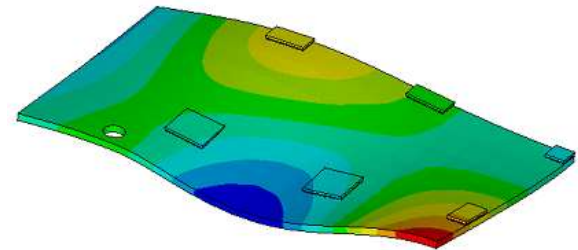
Mode 4: 1257 Hz



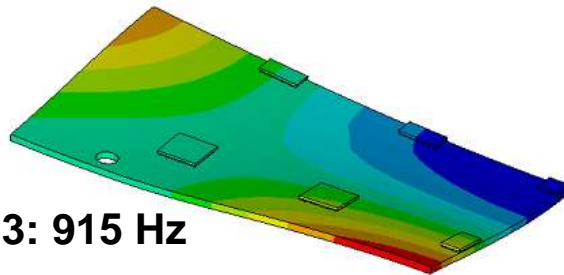
Mode 2: 551 Hz



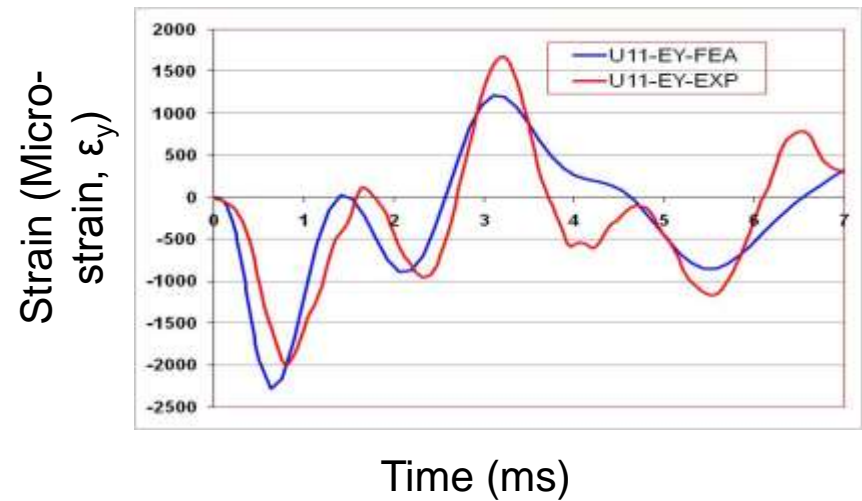
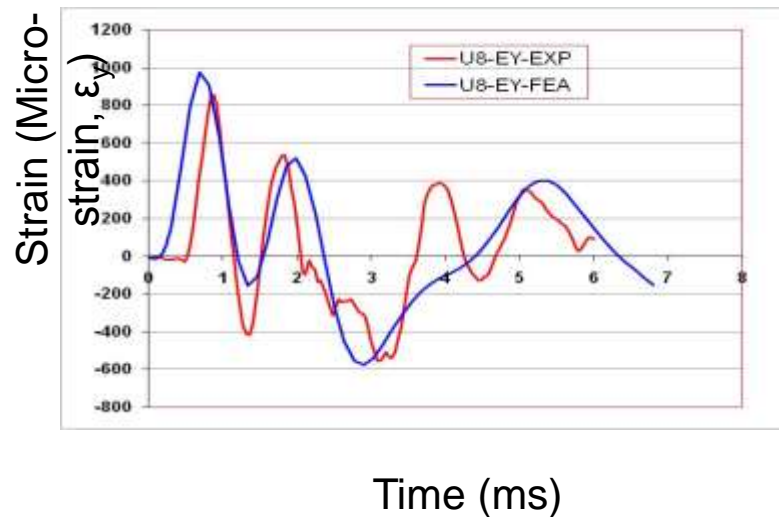
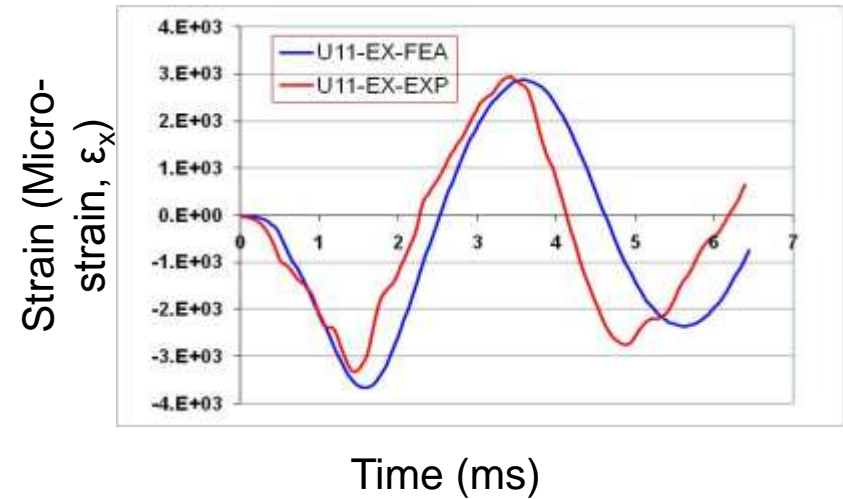
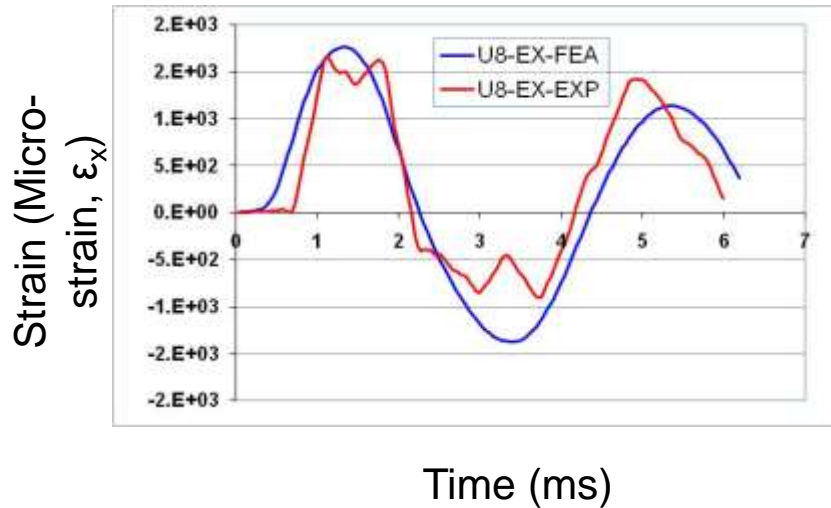
Mode 5: 2193 Hz



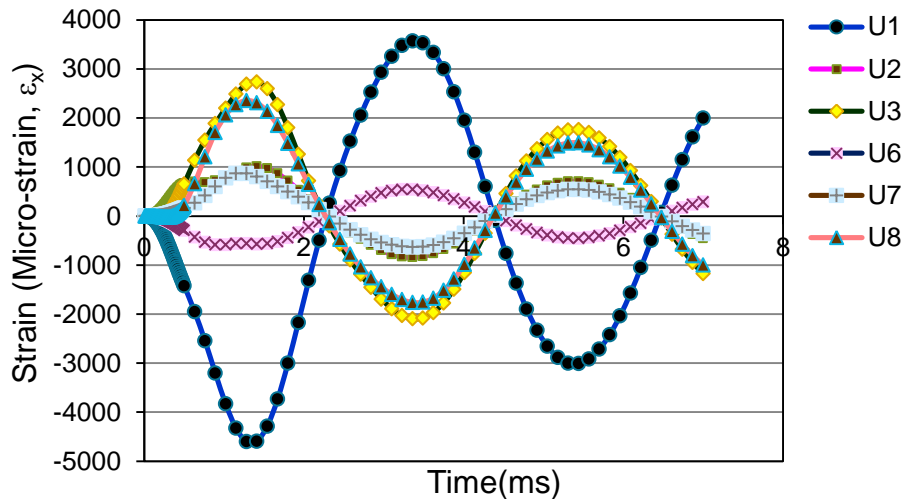
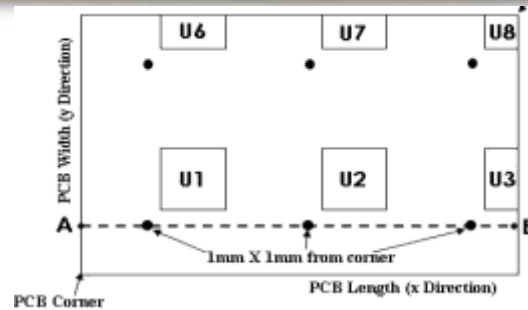
Mode 3: 915 Hz



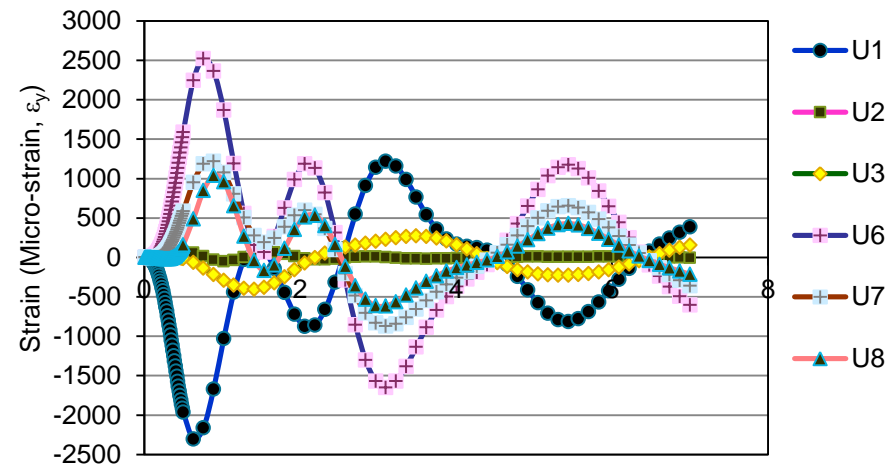
Experimental Validation



Board Strain Analysis: Package Corner Strain(ϵ_x)

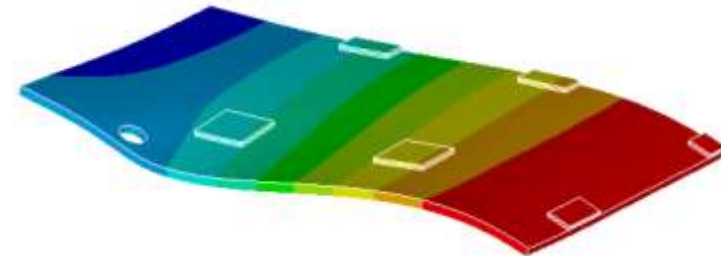


Strain History in x
Direction

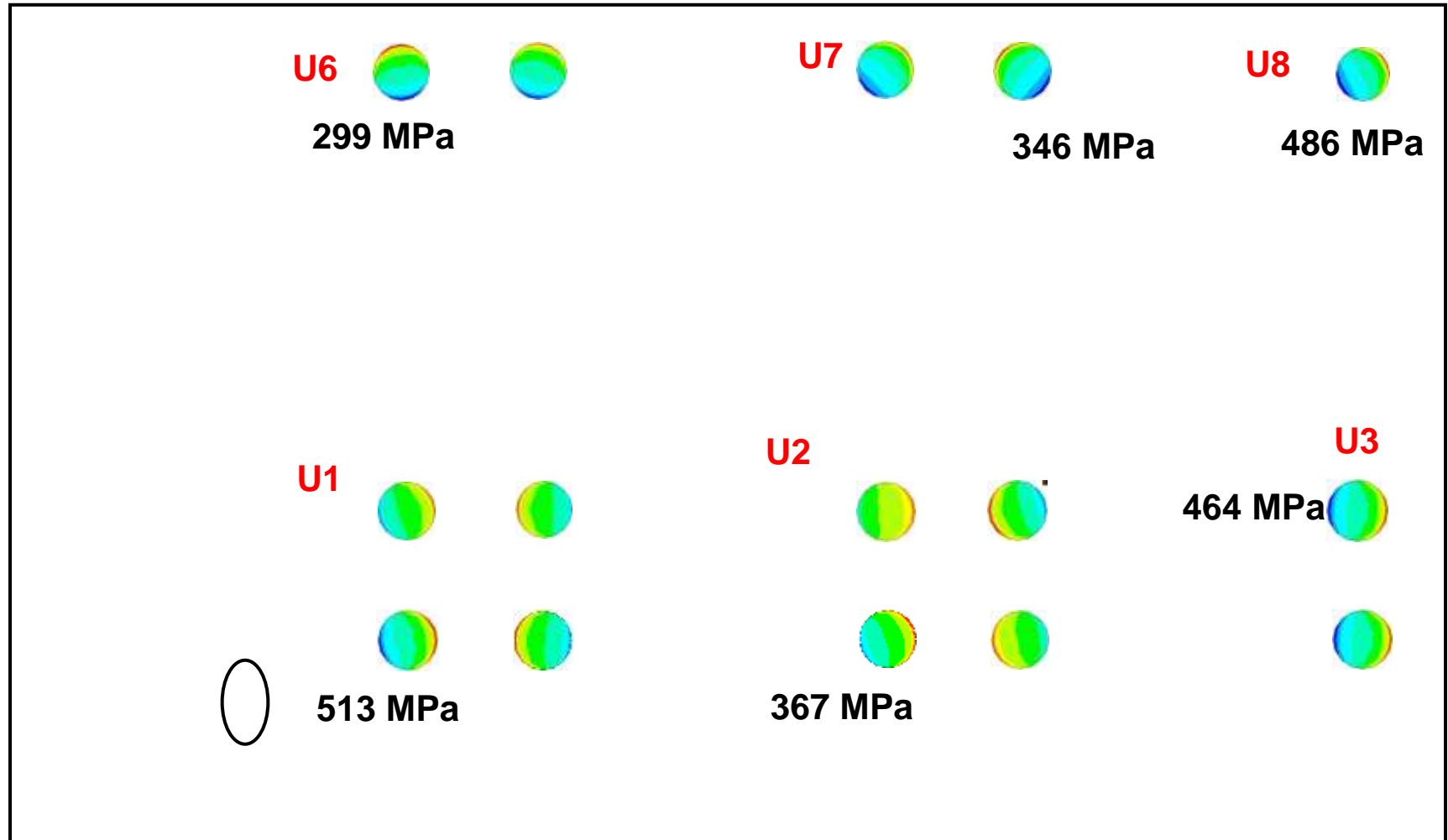


Strain History in y
Direction

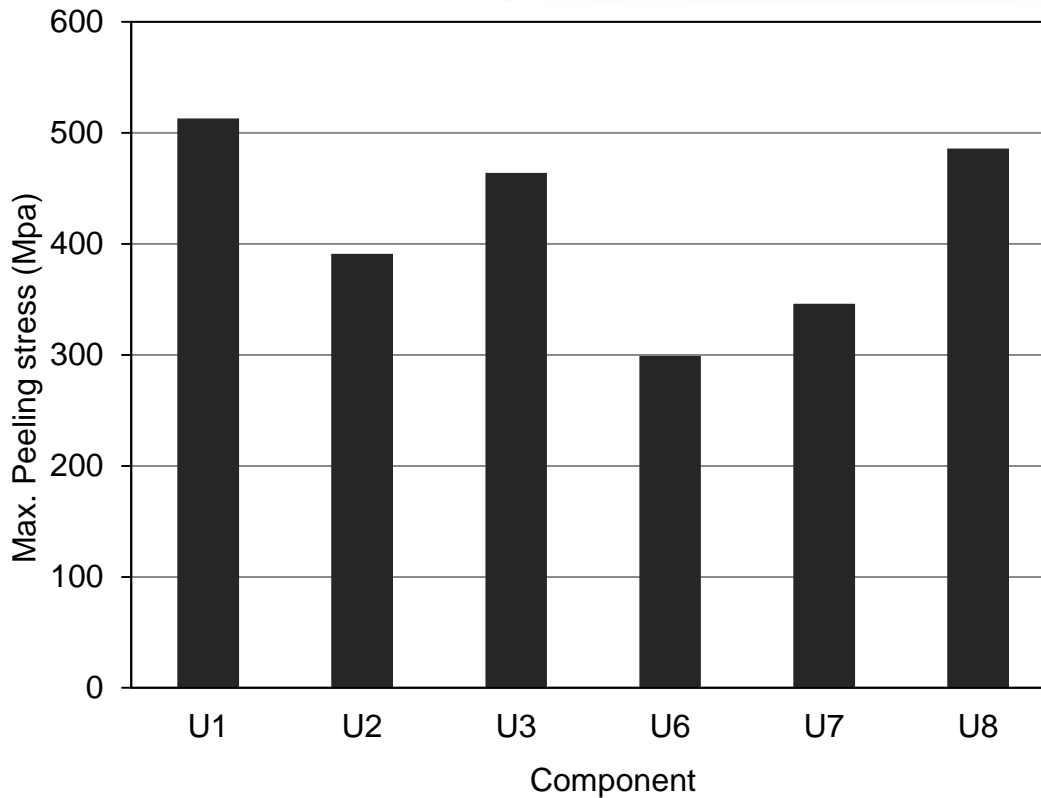
- U1>U3>U8>U2>U7>U6.
- U8 bends in opposite direction to U1.



Ball Stress Contour Plot on 1/4th JEDEC Board

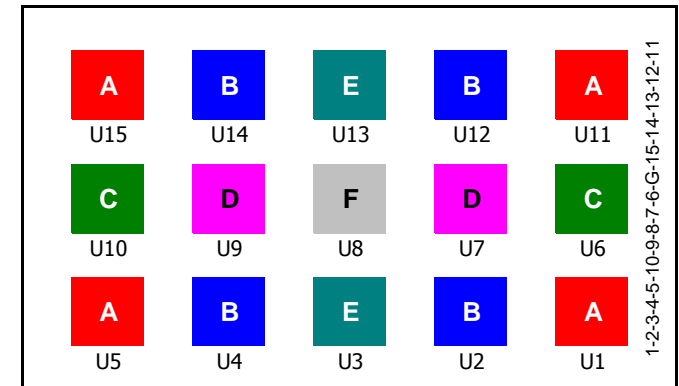


Maximum Peel Stress



Overall, the stress developed in components can be ranked as:

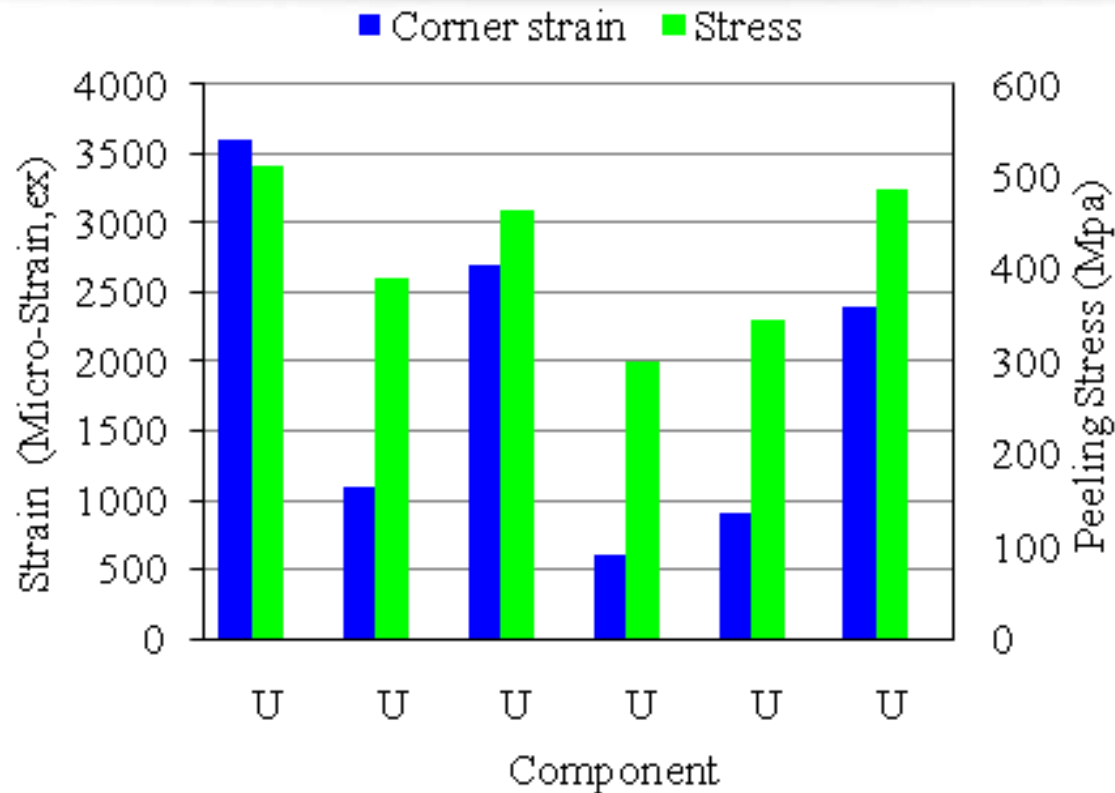
$U1 > U8 > U3 > U2 > U7 > U6$



Failure rank:

- **Group A (U1, U5, U11 and U15) > Group F (U8) > Group E (U3, U13).**

Correlation of Solder Joint Stress and Board Strain

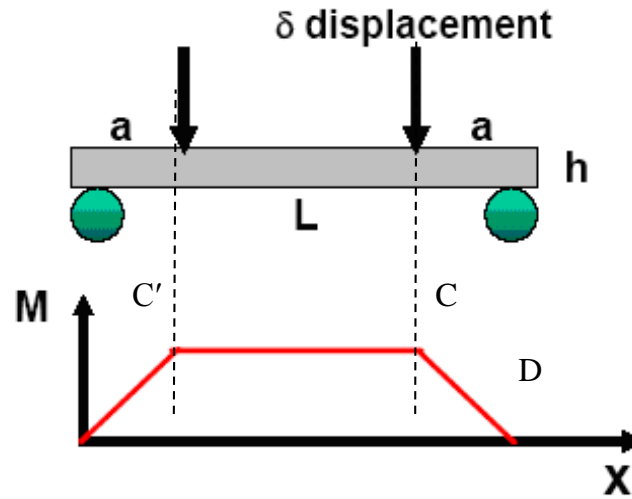


Board Strain Ranking: U1>U3>U8>U2>U7>U6

Solder Joint Stress Ranking: U1>U8>U3>U2>U7>U6

- **Exact correlation does not exist for strain and stress values.**

4-Point Bending Test



Segment CC'

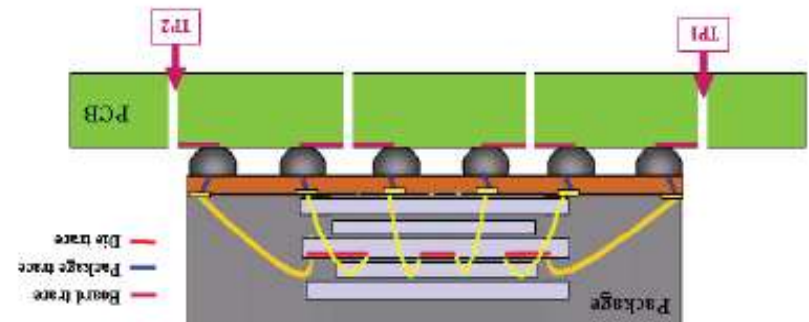
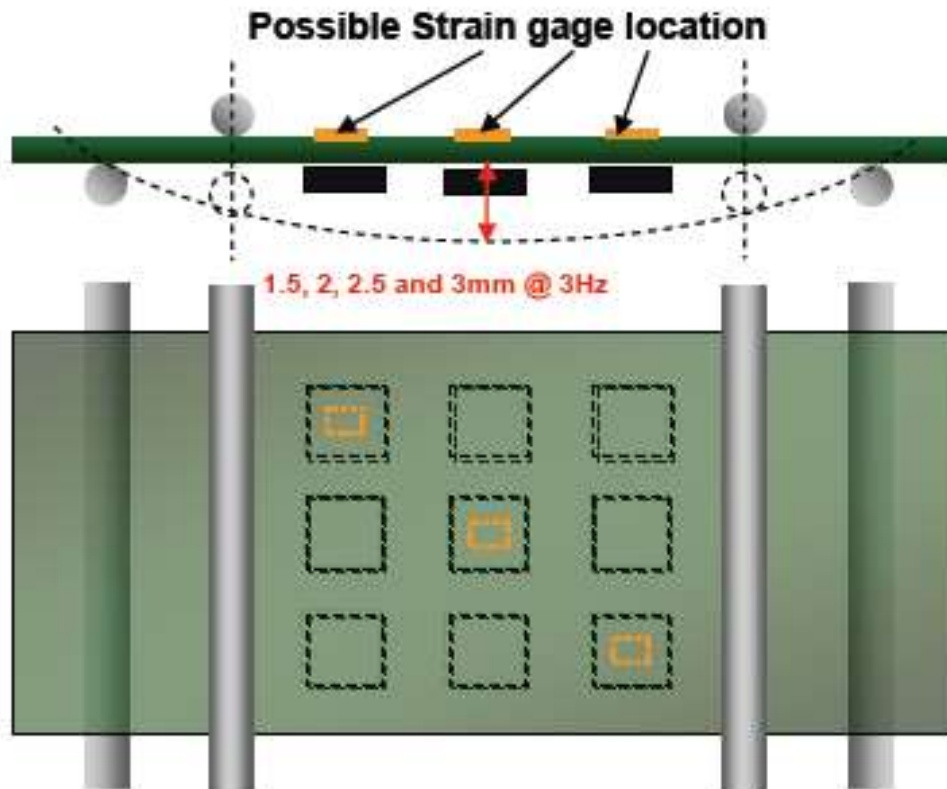
$$\varepsilon_x = \frac{3h\delta}{a(3L - 4a)}$$

Segment CD

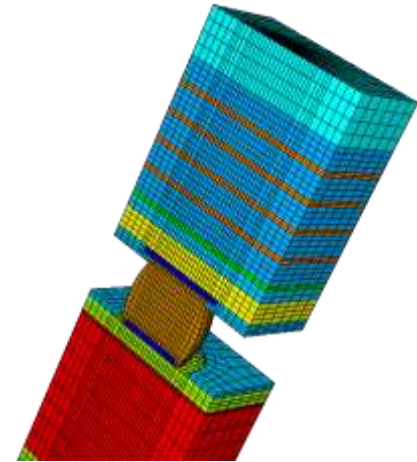
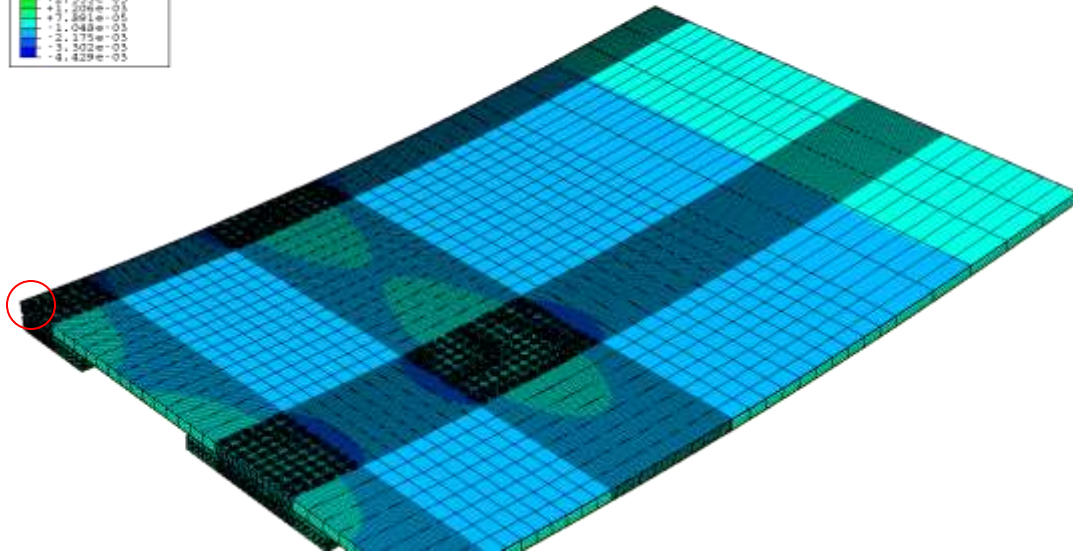
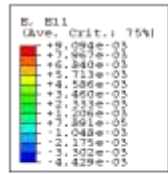
$$\varepsilon_x = \frac{12h\delta}{L^3} x$$

4-Point Bending Test

- 4PT bending test to take advantage of uniform strain BC for each component.
- JEDEC board + 9 components with electric monitoring for each one of them.
- Cyclic testing of freq 3Hz @ 1.5mm, 2mm and 2.5mm deflections to DC failure.

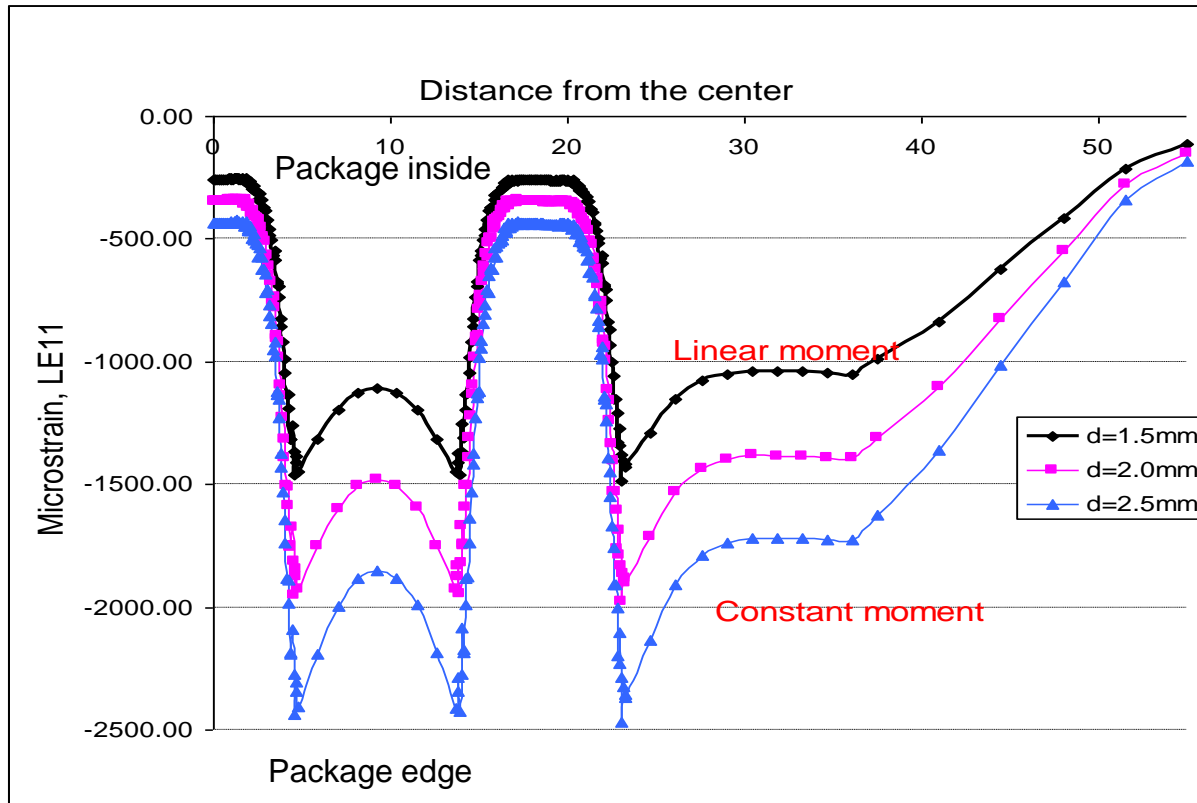


Cyclic Bending: Modeling Methodology

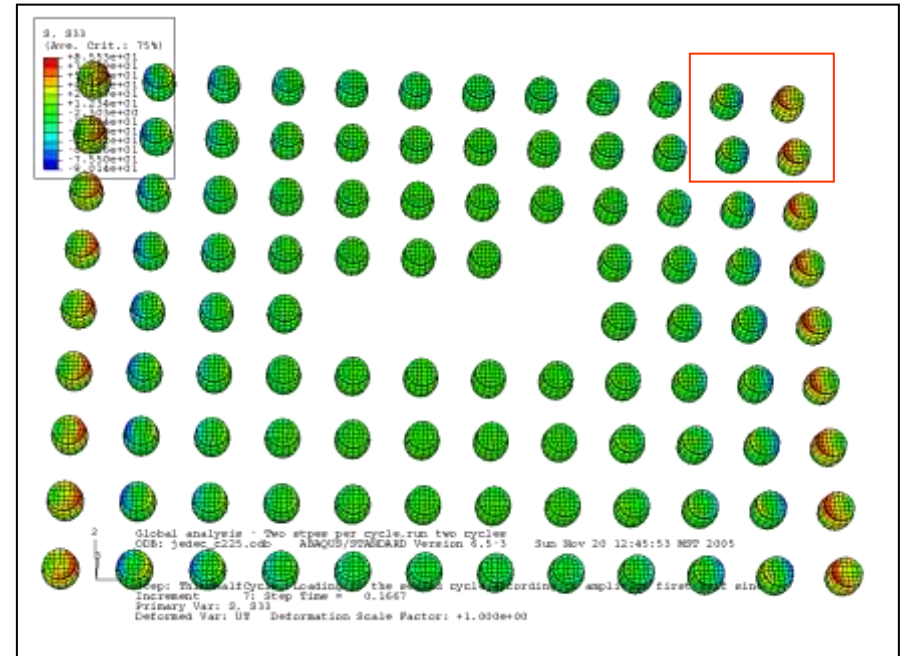
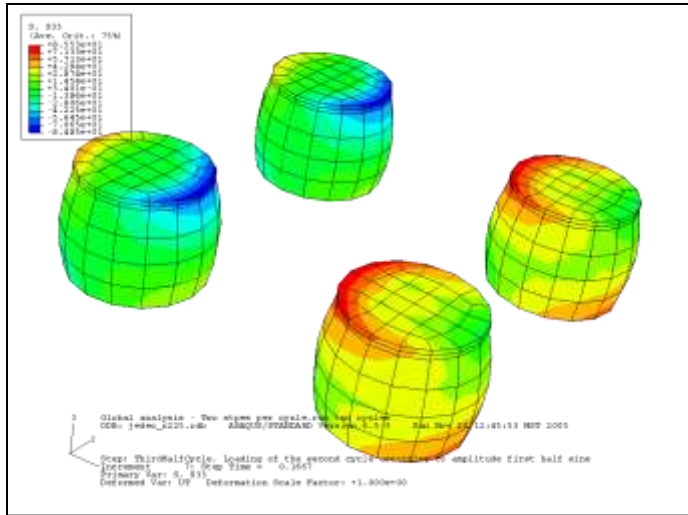


- Global/local modeling methodology
 - Elastic global model.
 - Local models use EPP SAC405 solder material properties.

Strain Distribution



Board Side or Package Side?

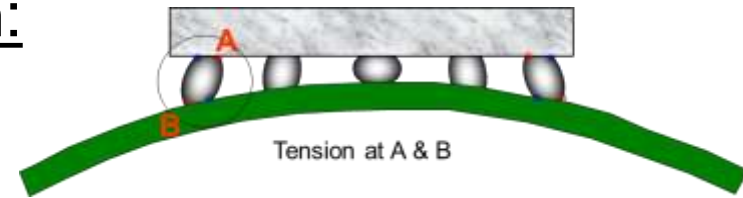


- Consistency with failure propagation in terms of peeling stress
 - Package side: inside-out
 - Board side: outside-in
- Failure happens more likely at the package side: higher failure parameter value.

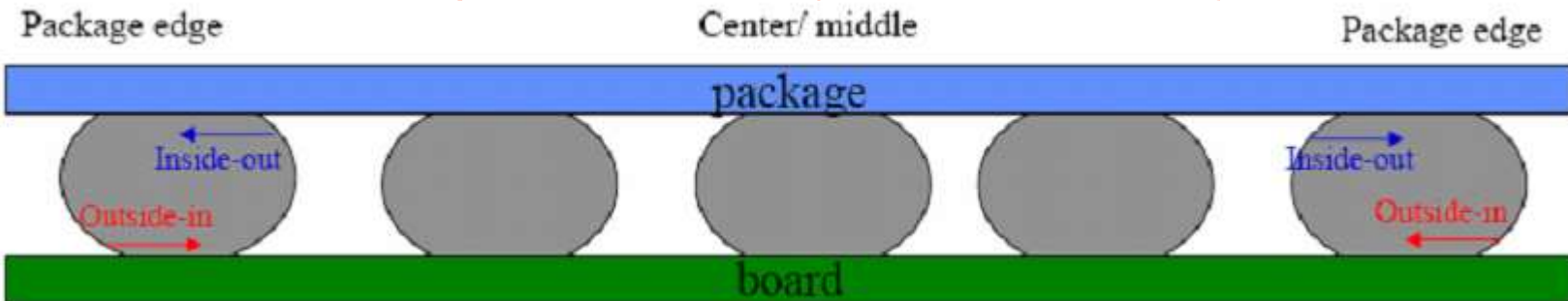
Failure Pattern

Solder joint crack propagation direction:

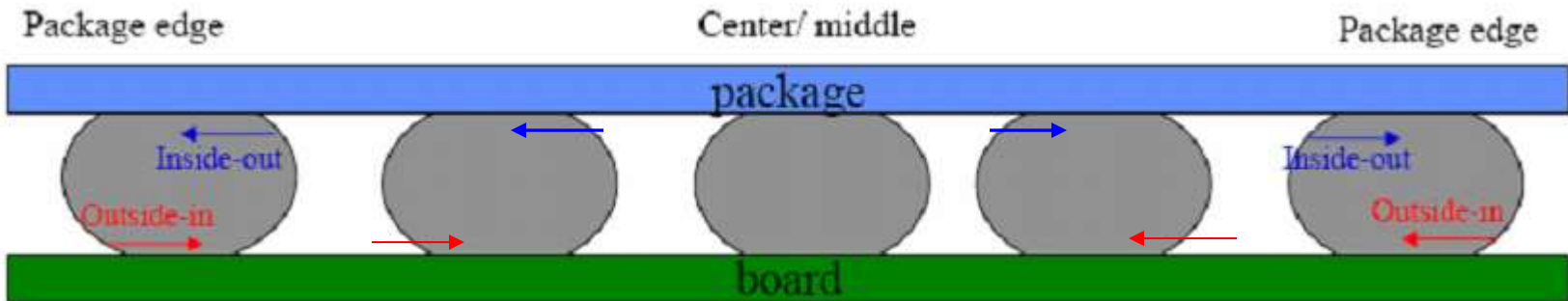
- Package side: inside-out
- Board side: outside-in



Sequential Failure (1.5mm deflection)



Simultaneous Failure (2.5mm deflection)



Summary

- **JEDEC drop test standard**
 - JESD22-B111, old one, with 15 components
 - JESD22-B111A, new one, with 4 components or 1 component
- **Finite element modeling**
 - Input G method, large mass method, input displacement method, direct acceleration method
 - Global/local modeling
 - Peel stress used as indicator for failure
- **Four-point bending test and modeling**
 - Global/local modeling
 - Global with linear elastic but nonlinear geometry analysis
 - Local model with elastic-plastic modeling

- Introduction
- Temperature Loading
- Mechanical Loading
- **Moisture Loading**
- Electrical Current Loading - Multi-Physics Modeling
- Summary

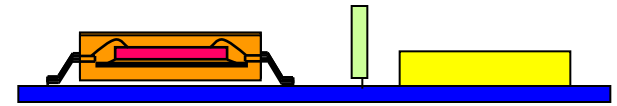
Outline

- **Introduction**
- **Moisture related reliability testing**
- **Moisture diffusion modeling**
- **Vapor pressure theory**
- **Hygroscopic swelling**

Moisture Absorption of Electronic Packages



Individual component



Surface mount to board



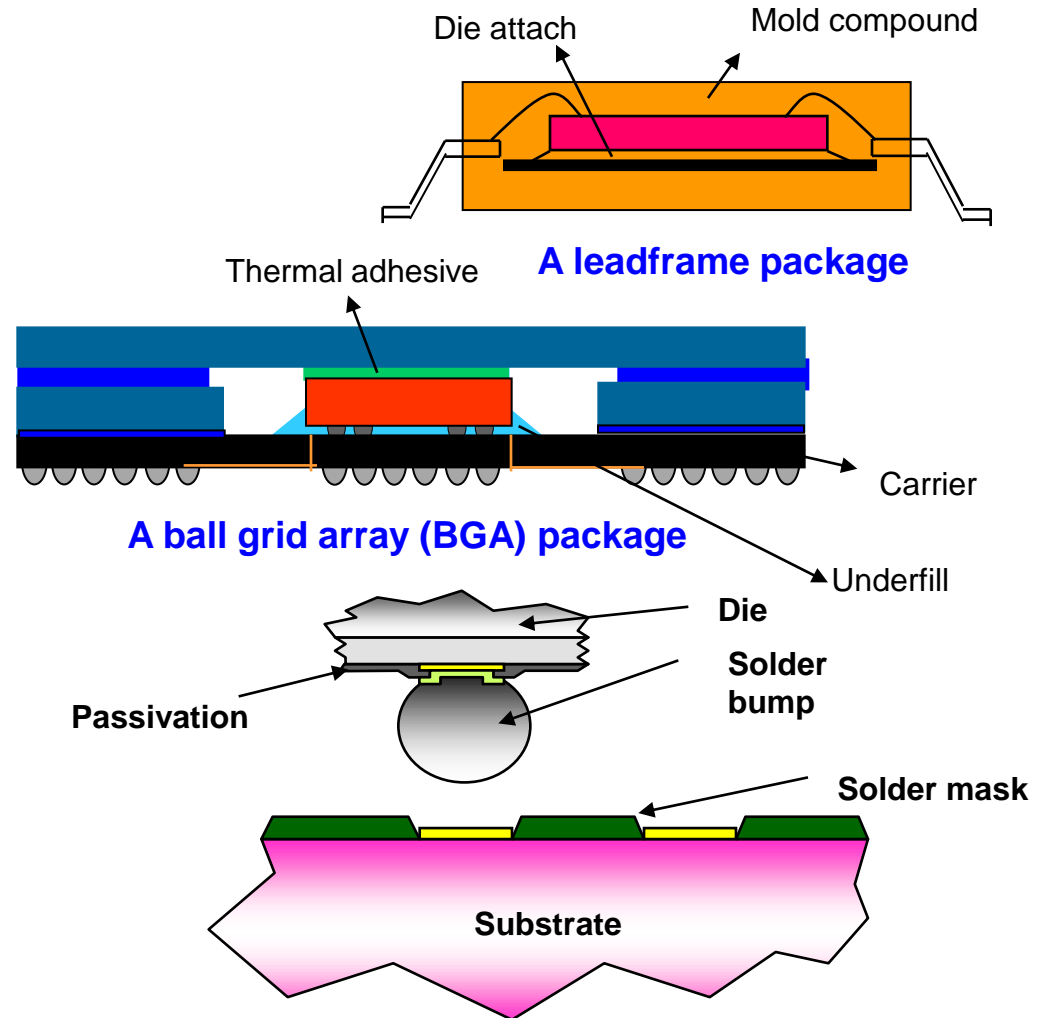
storage

shipment

- Electronic packages absorb moisture in uncontrolled humid conditions prior to the surface mount on board.

Polymer Materials in Electronic Packaging

- Bulk-form
 - encapsulation (e.g. mold compound)
 - substrate ...
- Adhesives
 - die-attach, underfill
 - thermal adhesives ...
- Thick- or thin- film
 - solder mask
 - passivation

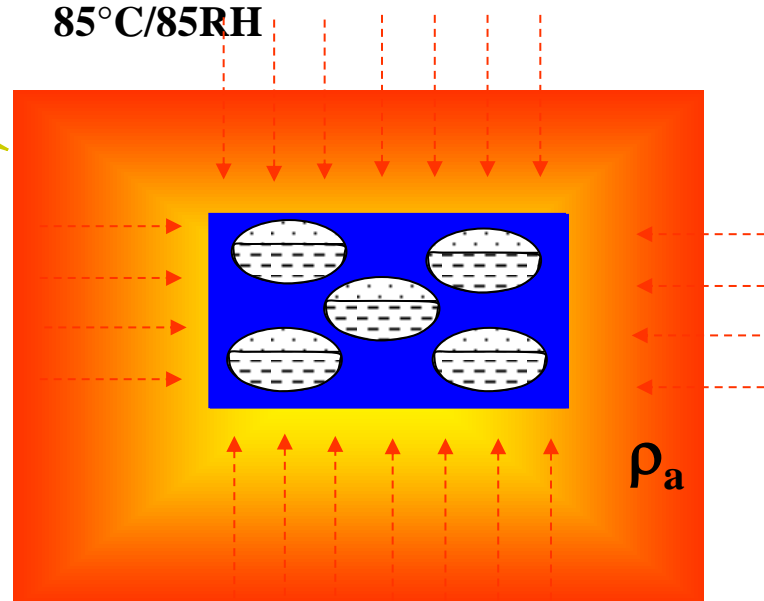


- **Polymeric materials are susceptible to moisture absorption.**

Moisture Absorption in Polymeric Materials

ρ_w : moisture density in polymeric material
 ρ_a : ambient moisture density under 85°C/85%RH

$$\rho_w = 81.2 \rho_a$$

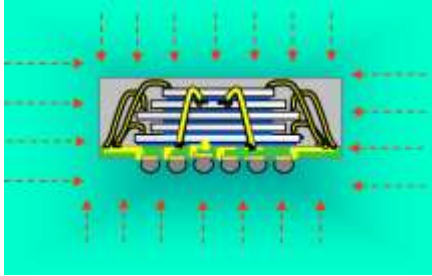


Moisture condensation in a typical underfill

- Under 85°C/85%RH condition, $\rho_w = C_{sat} = 2.47e-2 \text{ g/cm}^3 = 81.2 \rho_a$ (C_{sat} : saturated moisture concentration)
 - Moisture is condensed into liquid state.
 - Moisture exists in micro-pores or free volumes (in bulk or at interface).
 - Moisture vaporizes at reflow, possibly still at mixed liquid/vapor phases.

Moisture Related Testing

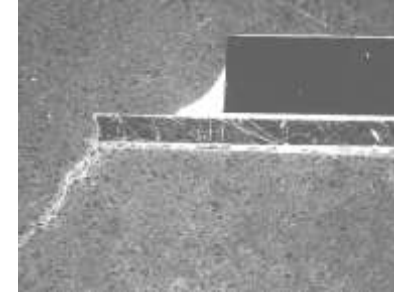
- Moisture sensitivity test



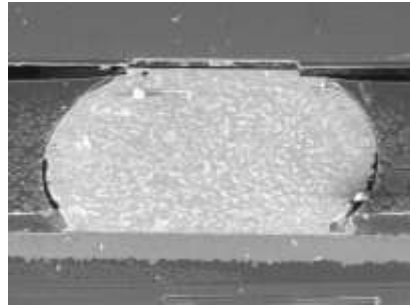
Stage 1: Moisture absorption



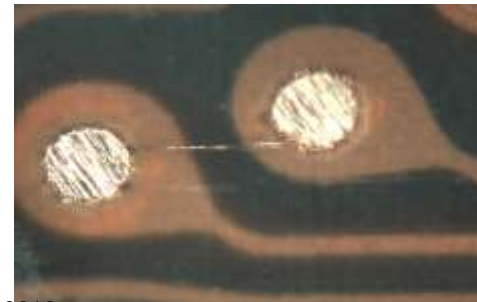
Stage 2: Soldering reflow



- Highly accelerated stress test (HAST)



- Biased HAST



Basic Concepts of Moisture Diffusion

- What is the Relative Humidity (RH)?
 - Defined as vapor pressure ratio associated with temperature T

$$RH = \frac{\text{Actual vapor pressure of the air}}{\text{Saturated vapor pressure of the air}} \times 100\%$$

- Moisture concentration, C (x, t; T, RH)
 - Mass of moisture per unit volume of substance.
- Diffusion Coefficient / Diffusivity, D(T)
 - Measures the rate of mass diffusion
 - Defined as the amount of mass flux per unit concentration gradient (m²/s)
 - A function of material and temperature
- Saturated Moisture Concentration, C_{sat}(RH, T)
 - The maximum mass of moisture per unit volume of the substance kg/m³.
- Solubility, S(T) – Henry's law
 - The ability of the substance to absorb moisture
 - Defined as the maximum mass of moisture per unit volume of the substance per unit pressure (kg/(m³Pa)).
 - A function of material and temperature

$$S = \frac{C_{\text{sat}}}{P}$$

where P = **ambient** pressure in given RH

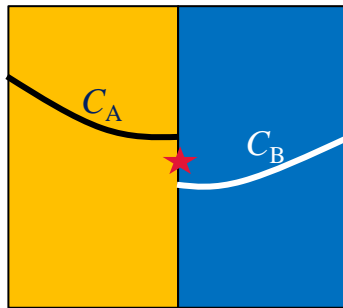
Moisture Diffusion Modeling

- Fickian diffusion theory

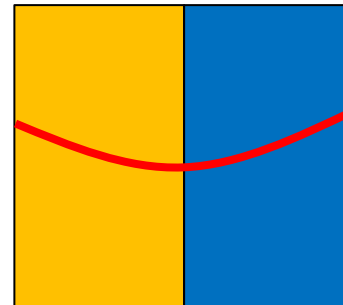
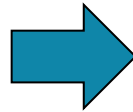
$$\frac{\partial C}{\partial t} = -\nabla \cdot (-D_0 \nabla C)$$

C : Concentration, kg/m³; D_0 : diffusivity, m²/s

- Discontinuity at interface



Concentration discontinuity



After normalization

An Overview of Moisture Diffusion Modeling

	Normalized field variable	C_{sat} : temperature-dependent	RH: time-dependent	Non-Henry's law	ANSYS	ABAQUS
Galloway et al. (1997)	C/S	✓	✓	✗	✗	✓
Wong et al. (1998)	C/C_{sat}	✗	✗	✗	✓	✗
Jang et al. (1993)	C/M	✗	✓	✗	✗	✗
Wong et al. (2016)	C/C_{sat}	✓	✓	✗	✗	✗
Markus et al. (2016)	C	✓	✓	✗	✗	✗
Chen et al. (2017)	a_w	✓	✓	✓	✓	✓
Ma et al. (2019)	C/K	✓	✓	✗	✓	✓

Ma L, Joshi R, Newman K, X.J. Fan, Improved Finite Element Modeling of Moisture Diffusion Considering Discontinuity at Material Interfaces in Electronic Packages, ECTC 2019

Water Activity Theory

- Water activity a_w is a continuous field variable derived from chemical potential.
- Moisture flux in terms of water activity gradient.

$$\mathbf{J}_m = -K_i D_{0,i} \nabla a_w$$

- Moisture concentration is given as

$$C = K a_w$$

K : generalized solubility, kg/m³

- Moisture diffusion equation

$$\frac{\partial(K_i a_w)}{\partial t} = \nabla \cdot (K_i D_{0,i} \nabla a_w)$$

- Water activity is continuous and no normalization is required for multi-material system.

Current Moisture Diffusion Theory in ANSYS

- $\bar{C} = \frac{C}{C_{sat}}$ is used as field variable.

$$\frac{\partial(C_{sat}\bar{C})}{\partial t} = \nabla \cdot (D\nabla(C_{sat}\bar{C})) + G$$

$$C_{sat} \frac{\partial \bar{C}}{\partial t} + \bar{C} \frac{\partial C_{sat}}{\partial T} \frac{\partial T}{\partial t} = \nabla \cdot \left([D]C_{sat} \nabla \bar{C} + \bar{C} \frac{\partial C_{sat}}{\partial T} \nabla T \right) + G$$

where

- D : diffusivity matrix
- $C(x,y,z,t)$: concentration
- G : diffusing substance generation rate per unit volume
- ∇ : gradient operator = $\left\{ \frac{\partial}{\partial x} \frac{\partial}{\partial y} \frac{\partial}{\partial z} \right\}$, $\nabla \cdot$: divergence operator

- In the above formulation, C_{sat} is considered as temperature-dependent only.
- However, C_{sat} is both time- and temperature- dependent in general.

Henry's Law in Moisture Diffusion

- $C_{sat} = p_{amb} S$

$$E_p = 4.01 \times 10^4 \left(\frac{\text{J}}{\text{mol}} \right) = 0.415 \text{ (eV)}$$

$$p_0 = 3.82 \times 10^{10} \text{ (Pa)}.$$

p_{amb} : ambient partial vapor pressure $p_{amb} = RH p_g(T) = RH p_0 e^{\left(-\frac{E_p}{RT}\right)}$ (p_g is saturated vapor pressure)

S : solubility $S(T) = S_0 e^{\left(\frac{E_s}{RT}\right)}$

$$C_{sat} = RH p_0 S_0 e^{\left(\frac{E_s - E_p}{RT}\right)} = RH K_0 e^{\left(\frac{-E_k}{RT}\right)} = RH(t) K(T)$$

$$K = K(T) = K_0 \exp\left(-\frac{E_k}{RT}\right)$$

$$K_0 = p_0 S_0, \quad E_k = E_p - E_s$$

- In general, $C_{sat}(RH(t), T(t))$ is both time- and temperature-dependent.
- Therefore, ANSYS cannot solve the problem with varying RH as function of time.

New Normalization Theory - \bar{C}_K Approach

- $\bar{C}_K = \frac{C}{K}$ is used as field variable.
- $K(T)$: generalized solubility (kg/m³)

$$\frac{\partial(K\bar{C}_k)}{\partial t} = \nabla \cdot (D\nabla(K\bar{C}_k))$$

$$\Rightarrow k \frac{\partial \bar{C}_k}{\partial t} + \bar{C}_k \frac{\partial K}{\partial T} \frac{\partial T}{\partial t} = \nabla \cdot \left([D]K\nabla\bar{C}_k + \bar{C}_k \frac{\partial K}{\partial T} \nabla T \right)$$

- The governing equation is exactly identical to the diffusion equation in ANSYS.
- \bar{C}_K turns out to be the water activity a_w (Chen et al.), which has been proved to be continuous at interface.

Analogy using ANSYS

- Field variable and material property input

Field Variable	Material Property
\bar{c}	C_{sat}
\bar{c}_k	K

$$K = \frac{C_{sat}}{RH}$$

- Boundary condition

	\bar{c}	\bar{c}_k
Absorption	1	RH
Desorption	0	0

$$\frac{C_{sat}}{K} = RH(t)$$

- Initial condition

	\bar{c}	\bar{c}_k
Absorption	0	0
Desorption	1 (if fully saturated initially)	1 (if fully saturated initially)

- In using ANSYS,

- \bar{c} is replaced by \bar{c}_k .
- Temperature-dependent material property C_{sat} is replaced by general solubility K .
- Boundary condition is now changed to $\bar{c}_k = RH$.

Element Options in ANSYS

- **Two different types of element option**
 - **Diffusion element**
 - Plane 238 (2D, Quadratic)
 - Solid 239 (3D, Quadratic)
 - ♦ Degree of Freedom (DOF) label; CONC
 - **Coupled element**
 - Plane 223(2D , Quadratic)
 - Solid 226(3D , Quadratic)
 - Thermal-diffusion (100010)
 - ♦ DOF label: TEMP,CONC
 - Thermal-structural-diffusion (100011)
 - ♦ DOF label: TEMP, UX, UY, UZ , CONC
 - Structural-diffusion (100001)
 - ♦ DOF label: UX,UY,UZ, CONC
- Since the general diffusion equation in ANSYS involves with temperature derivative with respect to time when K is temperature-dependent, the coupled element (thermal diffusion, or thermal-structural-diffusion) must be used.

Case Study – Varying RH and Temperature

- A 1-D bi-material absorption problem with varying RH and Temperature

- Initial condition: dry at 30°C
- Ambient temperature profile is given below to simulate a absorption process

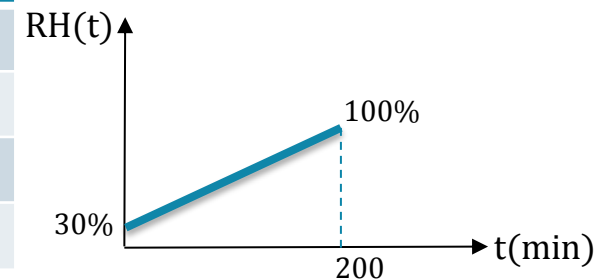
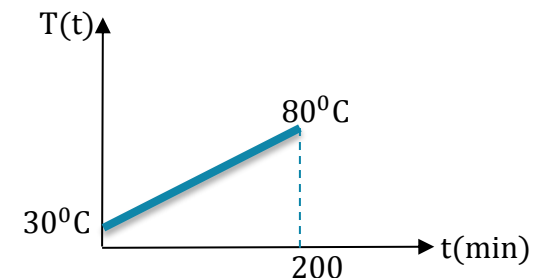
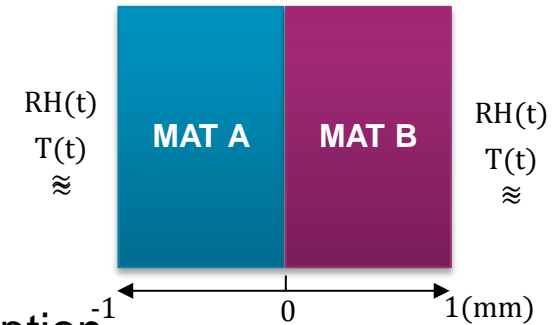
$$T(t) [^{\circ}\text{C}] = 30 + \frac{50t}{200} \quad 0 < t \leq 200 \text{ (min)}$$

- Relative humidity profile is given below to simulate a absorption process

$$\text{RH}(t) = 0.3 + \frac{0.7t}{200} \quad 0 < t \leq 200 \text{ (min)}$$

- Material properties

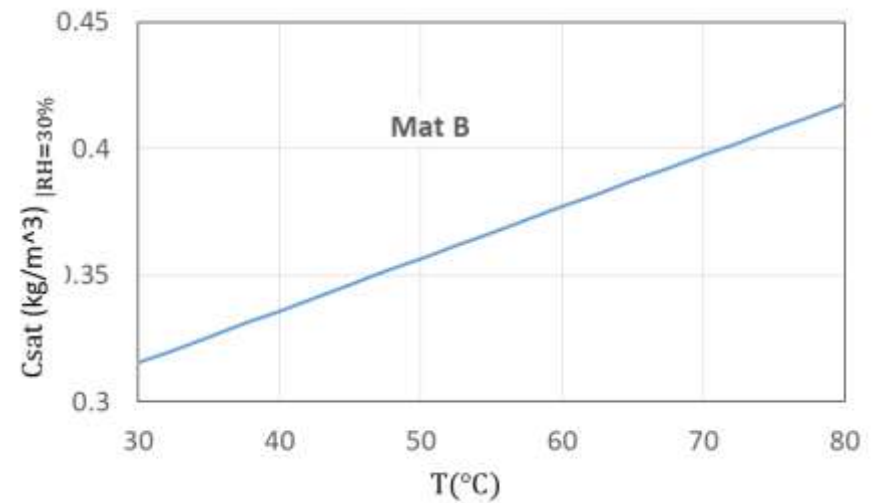
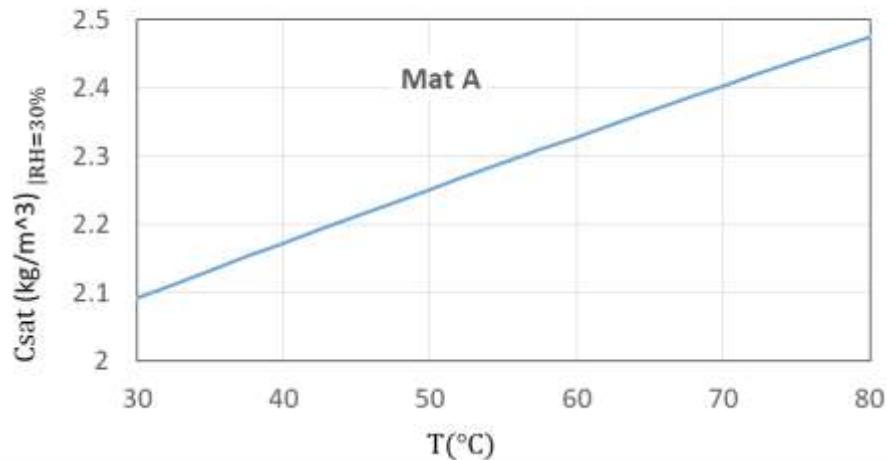
Properties	Mat A	Mat B
$D_0(\text{m}^2/\text{s})$	5.0×10^{-3}	4.0×10^{-3}
$E_D(\text{ev})$	0.518	0.518
$S_0(\text{Kg}/\text{m}/\text{Pa})$	6.0×10^{-10}	2.0×10^{-10}
$E_s(\text{ev})$	0.383	0.362



- For this problem, C_{sat} is both time and temperature-dependent.

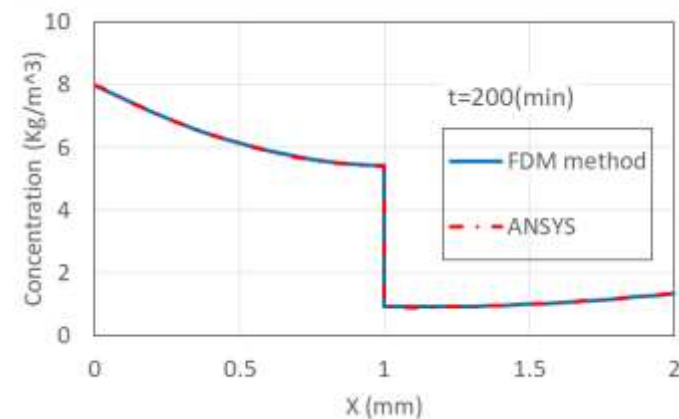
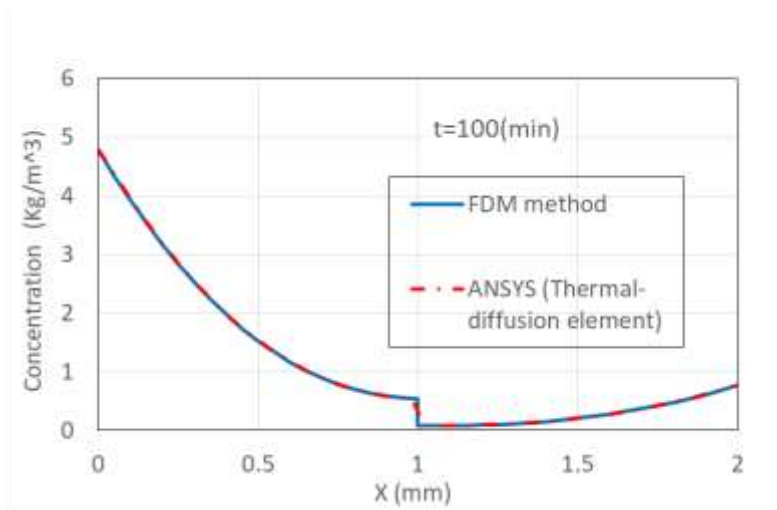
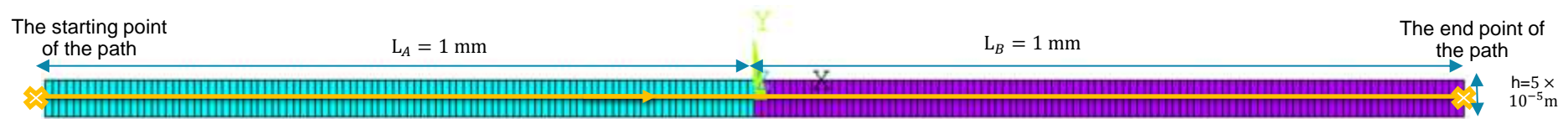
C_{sat} as a Function of Temperature

- For the given material properties (C_{sat} at 30%RH)



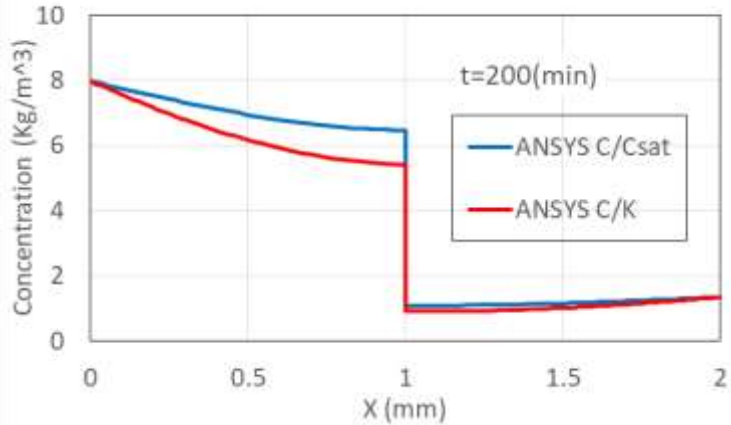
Finite Element Model and Results

- 1-D problem is constructed by a 2-D element strip model.
- Finite difference method (FDM) is used for verification.



- Coupled thermal-diffusion element presents correct results.

\bar{C} vs. \bar{C}_k Approach



- \bar{C} (coupled element with thermal-diffusion)

$$C_{sat} \frac{\partial \bar{C}}{\partial t} + \bar{C} \frac{\partial C_{sat}}{\partial T} \frac{\partial T}{\partial t} + \bar{C} \frac{\partial C_{sat}}{\partial t} = \nabla \cdot \left([D] C_{sat} \nabla \bar{C} + \bar{C} \frac{\partial C_{sat}}{\partial T} \nabla T \right)$$

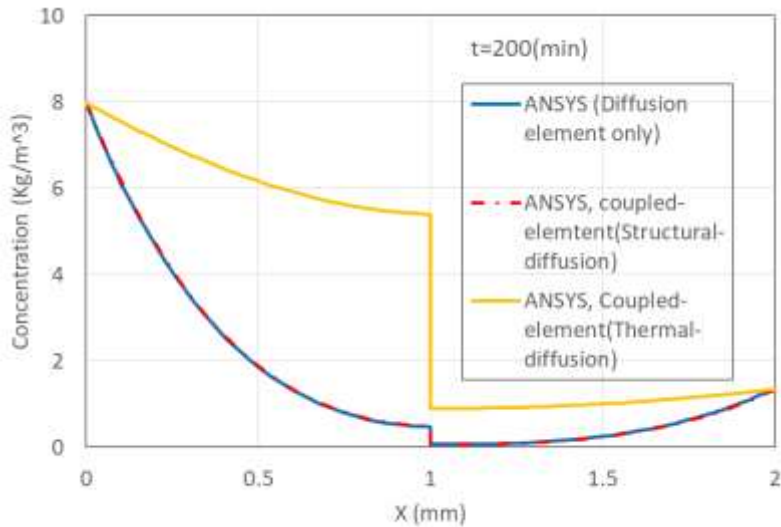
Not included in ANSYS

- \bar{C}_k (coupled element with thermal-diffusion)

$$K \frac{\partial \bar{C}_k}{\partial t} + \bar{C}_k \frac{\partial K}{\partial T} \frac{\partial T}{\partial t} = \nabla \cdot \left([D] K \nabla \bar{C}_k + \bar{C}_k \frac{\partial K}{\partial T} \nabla T \right)$$

- Since C_{sat} is temperature-dependent and time-dependent, the results obtained by \bar{C} are incorrect.

Comparison Among Different Elements in ANSYS



- Diffusion element only

$$K \frac{\partial \bar{C}_k}{\partial t} = \nabla \cdot ([D]K \nabla \bar{C}_k)$$

- Coupled element with structural-diffusion

$$K \frac{\partial \bar{C}_k}{\partial t} = \nabla \cdot ([D]K \nabla \bar{C}_k)$$

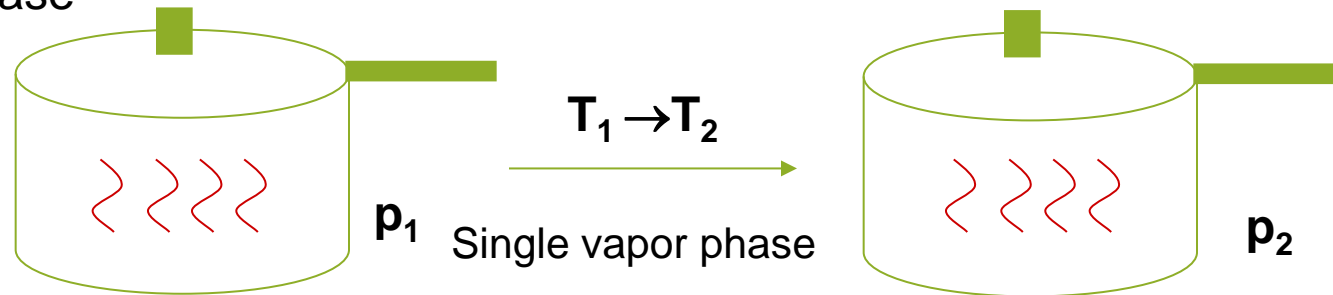
- Coupled element with thermal-diffusion

$$K \frac{\partial \bar{C}_k}{\partial t} + \bar{C}_k \frac{\partial K}{\partial T} \frac{\partial T}{\partial t} = \nabla \cdot \left([D]K \nabla \bar{C}_k + \bar{C}_k \frac{\partial K}{\partial T} \nabla T \right)$$

- The Diffusion element only and the coupled element with structural-diffusion option present the same incorrect results.

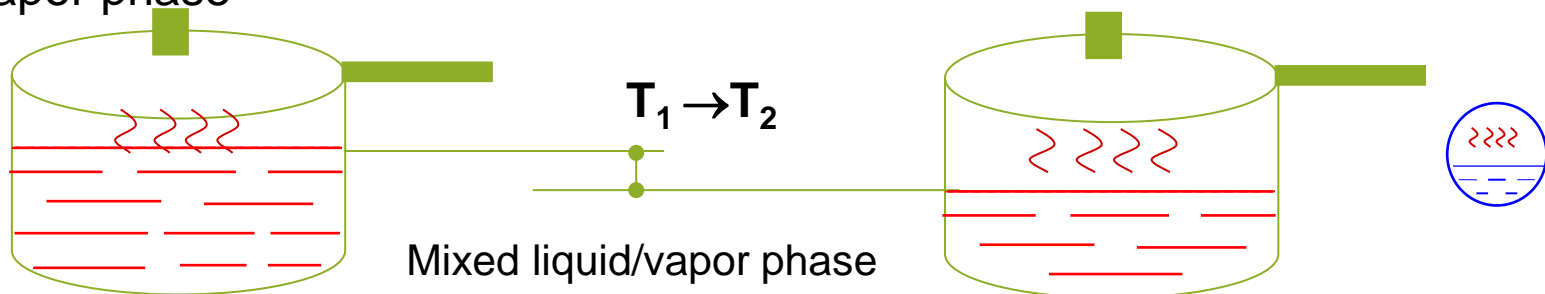
Vapor Pressure Model - Example: Pressure Cooker

- Vapor phase



- Moisture in single vapor phase;
- $\rho < \rho_g(T)$, ρ : **moisture density over the total volume of cooker**; $\rho_g(T)$: **saturated moisture vapor density**.
- **Ideal gas law can be used: $p_2 = p_1 T_2 / T_1$**

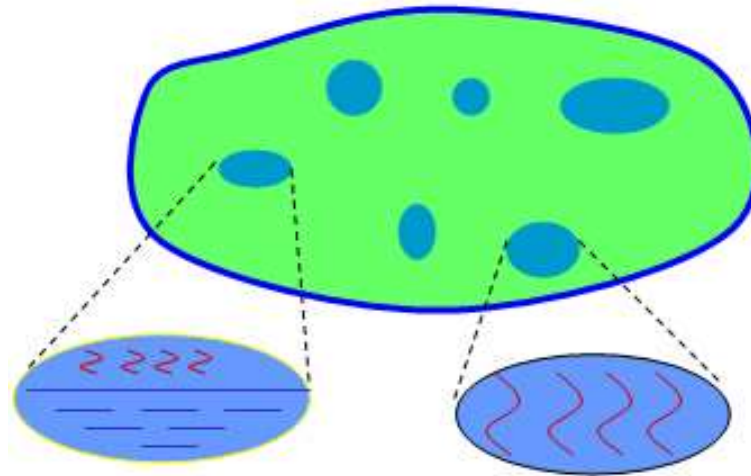
- Liquid/vapor phase



- Moisture in two-phases – water/vapor mixed;
- $\rho > \rho_g(T)$ - liquid-vapor phase , ρ : **moisture density over the total volume of cooker**; $\rho_g(T)$: **saturated moisture vapor density**
- Saturated vapor pressure remains regardless of water level

Vapor Pressure Model

- Moisture in free-volumes 'free' to vaporize
- Two distinct states
 - Single vapor phase
 - Mixed liquid/vapor phase
- Saturated moisture density $\rho_g(T)$



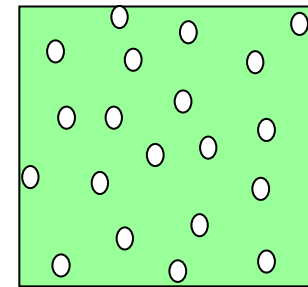
Mixed liquid/vapor phase

Single vapor phase

Vapor Pressure Model

$$\rho = \frac{dm}{dV_f} = \frac{dm}{dV} \frac{dV}{dV_f} = C / f$$

- dV_f : void volume in a REV (representative elementary volume)
 dV : total volume of a REV
 dm : total moisture mass
 f : void volume fraction dV_f/dV
 ρ : aparent moisture density



REV

$$p(T) = \frac{RT}{MM_{H_2O} f} \cdot C$$

when $C(T) / f < \rho_g(T)$

$$p(T) = p_g(T),$$

when $C(T) / f \geq \rho_g(T)$

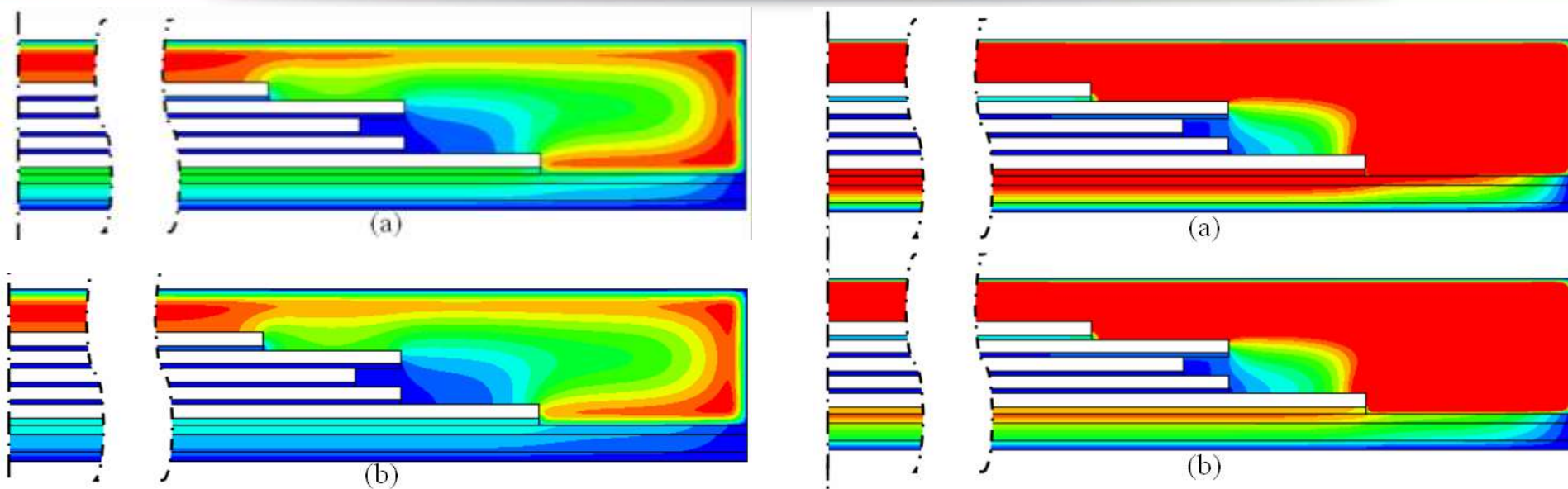
Fan XJ, Zhou J, Zhang GQ, Ernst LJ. A micromechanics based vapor pressure model in electronic packages. *ASME Journal of Electronic Packaging* 127 (3), 262-267. 2005.

Xie B, Fan XJ, Shi XQ, Ding H. Direct concentration approach of moisture diffusion and whole field vapor pressure modeling for reflow process: part I – theory and numerical implementation. *ASME Journal of Electronic Packaging* 131(3), 031010. 2009.

Steam Table - p_g

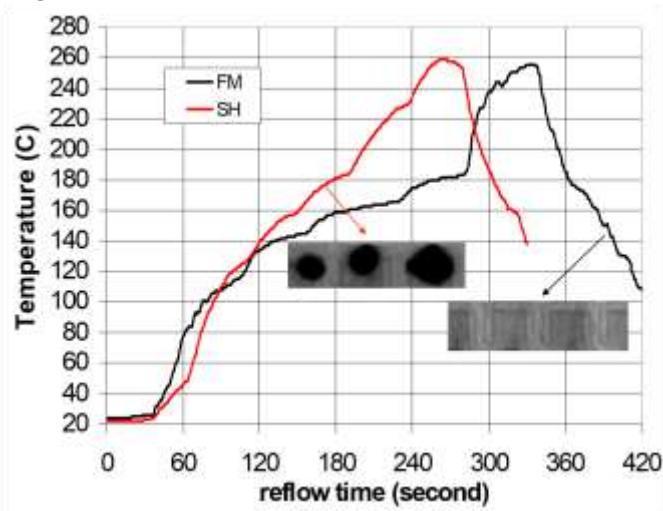
$T(^{\circ}\text{C})$	20	30	40	50	60	70	80
$\rho_g(\text{g}/\text{cm}^2 \times 10^{-3})$	0.017	0.03	0.05	0.08	0.13	0.2	0.29
$p_g(\text{MPa})$	0.002	0.004	0.007	0.01	0.02	0.03	0.05
$T(^{\circ}\text{C})$	90	100	110	120	130	140	150
$\rho_g(\text{g}/\text{cm}^2 \times 10^{-3})$	0.42	0.6	0.83	1.12	1.5	1.97	2.55
$p_g(\text{MPa})$	0.07	0.1	0.14	0.2	0.27	0.36	0.48
$T(^{\circ}\text{C})$	160	170	180	190	200	210	220
$\rho_g(\text{g}/\text{cm}^2 \times 10^{-3})$	3.26	4.12	5.16	6.4	7.86	9.59	11.62
$p_g(\text{MPa})$	0.62	0.79	1	1.26	1.55	1.91	2.32
$T(^{\circ}\text{C})$	230	240	250	260	270	280	290
$\rho_g(\text{g}/\text{cm}^2 \times 10^{-3})$	14	16.76	19.99	23.73	28.1	33.19	39.16
$p_g(\text{MPa})$	2.8	3.35	3.98	4.69	5.51	6.42	7.45

Vapor Pressure: Effect of Reflow Profiles



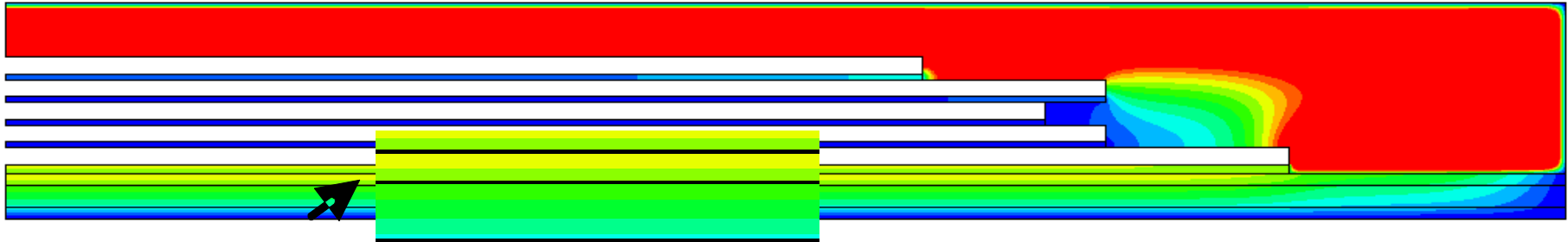
Moisture diffusion modeling: a) SH, b) FM

Vapor pressure modeling: a) SH, b) FM

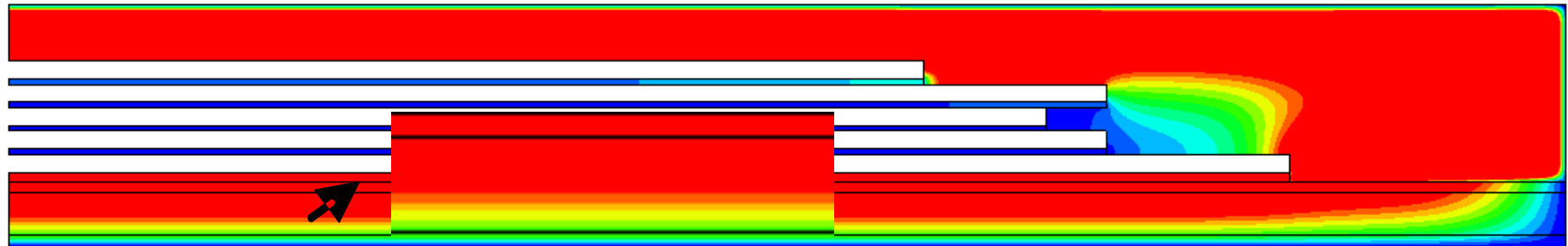


Vapor Pressure Modeling

CSP with thinner substrate

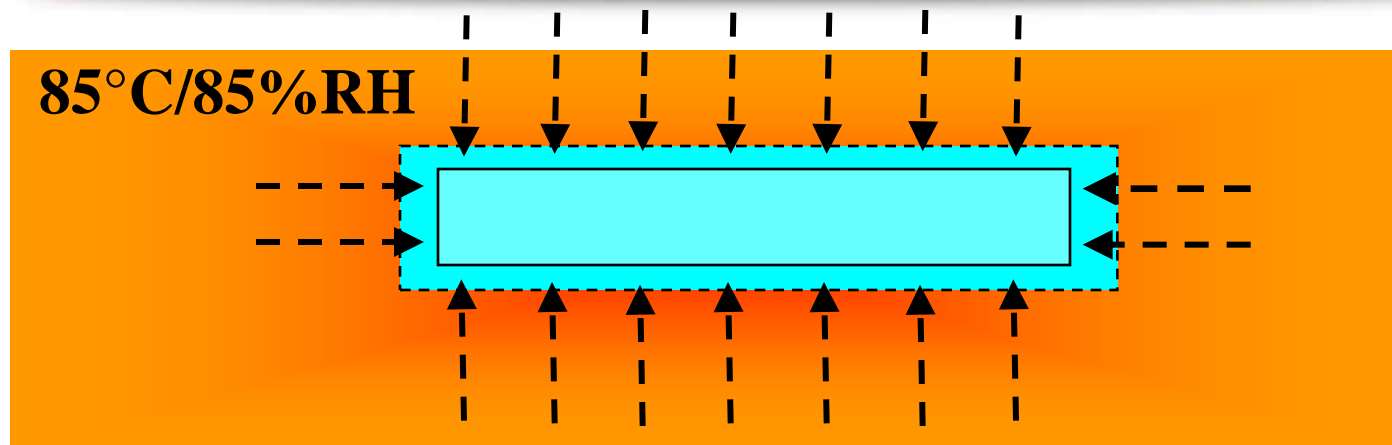


CSP with thicker substrate



- About 50% reduction on vapor pressure at 250°C in the bottom DA film between two thicknesses of substrate

Hygroscopic Swelling



Expansion strain
due to hygroscopic
swelling

~0.29%

$$\varepsilon^{\text{hygro}} = \beta * C$$

β – the coefficient of
hygroscopic swelling
 C – moisture concentration

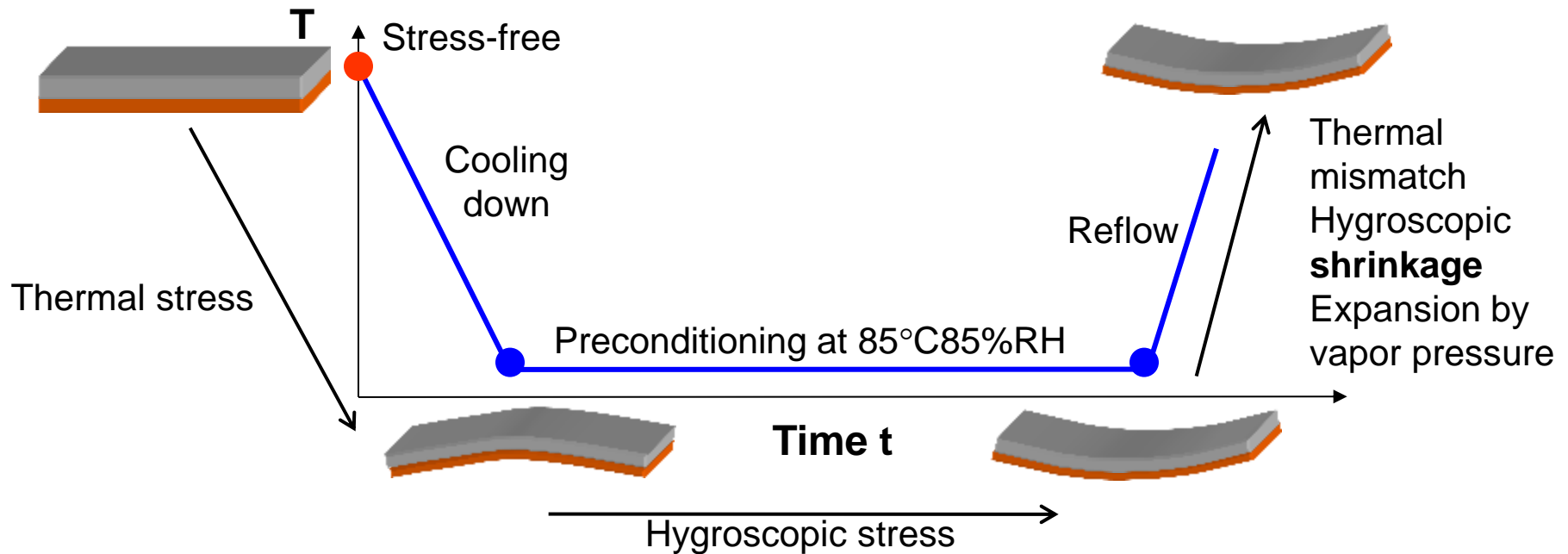
Expansion strain
due to temperature
change of 100°C

~0.25%

$$\varepsilon^{\text{thermal}} = \alpha * \Delta T$$

Hygroscopic mismatch is comparable to thermal mismatch in causing mechanical stresses

Temperature-Moisture-Deformation Problems



- How to perform integrated modeling to accurately capture stress developments in each stage?

Material Properties to be Needed

$$\varepsilon_{ij} = \frac{1+\nu}{E} \sigma_{ij} - \frac{\nu}{E} \sigma_{kk} \delta_{ij} + (\alpha\Delta T + \beta C + \frac{1-2\nu}{E} p) \delta_{ij}$$

α : coefficient of thermal expansion (CTE)

β : coefficient of hygroscopic swelling (CHS)

C: moisture concentration

T: temperature, p: vapor pressure

Implementation in ANSYS

$$\varepsilon^{\text{volu}} = \alpha \Delta T + \beta C + \frac{1-2\nu}{E} p$$

$$\Rightarrow \varepsilon^{\text{volu}} = \alpha \Delta T + \beta \left(C + \frac{(1-2\nu)p}{\beta E} \right)$$

BF,,TEMP,n_temp

BF,,FLUE, b1+b2

Body Forces in ANSYS: Temp and Flue

- Total volume strain

$$\varepsilon = \alpha (T - T_{\text{ref}}) + \beta C$$

\swarrow \searrow
BF,,TEMP,n_temp **BF,,FLUE,n_C**

ANSYS built-in swelling function $\varepsilon^{sw} = \beta(\Delta C)^n$

ε^{sw}	:swelling strain
β	:swelling coefficient
C	:a fluence

```

TB,SWELL,1
! C72=10 FOR ACTIVATION OF USERSW
! EPS=C67*(FLUENCE)^C68 WHERE C67=SWELLING COEFFICIENT
TBDATA,72,10
TBDATA,67,β,1
  
```

Implementation in ANSYS

```
TB,SWELL,MAT
TBDATA,72,10
TBDATA,67, $\beta$ ,1
```

```
Idread,temp,,,,,moisture,rth
```

```
*do,i,1,total_node
```

```
*get,C,node,i,ntemp
```

```
BF,i,FLUE,C
```

```
*enddo
```

```
BF,,Temp,T_reflow
```

```
solve
```

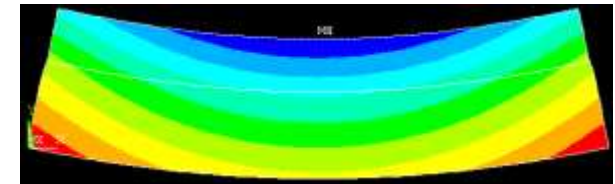
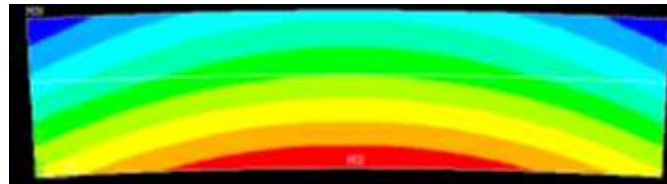
- Moisture diffusion (including desorption) modeling needs to be solved first and separately.
 - ANSYS 17.2 or later provides **coupled element**
 - Plane 223(2D , Quadratic)
 - Solid 226(3D , Quadratic)
 - Thermal-structural-diffusion (100011)
 - ◆ DOF label: TEMP, UX, UY, UZ , CONC

Example: Bi-Material Configuration

Moisture
concentration
contours

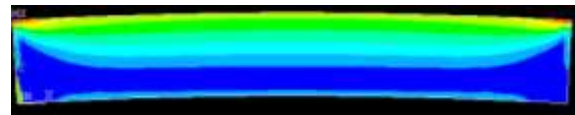


Deformed
shape
contours

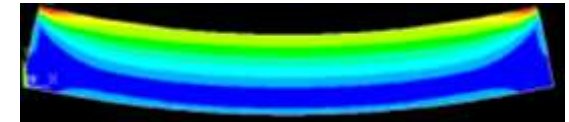


von Mises
stress
Contours in MC

Max stress =4.96MPa



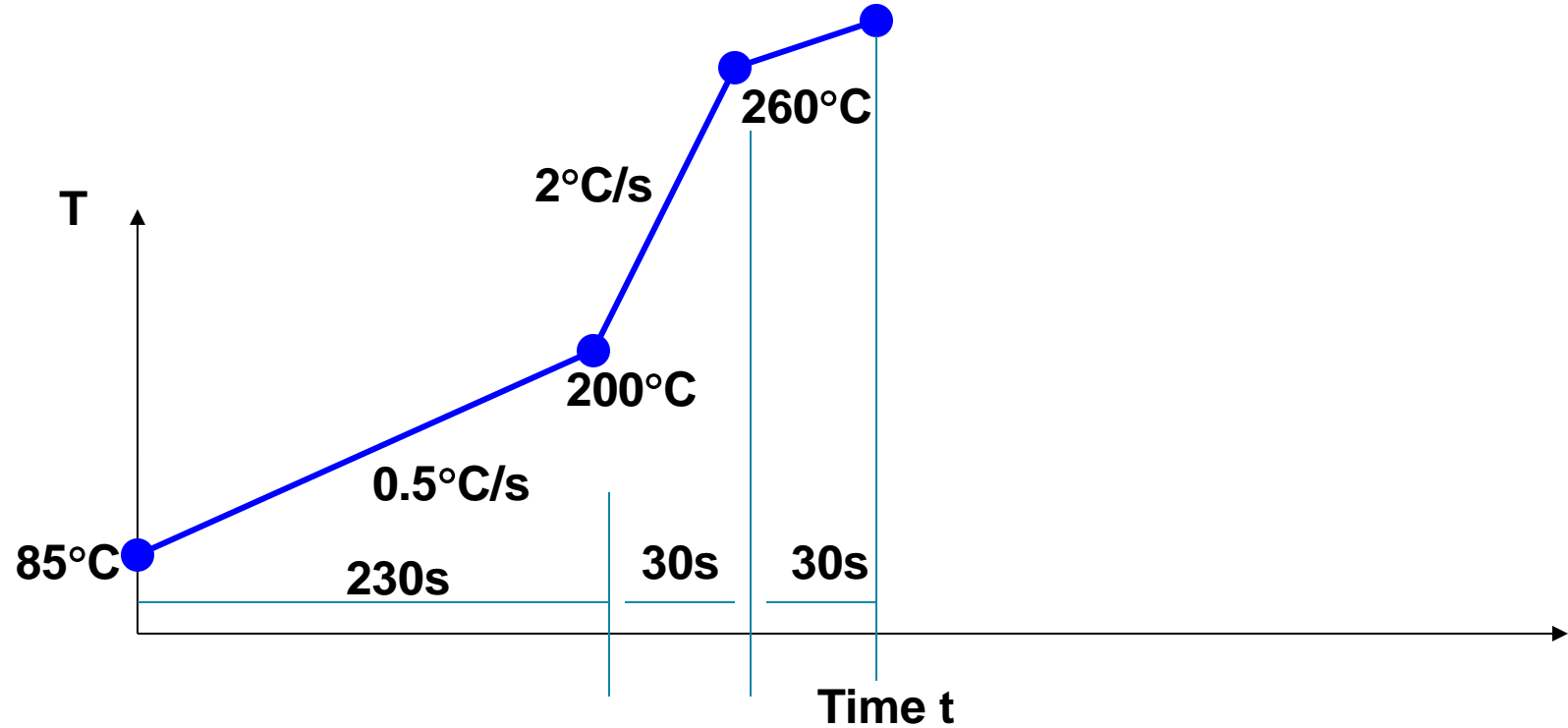
Max stress =20.99MPa



Time = 1hour @85°C/85%RH

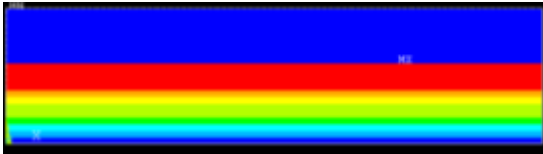
Time = 18hour @85°C/85%RH

Example: Reflow Profile Setting

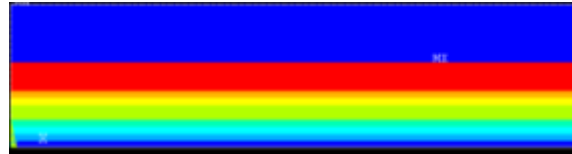


Moisture Concentration and Vapor Pressure Contours

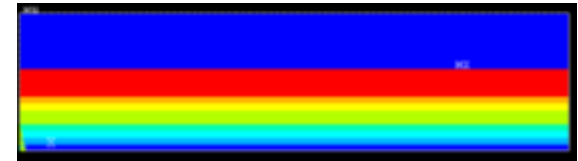
Max C = $3.54\mu\text{g}/\text{mm}^3$
T=200°C



Max C = $2.11\mu\text{g}/\text{mm}^3$
T=260°C



Max C = $1.05\mu\text{g}/\text{mm}^3$
T=265°C

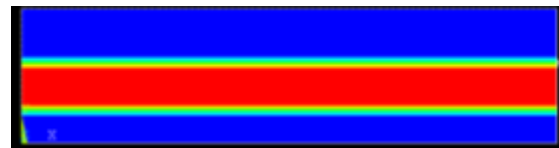


Moisture concentration contours

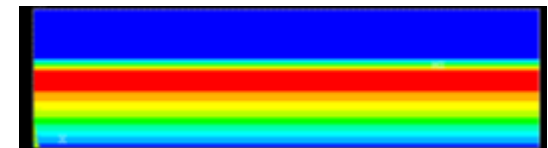
Max p = 1.55MPa
T=200°C



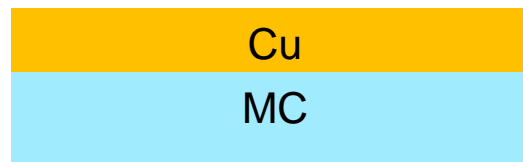
Max p = 5.84MPa
T=260°C



Max p = 0.5e-3MPa
T=265°C



Vapor pressure contours



Results of Integrated Stress Modeling

200°C

	Von Mises Stress in Epoxy (MPa)	Von Mises Stress in Cu (Mpa)
T	2.4	8.7
T + H	4.9	17.3
T + H + V	6.3	22.9

260°C

	Von Mises Stress in Epoxy (MPa)	Von Mises Stress in Cu (Mpa)
T	6.0	21.9
T + H	7.2	27.5
T + H + V	11.2	38.9

**260°C after
30sec hold**

	Von Mises Stress in Epoxy (MPa)	Von Mises Stress in Cu (Mpa)
T	6.0	21.9
T + H	6.3	24.7
T + H + V	6.3	24.7

Summary

- **For a general moisture diffusion problem with temperature-dependent C_{sat} and varying ambient RH and temperature with time**
 - \bar{C}_k must be used and the coupled element with thermal-diffusion or thermal-structural-diffusion option must be applied at the same time.
- **ANSYS built-in \bar{C} approach cannot solve the problem with varying RH correctly.**
 - ANSYS diffusion element only
 - C_{sat} must be temperature-independent. Temperature gradient is not considered.
 - ANSYS coupled element with structural-diffusion option.
 - C_{sat} must be temperature-independent. Temperature gradient is not considered.
 - ANSYS coupled element with thermal-diffusion option (or thermal-structural-diffusion) option
 - If \bar{C} is used, RH must be constant.
 - If \bar{C}_k is used, no restriction for any diffusion problems.
- **Vapor pressure model**
- **Thermal-hygro-mechanical modeling**

- Introduction
- Temperature Loading
- Mechanical Loading
- Moisture and Humidity
- **Electrical Current - Multi-Physics Modeling**
- Summary

Outline

- **Electrical-Thermal-Mechanical Modeling**
- **Electromigration**

Electrical-Thermal Modeling

- Governing equations

Current flow:
(electrostatics)

$$\nabla^2 V = 0 \quad \Longrightarrow \quad \frac{\partial^2 V}{\partial x^2} + \frac{\partial^2 V}{\partial y^2} + \frac{\partial^2 V}{\partial z^2} = 0$$

Heat conduction:
(steady-state)

$$k_T \nabla^2 T + \frac{1}{\rho} |\nabla V|^2 = 0 \quad \text{Joule heating} \quad \left| \vec{j} \cdot \nabla V \right| = \frac{1}{\rho} |\nabla V|^2$$

$$\vec{j} = -\frac{1}{\rho} \nabla V$$

- Current flow generates joule heating at any location.
- Joule heat is applied as heat source in heat diffusion in solids.

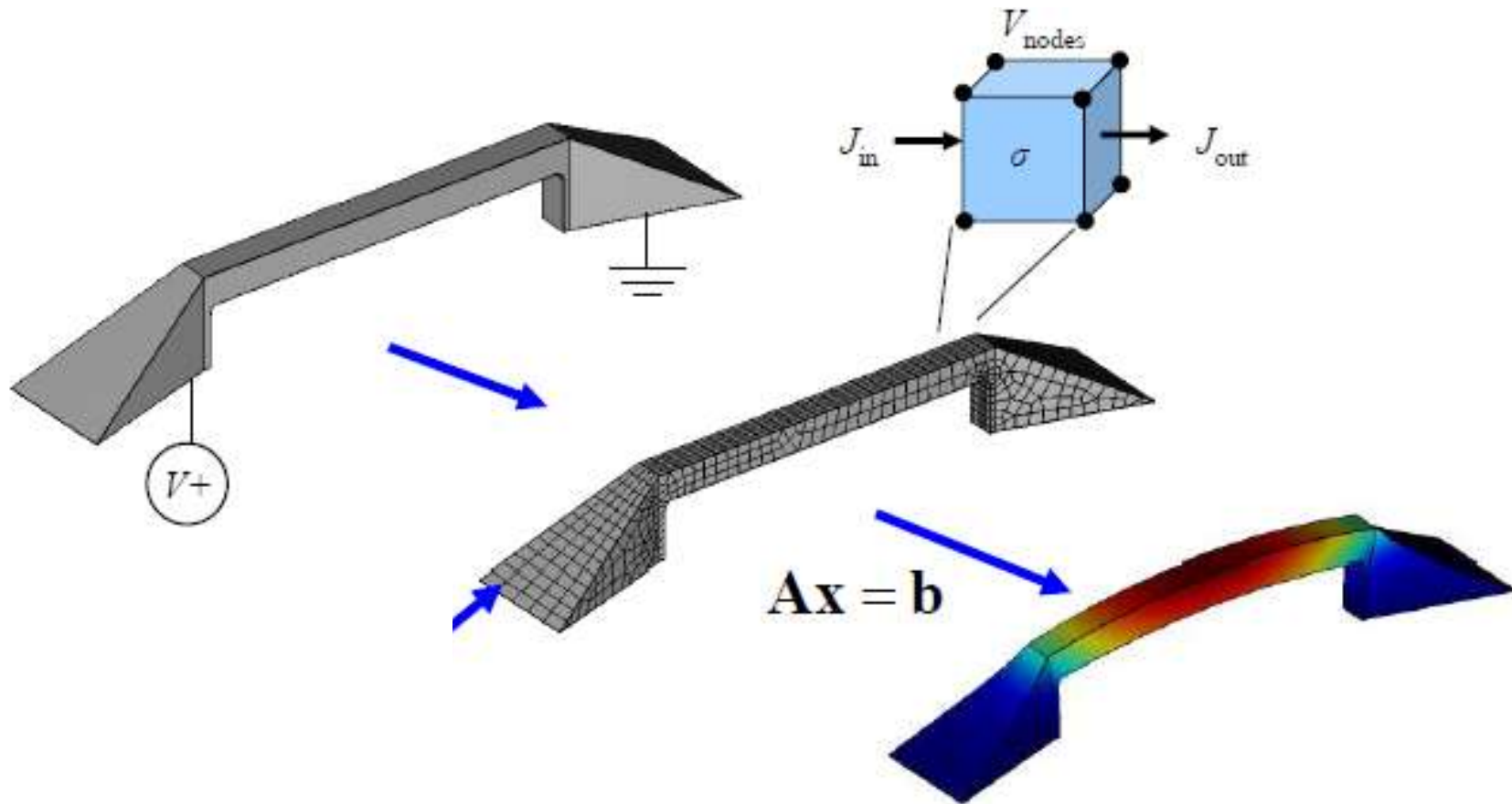
- Joule heat term:

$$q_0 = \frac{1}{\rho} |\nabla V|^2$$

- We solve this multi-physics problem one step at a time
 - Both problems are diffusion-type
 - Two problems are sequentially coupled if electric conductivity does not vary with temperature

Example: A Microresistor Beam in MEMS

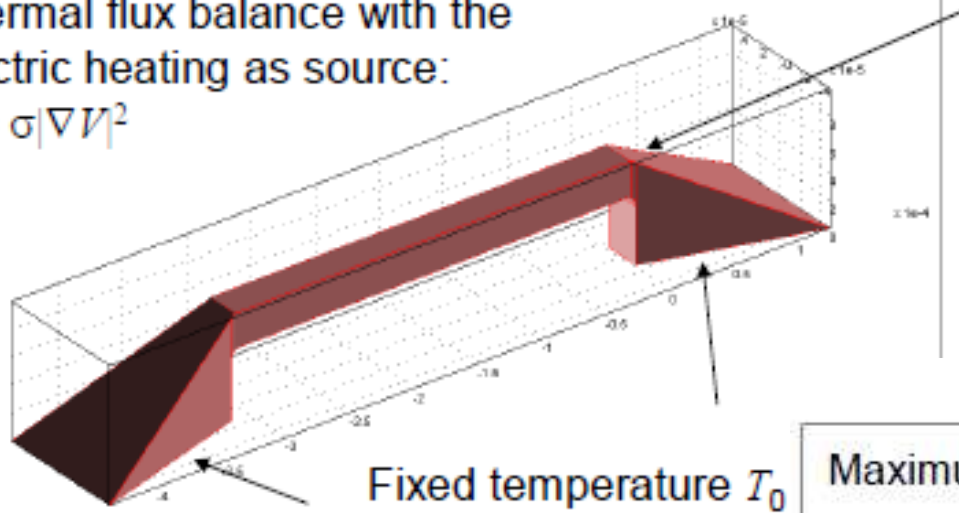
- Current flow modeling



Thermal Modeling

Inside the material:
Thermal flux balance with the
electric heating as source:

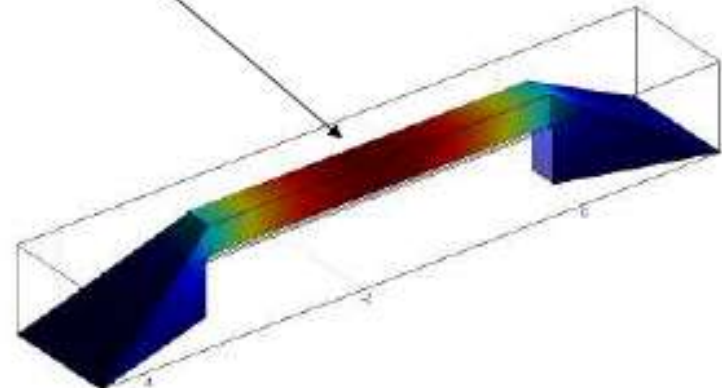
$$Q = \sigma |\nabla V|^2$$



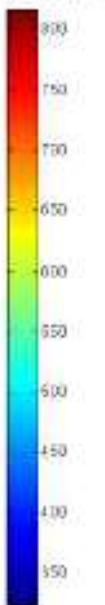
Convection
conditions
flux out:
 $h^*(T-T_{\text{amb}})$

Boundary Temperature

Maximum temperature



Max: 815.90



Min: 322.977

Thermal-Mechanical Modeling

- Governing equations

$$\nabla \cdot k[\nabla T] = c_p \rho \frac{\partial T}{\partial t}$$

$$G\nabla^2 u_i + (\lambda + G)e_{,i} - \left[\frac{E}{1-2\nu} \alpha T \right]_{,j} + X_i = 0$$

u_i : the component of displacement vector

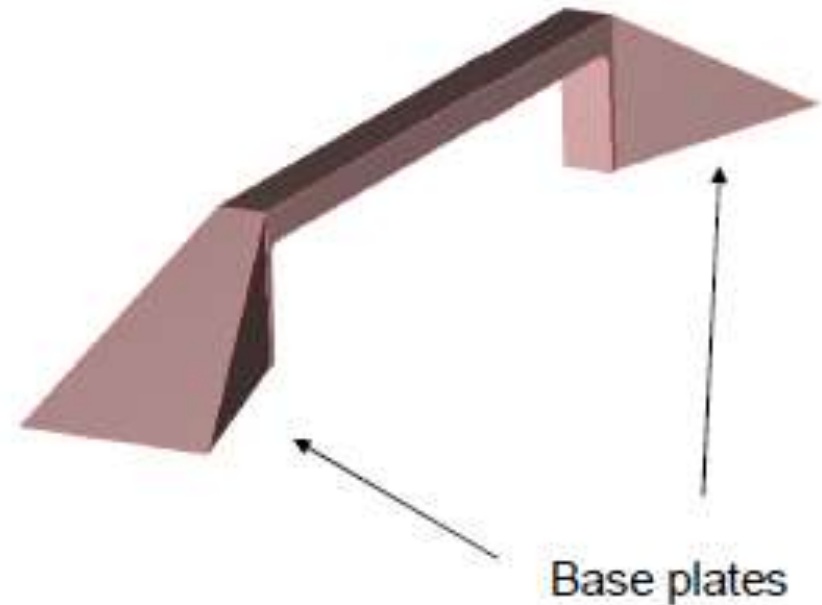
e : the total volumetric strain $e = u_{i,i}$

X_i : the component of body force vector.

- Temperature load is applied in the form of 'body force' in mechanical analysis

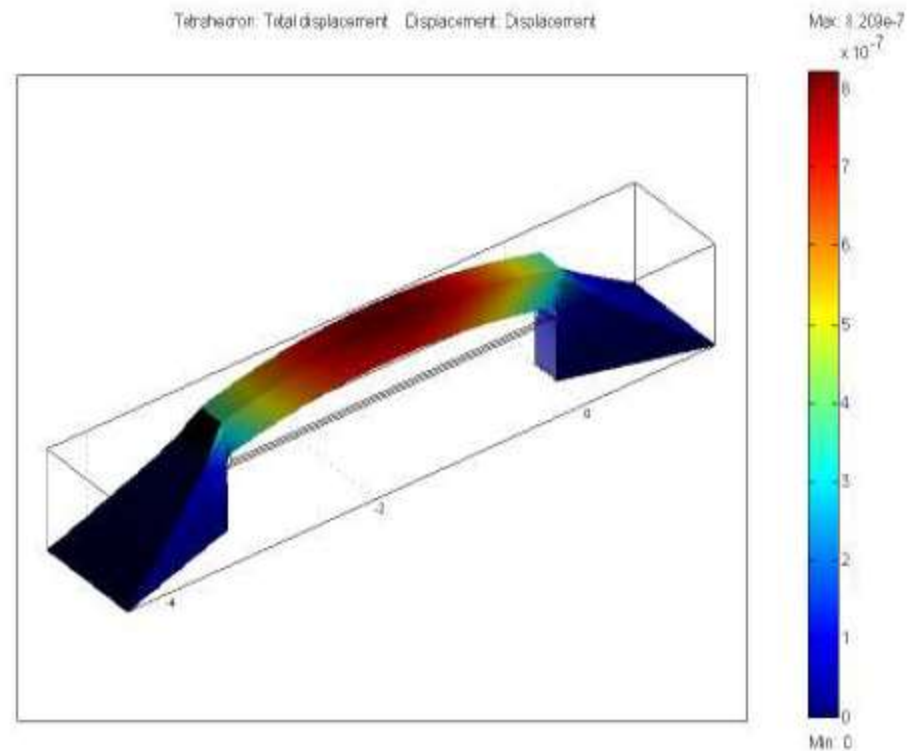
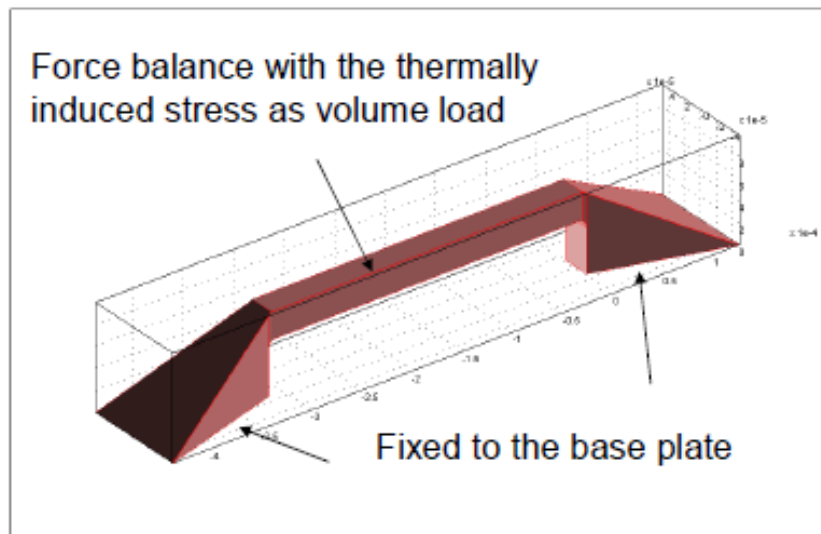
Electrical-Thermal-Mechanical Modeling

- MEMS: microresistor beam modeling
- MEMS actuator application
- Electrical current
 - DC balance for conductive media
 - Fixed potentials generate potential difference $\Delta V=2V$
- Heat transfer
 - Thermal flux balance with the resistive heating as source
- Structural analysis
 - Force balance with the thermally induced stress as volume load



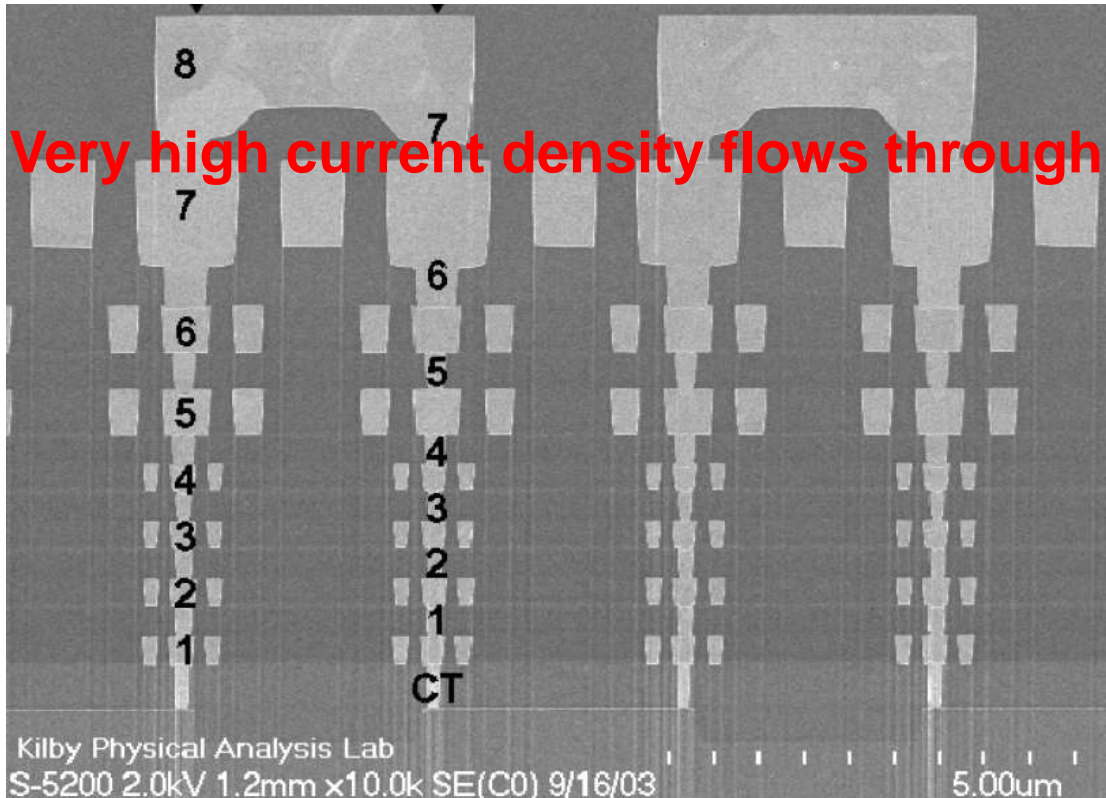
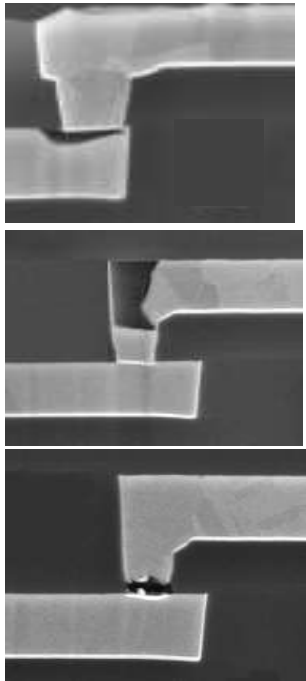
Mechanical Modeling

- Results, Deformation

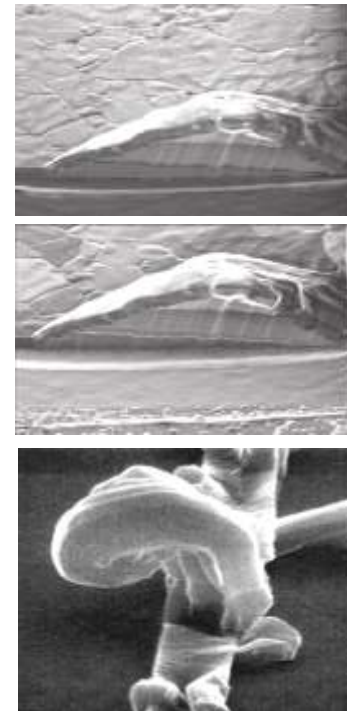


What is Electromigration (EM)?

Void



Hillock

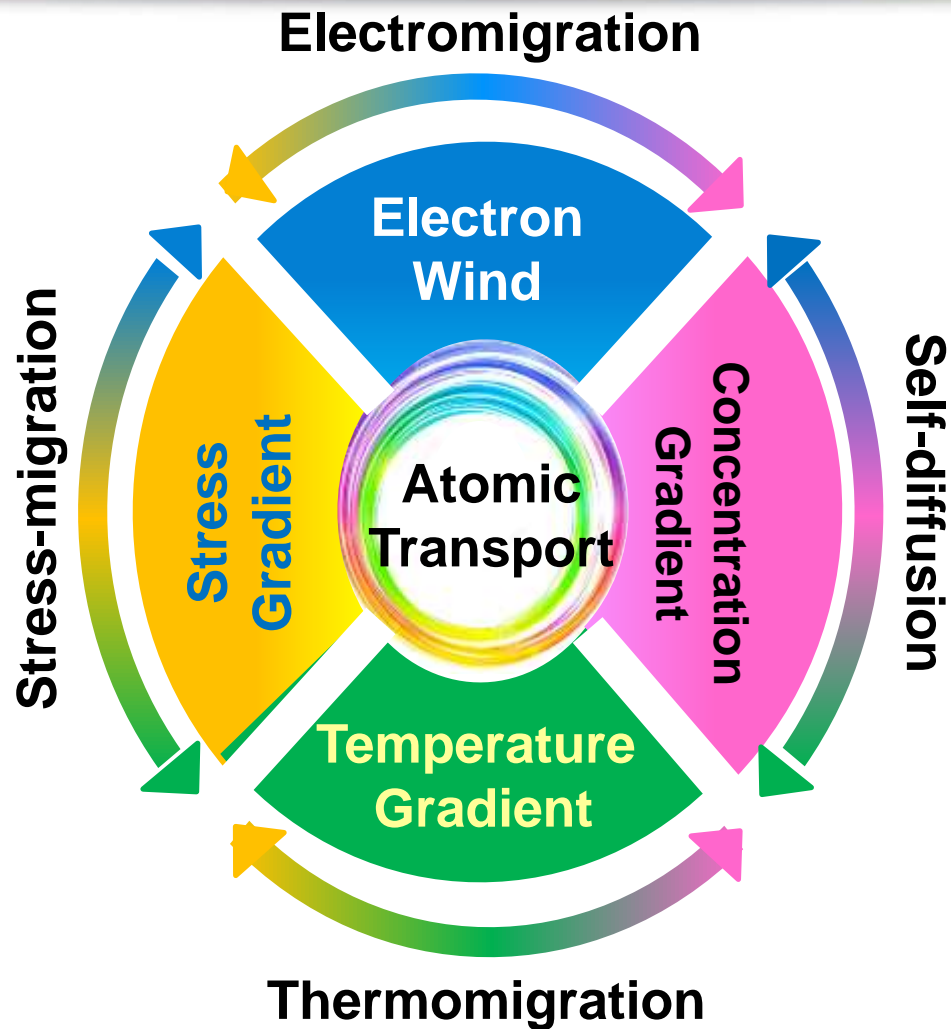


- Electromigration is a process of mass transport in the current-carrying metal under the driving forces generated by electric field.
- Electromigration is the most persistent reliability problem in interconnect technology in semiconductor device.

Similar to River Flow...

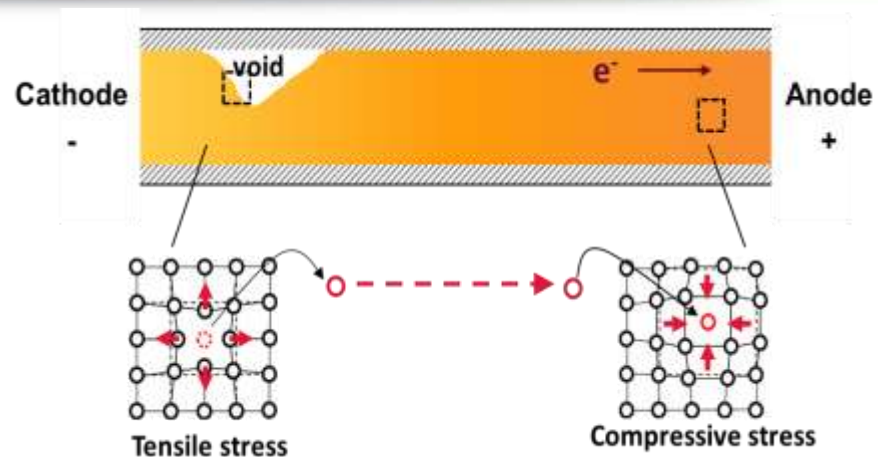
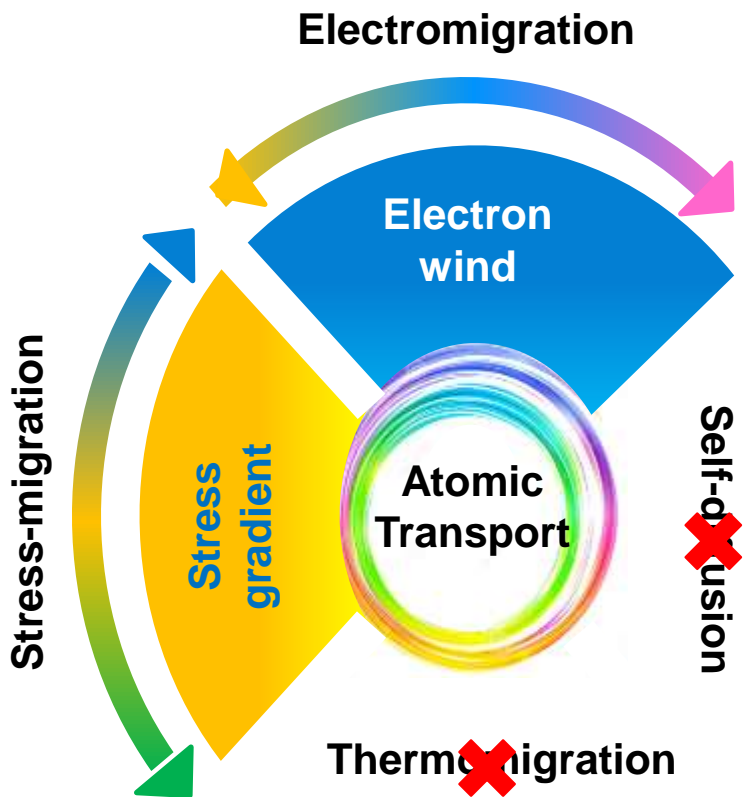


Driving Forces



- The atomic transport is caused by a combination of interacting driving forces that can generate voids at different locations.

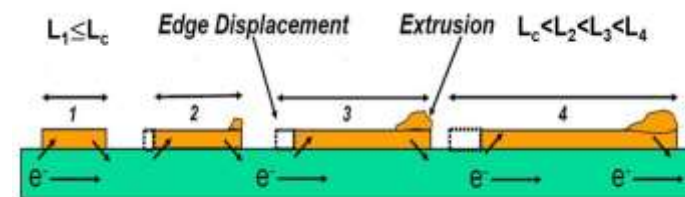
Blech's Theory (1976)



$$Z^* e \rho j + \Omega \frac{\partial \sigma}{\partial x} = 0$$



$$jL = \frac{(\sigma_{max} - \sigma_{min}) \Omega}{Z^* e \rho}$$



- EM flux is entirely balanced by the stress-induced counter flux.
- Blech Product, jL , provides a threshold condition of *maximum stress*, below which, electromigration failure will not occur - mechanical failure.

Electromigration – a Coupled, Multi-Physics Problem

Total flux in terms of C_v (vacancy concentration)

$$J_v = \frac{D_v C_v}{k_B T} \left(Z^* e \rho \mathbf{j} - \frac{k_B T}{C_v} \nabla C_v - f \Omega \nabla \sigma + \frac{Q^*}{T} \nabla T \right)$$

C_v vacancy concentration (m^{-3}),

D_v diffusivity

k_B Boltzmann's constant (J/K),

Z^* effective charge number (>0),

f volume relaxation ratio

Ω volume of per atom (m^3)

\mathbf{j} current density (A/m^2)

e elementary charge (C)

Q^* heat of transport (kJ/mol)

- EM is a multiphysics coupled field problem involved with electron wind, chemical potential, stress gradient and temperature gradient.

An Overview in Literature

	Flux by self-diffusion	Flux by stress	Source/sink term	Constraint condition	Stress equilibrium	EM strain in stress/strain
Shatzkes and Lloyd (1986)	✓	✗	✗	N/A	N/A	N/A
Kirchheim (1992)	✗	✓	✓	✗	✗	✗
Korhonen et al. (1993)	✗	✓	✓	✗	✗	✗
Clement and Thompson (1995)	✗	✓	✓	✗	✗	✗
Sarychev et al. (2000)	✓	✓	✓	N/A	✗	✓
Suo et al. (2003, 2011, 2014)	✗	✓	✓	✗	✗	✗
Sukharev et al. (2004, 2007)	✓	✓	✓	N/A	✓	✗
Maniatty et al. (2016)	✗	✓	✓	✗	✓	✓

- **Inconsistent and incomplete solutions appear in literature.**

New Theory – General Coupling Model

	Flux by self-diffusion	Flux by stress	Source/sink term	Constraint condition	Stress equilibrium	EM strain in stress/strain
Cui et al. (2019)	✓	✓	✓	✓	✓	✓

General coupling model for electromigration and one-dimensional numerical solutions

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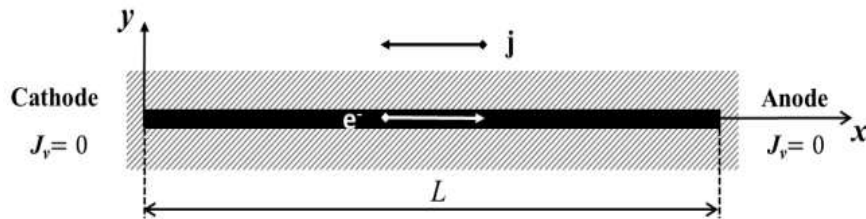
Puzzle One



Volume Strain in Confinement

Literature

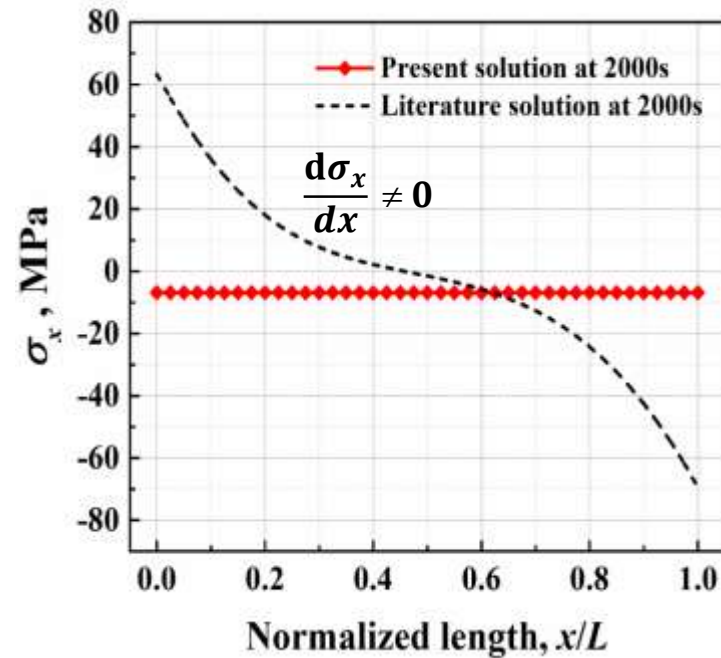
$\theta = 0$ (volume strain equals to zero) anywhere.



Confined conductor configuration

Present solution

$$u(L) = u(0) = 0 \rightarrow \int_0^L \varepsilon_x dx = 0 \rightarrow \int_0^L \theta dx = 0$$



- Equilibrium ($\frac{d\sigma_x}{dx} = 0$) is violated in the existing literature theory and solutions.

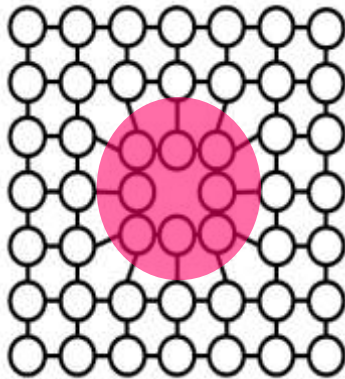
Puzzle Two



- Hydrostatic Stress σ vs. Vacancy Concentration C_v

Literature

$$\sigma = \frac{k_B T}{\Omega} \ln\left(\frac{C_v}{C_{v0}}\right)$$

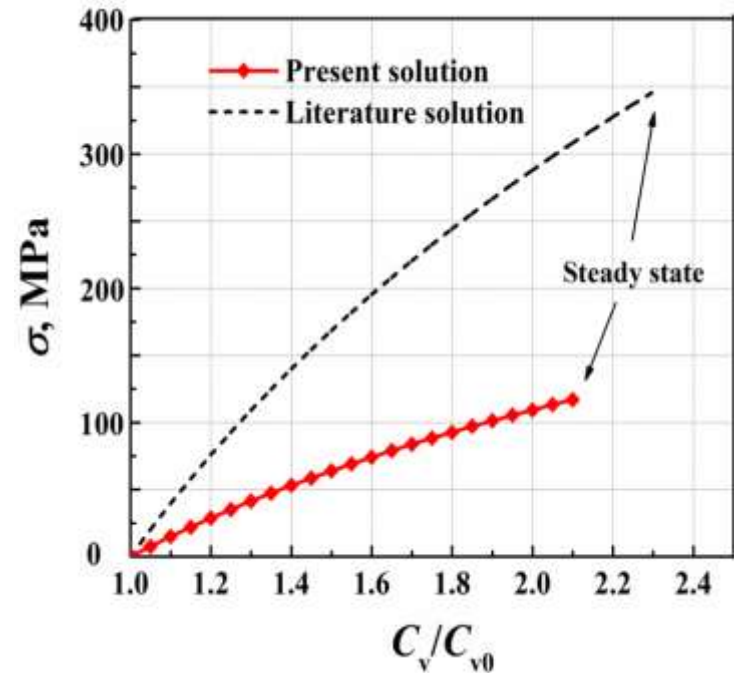


Confined configuration

C. Herring, J. Appl. Phys. 21, 437 (1950)

Present solution

$$\sigma = \frac{2EA}{9(1-\nu)} \left[\frac{1+\nu}{2(1-2\nu)L} \int_0^L \ln\left(\frac{C_v}{C_{v0}}\right) dx + \ln\left(\frac{C_v}{C_{v0}}\right) \right]$$



- With the exact σ - C_v equation, the stress level is significantly lower.



Puzzle Three

- Effect of Self-Diffusion

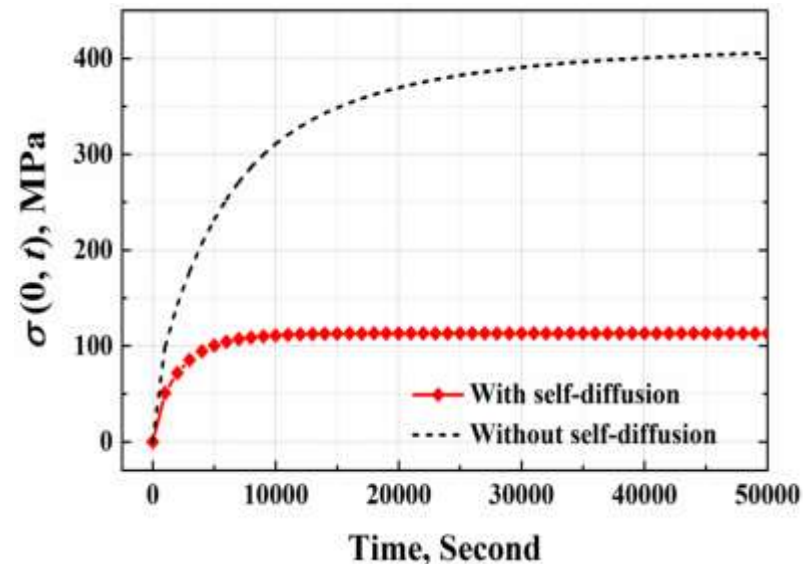
Literature

$$j_v = -\frac{D_v C_v}{k_B T} \left(f \Omega \frac{\partial \sigma}{\partial x} + Z^* e \rho j \right)$$

$$J_v = \frac{D_v C_v}{k_B T} \left(Z^* e \rho j + \frac{k_B T}{C_v} \nabla C_v - f \Omega \nabla \sigma + \frac{Q^*}{T} \nabla T \right)$$

Present solution

$$j_v = -\frac{D_v C_v}{k_B T} \left(\frac{k_B T}{C_v} \frac{\partial C_v}{\partial x} + f \Omega \frac{\partial \sigma}{\partial x} + Z^* e \rho j \right)$$



- When concentration gradient is considered, the stress becomes further lower.



Puzzle Four

- Stress-Strain Relation (Constitutive Equation)

Literature

Electromigration-induced volume strain

$$\varepsilon_{kk}^{EM} = -\frac{\sigma}{B}$$

$$\sigma = 2G\varepsilon + \lambda \text{tr}(\varepsilon)\mathbf{I} - B \text{tr}(\varepsilon^T)\mathbf{I}$$

$$\varepsilon = \varepsilon^M + \varepsilon^T + \varepsilon^{EM} = 0$$

Present solution

$$\varepsilon_{kk}^{EM} = g(C_v)$$

$$g(C_v) = -A \ln\left(\frac{C_v}{C_{v0}}\right) \quad A = \frac{k_B T}{B\Omega}$$

$$\varepsilon = \varepsilon^M + \varepsilon^T + \varepsilon^{EM}$$

$$\varepsilon^T = \alpha \Delta T \mathbf{I}, \quad \varepsilon^{EM} = \frac{g(C_v)}{3} \mathbf{I}, \quad g(C_v) = -A \ln\left(\frac{C_v}{C_{v0}}\right)$$

$$\sigma = 2G\varepsilon + \lambda \text{tr}(\varepsilon)\mathbf{I} - B \text{tr}(\varepsilon^T)\mathbf{I} - B \text{tr}(\varepsilon^{EM})\mathbf{I}$$

- A 3-D, general and self-consistent stress-strain constitutive equation is obtained.



Puzzle Five

Atomic Transport Equation

Literature

$$\begin{cases} \frac{\partial C_v}{\partial t} + \nabla \cdot J_v = G & \text{(sink/source term)} \\ G = \frac{\partial C}{\partial t} \end{cases}$$

$$\frac{\partial C}{C} = d\varepsilon_{kk}^{EM}$$

with approximation

$$\frac{\partial C_v}{\partial t} \ll \frac{\partial C}{\partial t}$$

$$\frac{\partial \varepsilon_{kk}^{EM}}{\partial t} = \Omega \nabla \cdot J_v$$

Present solution

$$\begin{cases} \frac{\partial \theta}{\partial t} = \Omega \nabla \cdot J_v \\ \theta = \varepsilon_{kk}^M + \varepsilon_{kk}^T + \varepsilon_{kk}^{EM} \end{cases}$$

without any approximation

$$\frac{\partial \varepsilon_{kk}^M}{\partial t} + \frac{\partial \varepsilon_{kk}^T}{\partial t} + \frac{\partial \varepsilon_{kk}^{EM}}{\partial t} = \Omega \nabla \cdot J_v$$

- Mass conservation equation is used to describe the atomic transport. Thus, the sink/source term is considered naturally.



New Theory – General Coupling Model

Mass conservation equation:

$$\frac{\partial \theta}{\partial t} = \Omega \nabla \cdot \mathbf{J}_v$$

$$\mathbf{J}_v = -D_v \nabla C_v + D_v C_v \frac{Z^* e \rho \mathbf{j}}{k_B T} - D_v C_v \frac{\Omega}{k_B T} \nabla \sigma + D_v C_v \frac{Q^*}{k_B T} \nabla T$$

Constitutive equation:

$$\theta = \text{tr}(\boldsymbol{\varepsilon}), \quad \boldsymbol{\varepsilon} = \boldsymbol{\varepsilon}^M + \boldsymbol{\varepsilon}^T + \boldsymbol{\varepsilon}^{EM}$$

$$\boldsymbol{\varepsilon}^T = \alpha \Delta T \mathbf{I}, \quad \boldsymbol{\varepsilon}^{EM} = \frac{g(C_v)}{3} \mathbf{I}, \quad g(C_v) = -A \ln \left(\frac{C_v}{C_{v0}} \right)$$

$$\boldsymbol{\sigma} = 2G\boldsymbol{\varepsilon} + \lambda \text{tr}(\boldsymbol{\varepsilon}) \mathbf{I} - B \text{tr}(\boldsymbol{\varepsilon}^T) \mathbf{I} - B \text{tr}(\boldsymbol{\varepsilon}^{EM}) \mathbf{I}$$

$$\sigma = \text{tr} \left(\frac{\boldsymbol{\sigma}}{3} \right)$$

Field equations:

$$\nabla \cdot \boldsymbol{\sigma} + \mathbf{F} = 0 \quad \boldsymbol{\varepsilon} = \frac{1}{2} (\nabla \mathbf{u} + \mathbf{u} \nabla)$$

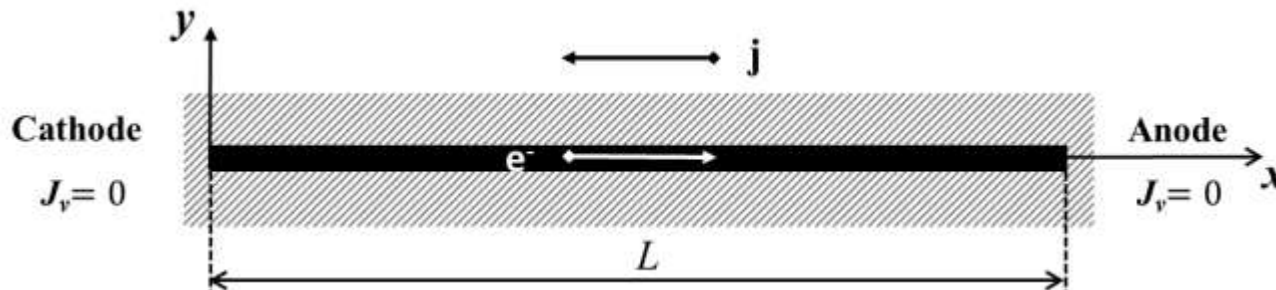
$$\nabla \cdot \mathbf{j} = 0, \quad \mathbf{j} = \frac{\mathbf{E}}{\rho} = -\frac{\nabla V}{\rho}$$

$$k \nabla^2 T + \mathbf{j} \cdot \mathbf{E} = 0$$

Totally fixed configuration – 1-D problem

- **Assumptions**

- 1-D problem: $\varepsilon_y = \varepsilon_z = \varepsilon_{xy} = \varepsilon_{yz} = \varepsilon_{xz} = 0$
- Current density j is constant and in negative x-axis direction.
- Temperature is constant – no temperature gradient.
- Perfectly blocking condition for C_V : $J_v(0,t) = J_v(L,t) = 0$.
- Mechanically totally fixed: $u(L,t) = u(0,t) = 0$



1-D Governing Equations

$$\frac{\partial \theta}{\partial t} + D_v \Omega \left[\frac{\partial \left(\frac{\partial C_v}{\partial x} + \frac{f \Omega C_v}{k_B T} \frac{\partial \sigma}{\partial x} + \frac{Z^* e p j C_v}{k_B T} \right)}{\partial x} \right] = 0$$

$$\theta = -A \ln \left(\frac{C_v}{C_{v0}} \right) + \frac{3(1-2\nu)\sigma}{E}$$

$$\sigma = \frac{2EA}{9(1-\nu)} \ln \left(\frac{C_v}{C_{v0}} \right) + \frac{1+\nu}{3(1-\nu)} \sigma_x$$

$$\sigma_x = \frac{(1-\nu)E}{(1+\nu)(1-2\nu)} \varepsilon_x + \frac{EA}{3(1-2\nu)} \ln \left(\frac{C_v}{C_{v0}} \right)$$

$$\star \frac{d\sigma_x}{dx} = 0$$

$$\star \int_0^L \varepsilon_x dx = 0 \quad u(L, t) = u(0, t) = 0 \rightarrow \int_0^L \varepsilon_x dx = 0 \rightarrow \int_0^L \theta dx = 0$$

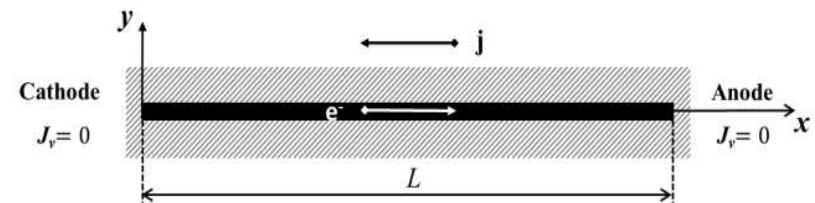
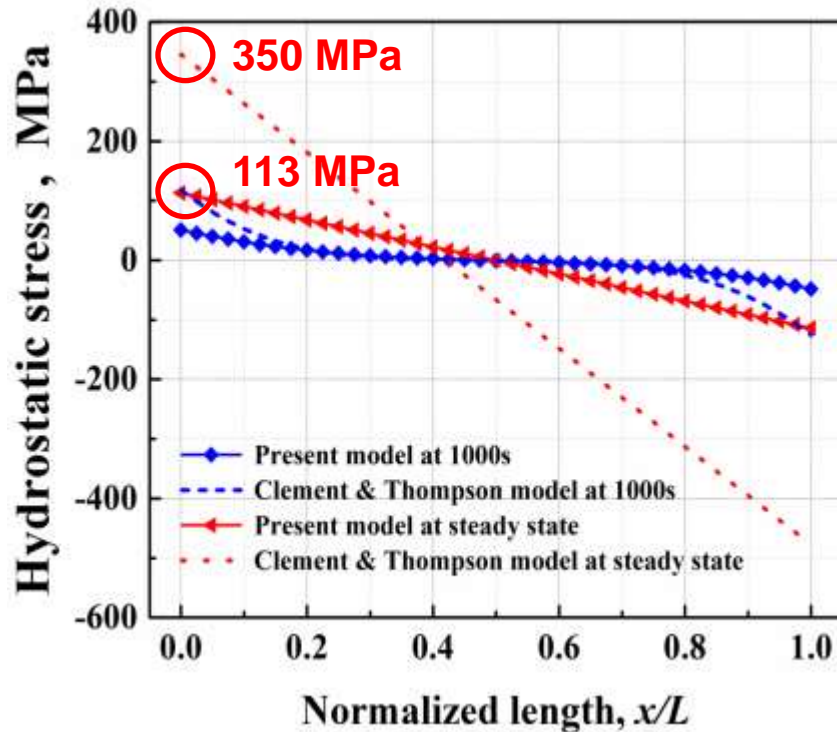
$$\frac{\partial C_v}{\partial t} - \frac{1}{L} \int_0^L \frac{\partial C_v}{\partial t} dx = \frac{3(1-\nu)D_v C_v \Omega}{(1+\nu)A} \left[\left(1 + \frac{2EA\Omega}{9(1-\nu)k_B T} \right) \frac{\partial^2 C_v}{\partial x^2} + \frac{Z^* e p j}{k_B T} \frac{\partial C_v}{\partial x} \right]$$

$$\sigma = \frac{2EA}{9(1-\nu)} \left[\frac{1+\nu}{2(1-2\nu)L} \int_0^L \ln \left(\frac{C_v}{C_{v0}} \right) dx + \ln \left(\frac{C_v}{C_{v0}} \right) \right]$$

Parameters and Physical Properties

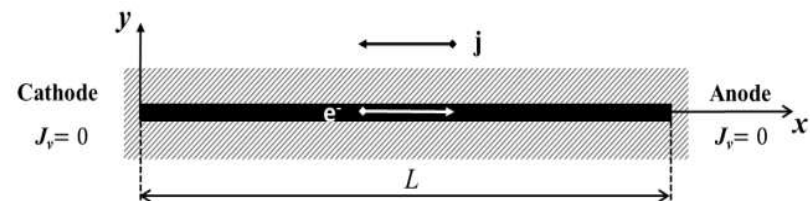
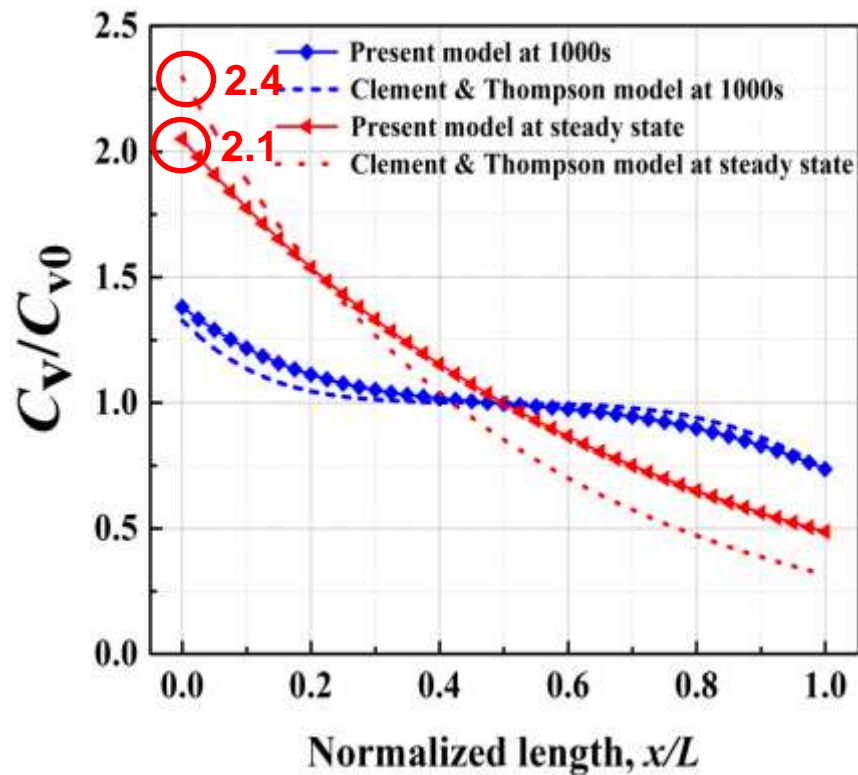
Length of interconnect (L)	50 μm	Temperature (T)	500K
Young's modulus (E)	70×10^9 Pa	Poisson ratio (ν)	0.3
Atomic diffusivity (D_a)	3×10^{-16} m ² /s	Vacancy diffusivity (D_v)	3×10^{-9} m ² /s
Atomic volume (Ω)	1.66×10^{-29} m ³	Initial vacancy concentration (C_{v0})	6.02×10^{21} m ⁻³
$D_v C_v = D_a C_a, \quad C_a = 1/\Omega$		Electrical resistivity (ρ)	4.88×10^{-8} Ohm·m
Current density (j)	10^{10} A/m ²	Elementary charge (e)	1.60×10^{-19} C
Effective charge number (Z^*)	3.5	Boltzmann constant (k_B)	1.38×10^{-23} J/K
Coefficient of electromigration strain (A)	0.0071		

Numerical Results (σ)



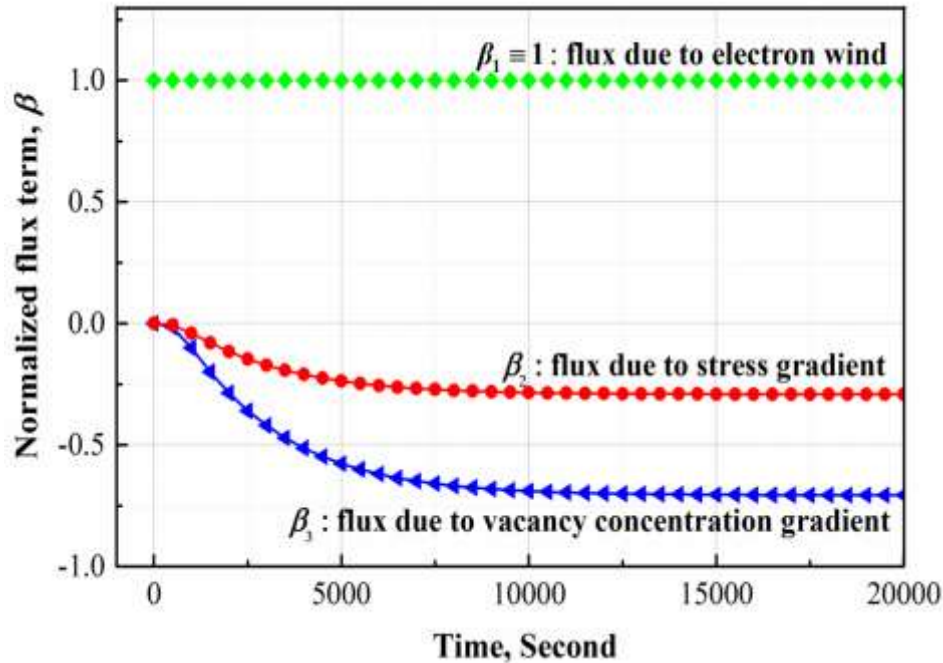
- The present results show significant differences with the previous results, particularly in the magnitude of hydrostatic stress. **(113 MPa vs. 350 MPa: 30% of the stress value obtained).**

Numerical Results (C_v)



- Vacancy concentration is comparable with the previous results, even though stress is significantly lower.

Effect of Self-Diffusion



- Normalized flux term,

$$\beta_1 = \frac{F_E(x, t)}{Z^* e \rho j}, \quad \beta_2 = \frac{F_S(x, t)}{Z^* e \rho j}, \quad \beta_3 = \frac{F_C(x, t)}{Z^* e \rho j}$$

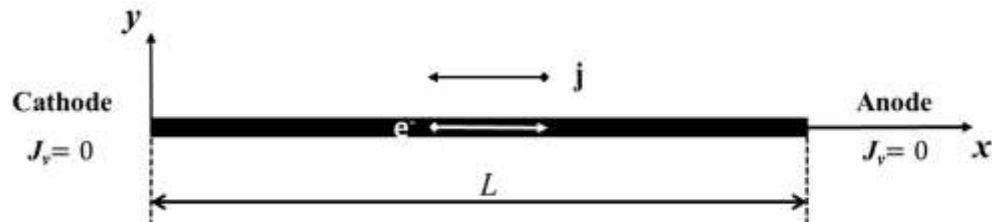
where

$$F_E = Z^* e \rho j, \quad F_S = -\Omega \frac{\partial \sigma}{\partial x}, \quad F_C =$$

$$-\frac{k_B T}{c_v} \frac{\partial c_v}{\partial x}$$

- Self-diffusion plays more important role than stress in electromigration.

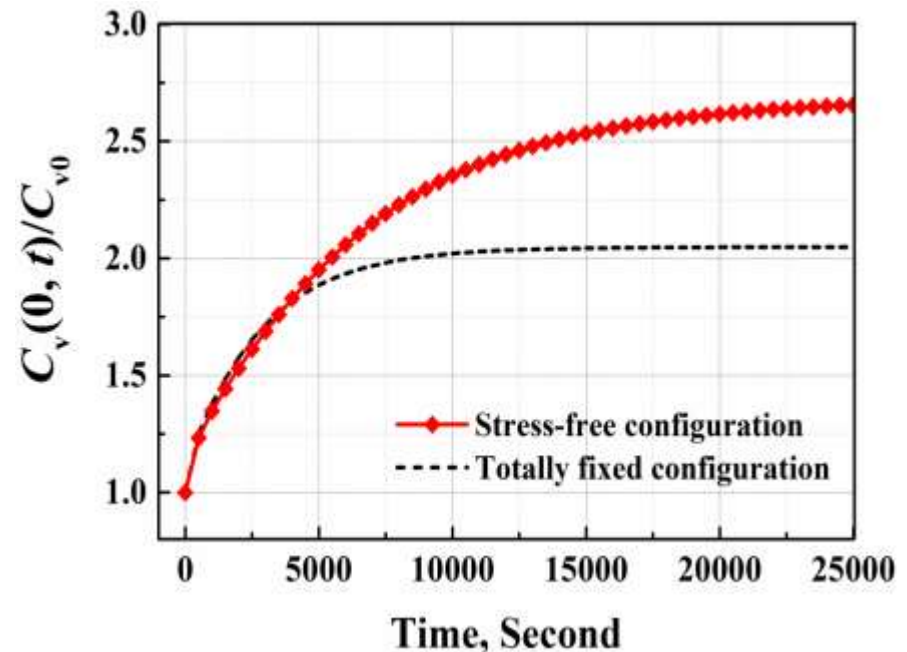
Numerical Results: Stress-Free Condition



- Governing equations,

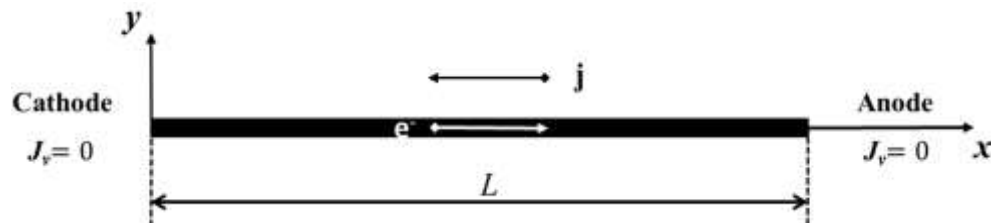
$$\frac{\partial C_v}{\partial t} = \frac{D_v C_v \Omega}{A} \left[\frac{\partial^2 C_v}{\partial x^2} + \frac{Z^* e \rho j}{k_B T} \frac{\partial C_v}{\partial x} \right]$$

$$\sigma = 0$$



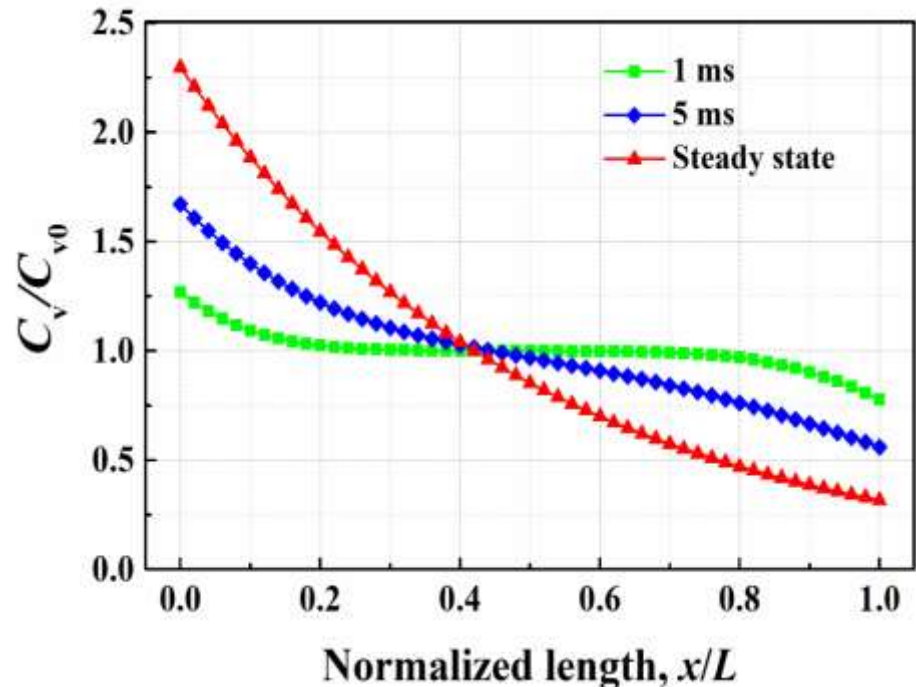
- Electromigration happens on the same time-scale.**
- Electromigration is resisted by the counter force due to concentration gradient.**

Stress-Free Condition (Lloyd's Results)



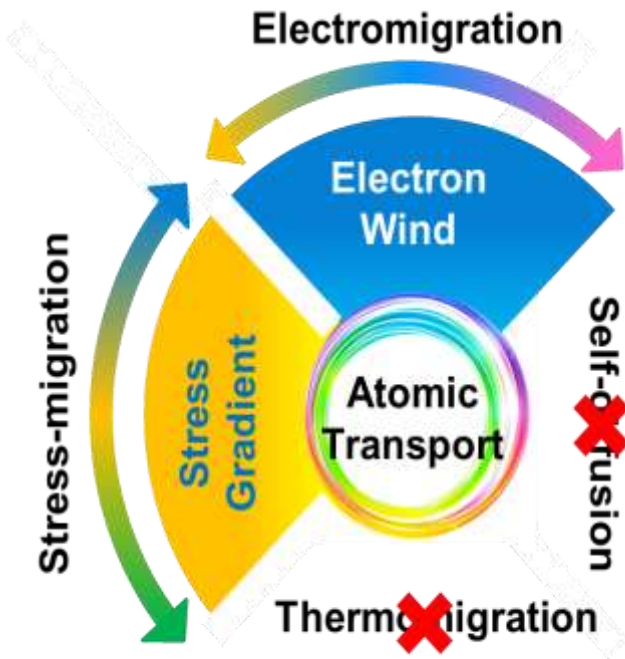
- Governing equations

$$\frac{\partial C_v}{\partial t} = D_v \left[\frac{\partial^2 C_v}{\partial x^2} + \frac{Z^* e \rho j}{k_B T} \frac{\partial C_v}{\partial x} \right]$$

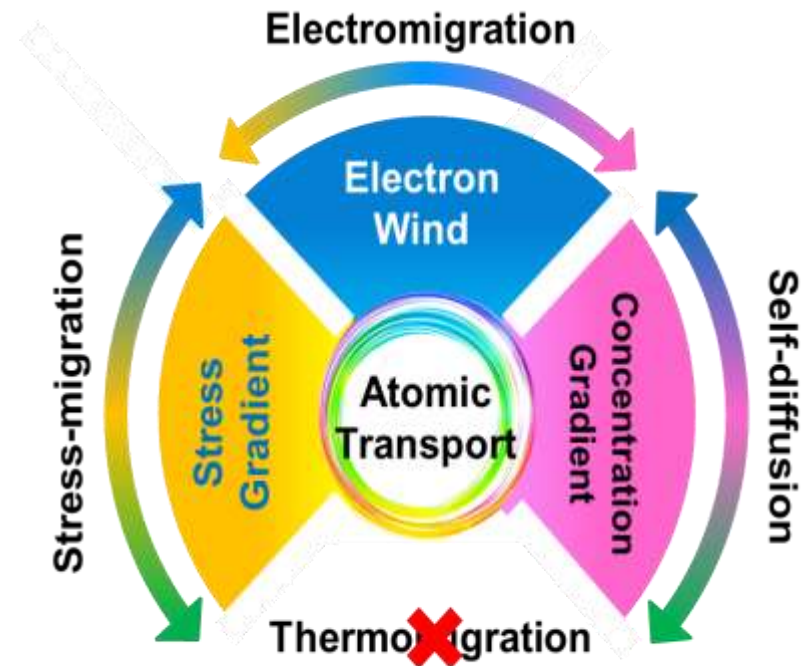


- Electromigration happens in milliseconds.**
- 10^7 orders difference compared to the confined configuration.**

Revisit – Blech's Theory



Blech's theory (1976)

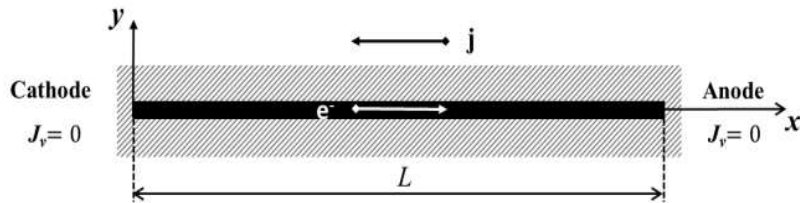


Cui, Fan, Zhang (2019)

- The effect of thermomigration is not taken into the consideration for the purpose of comparison.

Blech Theory's Extension

• Confined Configuration

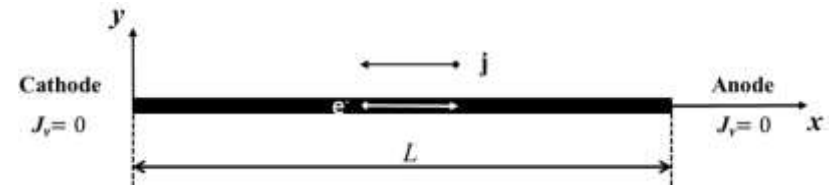


$$\left(1 + \frac{2EAf\Omega}{9(1-\nu)k_B T}\right) \frac{\partial^2 C_v}{\partial x^2} + \frac{Z^* e \rho j}{k_B T} \frac{\partial C_v}{\partial x} = 0$$



$$jL_B = \frac{2EAf\Omega + 9(1-\nu)k_B T}{9(1-\nu)Z^* e \rho} \ln\left(\frac{C_{v,max}}{C_{v,min}}\right)$$

• Stress-free Configuration



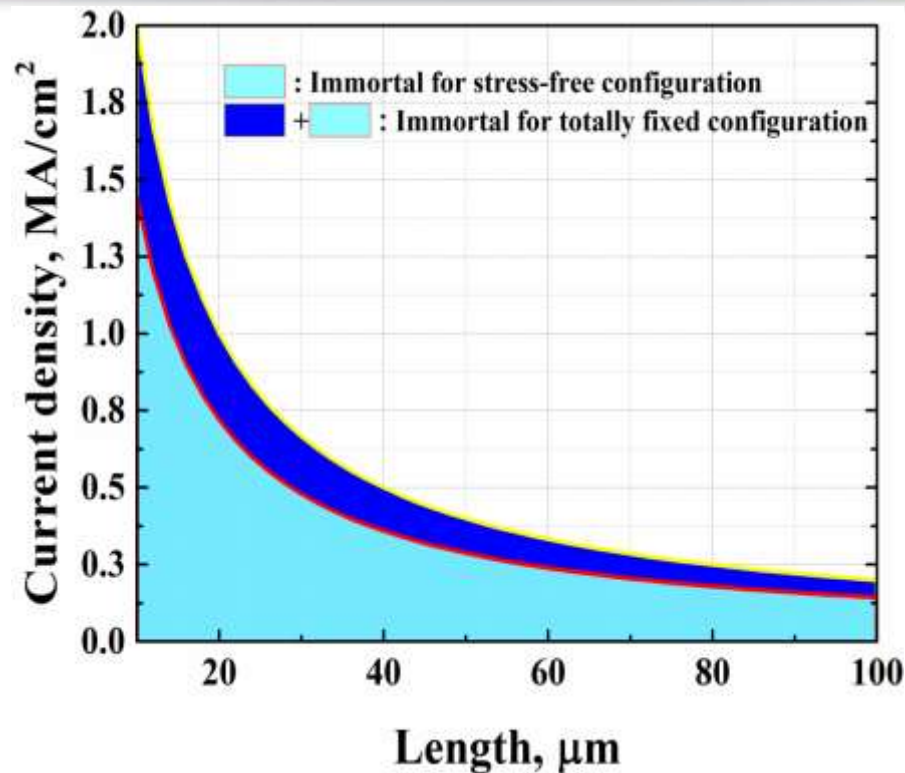
$$\frac{\partial^2 C_v}{\partial x^2} + \frac{Z^* e \rho j}{k_B T} \frac{\partial C_v}{\partial x} = 0$$



$$jL_B = \frac{k_B T}{Z^* e \rho} \ln\left(\frac{C_{v,max}}{C_{v,min}}\right)$$

- The product of jL , is obtained for both confined and stress-free conditions.
- The vacancy concentration is used as threshold condition for failure criterion.

Threshold Product Curve



- The confined metal line can sustain the higher current density than that of stress-free condition.
- The experimental results are consistent with the present theoretical predictions.
 - In Blech's experiment, the measured threshold product for the uncovered metal line is lower than that for the confined metal line.

Summary

- **Electrical-thermal-mechanical modeling – sequential approach**
- **Electromigration**
 - A 3-D general coupling model for electromigration is developed.
 - 1D solution for the confined configuration is obtained.
 - Mechanical stress-based failure criterion may not be valid anymore.
 - Blech's theory is revisited and reanalyzed.
 - Vacancy concentration, instead of hydrostatic stress, used for failure criterion – a departure from the original view.
 - The new predictions are consistent with experimental observations.

- Introduction
- Temperature Loading
- Mechanical Loading
- Moisture Loading
- Electrical Current Loading - Multi-Physics Modeling
- **Summary**

Module 1 – Temperature Loading

- **Thermal mismatch vs. temperature gradient**
- **Analytical solution**
 - Layered structure (stress, warpage, effective CTE)
 - Cylindrical structure (TSV)
- **Die-level thermal stress – thermal stress in TSV**
- **Package-level thermal stress problem – warpage**
- **Chip-package interaction (CPI) – submodeling technique**
- **Board level thermal stress problem**
 - Solder ball thermal cycling (Flip chip BGA, WLP)
 - Creep equations
 - Best method for practice
 - Initial stress free condition; full model vs. global/local model; worst solder ball location, volume averaging
- **Stress singularity of joint materials**

Module 2 – Mechanical Loading

- **JEDEC drop test standard**
 - JESD22-B111, old one, with 15 components
 - JESD22-B111A, new one, with 4 components or 1 component
- **Finite element modeling**
 - Input G method, large mass method, input displacement method, direct acceleration method
 - Global/local modeling
 - Peel stress used as indicator for failure
- **Four-point bending test and modeling**
 - Global/local modeling
 - Global with linear elastic but nonlinear geometry analysis
 - Local model with elastic-plastic modeling

Module 3 – Moisture Diffusion and Vapor Pressure

- **For a general moisture diffusion problem with temperature-dependent C_{sat} and varying ambient RH and temperature with time**
 - \bar{C}_k must be used and the coupled element with thermal-diffusion or thermal-structural-diffusion option must be applied at the same time.
- **ANSYS built-in \bar{C} approach cannot solve the problem with varying RH correctly.**
 - ANSYS diffusion element only
 - C_{sat} must be temperature-independent. Temperature gradient is not considered.
 - ANSYS coupled element with structural-diffusion option.
 - C_{sat} must be temperature-independent. Temperature gradient is not considered.
 - ANSYS coupled element with thermal-diffusion option (or thermal-structural-diffusion) option
 - If \bar{C} is used, RH must be constant.
 - If \bar{C}_k is used, no restriction for any diffusion problems.
- **Vapor pressure model.**
- **Thermal-hygro-mechanical modeling.**

Module 4 – Electrical Current Loading

- **Electrical-thermal-mechanical modeling – sequential approach**
- **Electromigration**
 - A 3-D general coupling model for electromigration is developed.
 - 1D solution for the confined configuration is obtained.
 - Mechanical stress-based failure criterion may not be valid anymore.
 - Blech's theory is revisited and reanalyzed.
 - Vacancy concentration, instead of hydrostatic stress, used for failure criterion – a departure from the original view.
 - The new predictions are consistent with experimental observations.

Thank you for your attention.

Questions?

